

THE INFLUENCE OF PLY ORIENTATION ON THE OPEN-HOLE  
TENSION STRENGTH OF COMPOSITE LAMINATES

By

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To the Faculty of Washington State University:

The members of the Committee appointed to examine the thesis of  
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Abstract

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Composite laminates can be tailored for a specific loading scenario by varying the ply orientation. In open-hole tension, a circular area is removed from the center of the composite specimen to simulate the stress concentration that occurs during the introduction of a bolt or rivet. This stress concentration leads to a decrease in the strength of the specimen, and is likely to be the location of crack initiation in an engineering structure. Increases in first ply failure and ultimate tensile strength have been found by tailoring the ply orientation for open-hole tension.

The strain fields of the tailored laminates have been characterized using the digital image correlation method (DICM), Moiré interferometry (MI), and finite element analysis (FEA). Results indicate that DICM is a useful tool for measuring the first ply failure of the laminate. DICM accurately characterized the strain around the hole throughout damage progression, and was verified using MI and FEA.

The failure criteria examined in the current study include Maximum Strain, Maximum Stress, Tsai-Wu, Tsai-Hill, Hoffman, Hashin, and Yamada-Sun. The first ply failure predictions of these criteria were compared to those experimentally measured by DICM. The Maximum Strain and Maximum Stress failure criteria most accurately predicted first ply failure.

Eight non-traditional laminates were optimized to reduce the stress concentration and increase the first ply failure. The influence of the ply orientation on the open-hole tension strength of these laminates was examined. Significant increases in strength were found by varying the ply orientation of the secondary load bearing plies ( $\pm 45^\circ$  and  $90^\circ$ ), with relatively small material property trade-off for some laminates. Thus, improvements in strength were found for the tailored laminates in the open-hole tension loading scenario.

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# 1: Literature Review

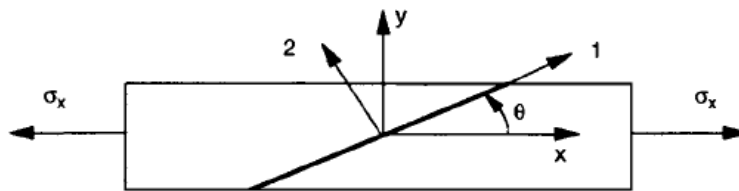
## 1.1: Introduction

The operating strain level in engineering structures is often restricted by the presence of bolts or rivets. The hole drilled for such a fastener introduces a stress concentration that increases stress by a factor of three in an ideal material. While this effect may be reduced in metal structures by cold working holes, the same is not possible for carbon fiber reinforced plastics (CFRP) due to negligible plastic deformation. However, the strength degradation caused by the hole may be reduced by tailoring the ply orientation of the laminate.

Engineers, especially in aerospace, have expressed a strong preference for composite laminates with fiber angles arranged in three directions, such as  $[0_m/\pm 45_n/90_o]_s$ , where  $m$ ,  $n$ , and  $o$  are the number of plies. These layups provide robustness against primary and secondary loading conditions [1]. With this design element in mind, the current research began with a standard quasi-isotropic baseline layup of  $[(45/90/-45/0)_2]_s$ . The subscript “2” indicates that the plies inside the parenthesis are repeated, and the subscript “s” indicates that the bracketed description is repeated symmetrically. The ply angles ( $\pm 45^\circ$ ,  $90^\circ$ , and  $0^\circ$ ) are taken relative to the longitudinal (loading) axis, as shown in Fig. 1.1.1. This layup has enough ply orientations to inhibit matrix cracking, and also has dispersed orientations to inhibit matrix macrocracking [2]. Dispersed layups have angles spread out through-the-thickness, and thus do not have large groupings of the same ply angle.

The following literature review examines the relation between strength and the ply orientation, stacking sequence, and layup of filled- and open-hole composite laminates, as well as the ability of common failure criteria to predict this strength. A filled-hole

includes a bolt or pin. The ply orientation,  $\theta$ , is defined as the angle between the fiber and the longitudinal direction (Fig. 1.1.1), and layups with ply orientations varied from the baseline are termed non-traditional laminates. In Fig. 1.1.1, “1” and “2” are directions longitudinal and transverse to the fiber, “x” and “y” are directions longitudinal and transverse to the applied stress,  $\sigma_x$ , and the corresponding through-the-thickness directions are 3 and z. The stacking sequence is the specific ordering of plies, such as [45/90/-45/0] or [45/0/-45/90]. The layup refers to the specific ply configuration, such as [0/±60] or [±45].



**Figure 1.1.1:** Definition of fiber angle in a composite laminate under tensile stress [3]

The literature review will provide the background for the current research, examining well researched areas as well as current gaps in understanding. This review will serve as a reference throughout the thesis and as a comparison for experimental and theoretical results to follow. It is found that much literature has examined bolt-hole interaction, but much less has concerned itself with the interaction between open-hole tension and a specific variable of interest, such as laminate stacking sequence, layup, or ply orientation. An understanding of these variables helps reveal the physical nature of the notched composite, which is necessary for selecting appropriate failure criteria and performing strength optimization.



## 1.2: Laminate Stacking Sequence

The laminate stacking sequence (LSS) has an important effect on strength. When  $0^\circ$  fibers are placed on the surface of a coupon they buckle more easily due to reduced support, and the laminate loses compressive strength [4]. Laminates with a dispersed LSS have been found to yield higher in-situ ply shear strength [2]. Tay et. al. [5] examined  $[45/0/-45/90]_s$  and  $[\pm 45/90/0]_s$  stacking sequences in open-hole tension laminates. Results indicated that the pattern of damage progression in each ply depends on the stacking sequence. Tay et. al. also found that delamination and ultimate failure load significantly depend on the stacking sequence, with  $[\pm 45/90/0]_s$  having higher ultimate strength.

Research has been performed considering the bolt and washer interaction with the hole. In two articles, Park [6,7] studied the effect of stacking sequence and clamping force on the delamination bearing strength and ultimate bearing strength of pinned and bolted joints in tensile loading. Acoustic emission (AE) was used to determine the onset of delamination failure. Park found that while the ultimate bearing strengths of  $[90_6/0_6]_s$  and  $[0_6/90_6]_s$  were approximately the same, the  $[90_6/0_6]_s$  delamination bearing strength was about twice as high, indicating that designing a layup with  $90^\circ$  plies on the surface will increase the delamination bearing strength. Park found that the  $[90_3/0_3/\pm 45]_s$  has the second highest ultimate bearing strength but the highest delamination bearing strength of the possible stacking sequence variations, and concludes that this layup should be preferred from the view point of fail-safe delamination failure. Park also concluded that the clamping pressure of the bolted joint suppressed delamination onset, and continuously suppressed interlaminar crack propagation, increasing both delamination and ultimate

bearing strengths (though the benefit to ultimate bearing strength reached saturation.) Oh et. al. [8] examined 14 different stacking sequence variations of the layup  $[0C/\pm 45G/\pm 45C/90C]_s$  for hybrid composites, where C is a given percentage of carbon-epoxy plies and G is a given percentage of glass-epoxy plies. Oh et. al. found that as the  $\pm 45^\circ$  plies were evenly distributed through the thickness, the bearing strength increased, irrespective of the C to G ratio. Oh et. al. found equivalent results to Park, in that the bearing strength increased as higher clamping pressure was applied, and that this strength benefit eventually became saturated at a constant value. Oh et. al. also found that as the washer diameter increased the ultimate bearing strength increased and then became saturated, eventually changing the failure mode from bearing to tension.

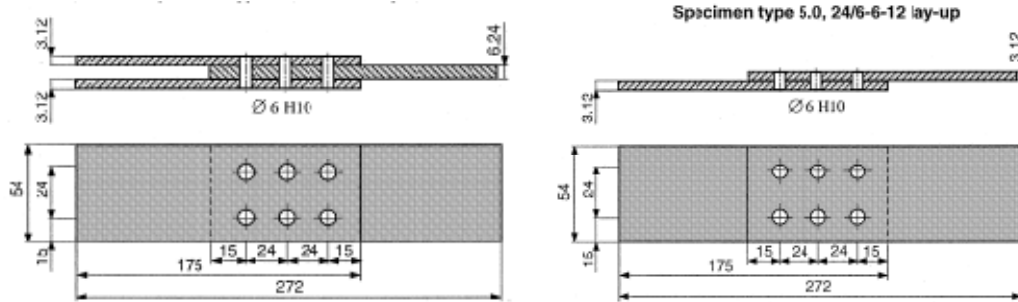
Some LSS optimization methods have been designed to increase composite strength. Todoroki et. al. [9] developed a computer program to optimize the stacking sequence of a composite laminate by reducing the number of plies, while remaining under an allowable strain (calculated from classical lamination theory). Their algorithms allowed constraints to be placed on the final layup, based on experimental knowledge, so that delamination and matrix cracking could be avoided. Their sequential decision algorithm optimized one outer ply at a time from available ply angles. Their branch-and-bound algorithm used a combinatorial optimization approach [9]. The sequential decision algorithm was effective when loading was dominated by bending, but the branch-and-bound algorithm was chosen for having appropriate results within a short computation time. Layups for each algorithm were found for anisotropic, isotropic, and isotropic in-plane conditions with a low applied load. The in-plane layups for the sequential decision approach and the

branch-and-bound approach were  $[90/-45/0/45_2/0/-45]_s$  and  $[30/45/0/-30/-45/90]_s$ , respectively. However, the suggested layups were not compared to experiment.

As shown in this section, LSS has been examined in literature for strength optimization. However, much of the focus has been on bolt-hole interaction [6, 7, 8], and not on characterizing the material behavior, especially in open-hole tension. Some optimization programs have been designed [9], but not tested experimentally. When experimental results have been produced [5], numerical values have either not been offered due to proprietary constraints or have not been compared to other optimization methods. The current study repeats and quantifies some of the experimental results of [5], and compares these results to another LSS and to other optimization methods.

### **1.3: Layup**

The layup of a composite laminate determines to a large extent its response to stress, resulting in specific strength properties for a given loading scenario. The bolt-hole interaction under tensile loading was studied by Starikov et. al. [10]. Starikov examined the response of single and double lap bolted joints with two, four, or six bolts under tensile and compressive loading. Starikov et. al. generally found a greater compressive strength than tensile strength for the same specimen type. Higher strengths were achieved with the zero dominated layup  $[\pm 45/0/90/0_4/90/0_3]_s$  compared to  $[(\pm 45/0/90)_3]_s$ . Starikov et. al. also found that the first bolt row (Fig. 1.3.1) generally transferred the largest amount of load, and that specimens joined by six bolts had the highest tensile and compressive strengths. It was also found that drilling holes can create large areas of delaminated material. It is important to note that these observations are related to Starikov's joint configuration, and cannot be assumed to pertain to all joints.



**Figure 1.3.1:** Joint configuration (dimensions in mm) [10]

The bolt-hole interaction for various layups were also examined by Yan et. al. [11] in an extensive study for Boeing and the FAA. The research involved the failure of composites containing a circular cutout with and without a mechanically tightened bolt, for both double and single lap joints. Yan et. al. found equivalent results to Oh [8] (Section 1.2: Stacking Sequence), in that clamping pressure benefited bolted joints by shifting the mode of failure from bearing to net-tension<sup>1</sup>. Another beneficial result of clamping pressure was the suppression of fiber-matrix splitting. Yan et. al [11] also examined the  $[\pm 45/0/0/-45/0/90/0/90]_s$  layup in bolted joint tests, and found this layup to be prone to delamination and fiber matrix splitting due to the large number of  $0^\circ$  plies. A 15% strength reduction was found when introducing a high number of  $0^\circ$  plies. Yan et. al. found that in tension tested filled-hole laminates, fiber-matrix splitting was the major factor causing strength reduction while delamination was secondary. In contrast, Oh et. al. [8] found that delamination was more critical to bolted-joint failure than in-plane damage in a study that examined stacking sequence, ply angle, clamping pressure, and washer diameter on ultimate bearing strength<sup>2</sup> of hybrid (carbon-epoxy and glass-epoxy)

<sup>1</sup> Sun et. al. [16,17] was able to model this type of shift in failure mode (a shift unique to composite joints) in an in-house FEA program called 3DBOLT.

<sup>2</sup> Where ultimate bearing strength is defined as  $\sigma = P/dt$ , where P is the ultimate failure load, and d is the hole diameter, and t is the thickness of the laminate

laminates. This indicates that conclusions drawn for open-hole and filled-hole laminates are not necessarily applicable to single and double lap bolted joints.

The affects of various layups on through-the-thickness properties were examined by Kostreva et. al. [12]. An experimental procedure was developed to determine through-the-thickness compressive (TTTC) material properties for four different graphite/epoxy material systems. TTTC strength increased as layups were changed from [0], [0/90], [0/±45/90] and [0/±30/±60/90] (TTTC strengths were unaffected by thickness). Kostreva found that while the matrix was dominant in determining TTTC modulus, it was also affected by fiber orientations which doubled the stiffness as layups were changed from cross ply to quasi-isotropic.

Finite element analysis (FEA) has been used to understand the strain response of various layups. Dano et. al. [13] developed a finite element model to predict the bearing response of pin-loaded graphite epoxy composite plates with different isotropic layups. Non-linear shear behavior was found to be very important for strength prediction of two layups, [(0/90)<sub>6</sub>]<sub>s</sub> and [(±45)<sub>6</sub>], but only had a slight effect for [(0/±45/90)<sub>3</sub>]. McCarthy et al. [14,15] verified a non-linear finite element code MSC.Marc with a 3-D ABAQUS code and an in-house finite element code entitled STRIPE. The FEA codes modeled a single bolt, single lap joint with a layup of [45/0/-45/90]<sub>5s</sub>, and a similar layup with a higher percentage of 0° plies. All the models predicted the same secondary bending, bolt tilting, twisting, and through-thickness variations in stress and strain. Surface strain distribution was not affected by bolt-hole clearance, except at points very close to the loaded side of the hole. However bolt-hole clearance had a strong effect on the load at which first significant failure (bearing failure) occurred. Another important finding was

that once significant damage has occurred, strain is no longer constant through-the-thickness. Modeling the gripped area improved the FEA results. A study supporting McCarthy's results was performed by Sun et. al. [16, 17] for the effect of washer clamping area on the ultimate bearing load of a bolt-filled hole. Sun et al. tested a cross ply layup  $[(0/90)_6]$  and a quasi-isotropic layup  $[(0/\pm 45/90)_3]$ . Sun et al. found experimentally that bolt bearing failure is a 3-D phenomenon, and that joint strength depends on the initial clamping force in the washer but more importantly on the size of the clamped area. Sun et. al also found that an increase in the clamping area increased the ultimate bearing load.

Similar to research on LSS, various layups have been examined in the literature for bolt-hole interactions, but much less research has quantified the layup's affect on open-hole tension strength. What has been presented experimentally often has not been compared theoretically, or visa-versa. Proprietary constraints make it difficult to compare the experimental results of one study with the theoretical results of another. Thus, the current research attempts to compare both experimental and theoretical observations for the selected non-traditional laminates, and to quantify these results.

#### **1.4: Ply Orientation**

The ply orientation may be tailored to increase the strength of composite laminates, and some literature has examined the open-hole tension loading scenario. Tan [18] tested the influence of laminate thickness and ply orientation on the strength of graphite epoxy laminates with circular holes centered in an infinite plate. Tan developed a computer model using FORTRAN and the Quadratic failure criterion to estimate strength and optimize the laminate, which successfully represented the behavior of a composite until

initial (first ply) failure. Tan found that  $[0/\pm 29]_s$  had a 41% strength increase over aluminum. Tsau et. al. [19] used the simplex method to optimize the ply orientations of four, six, and eight layer laminates, and used the Hashin failure criterion as an objective function. The simplex method is a direct searching method, appropriate for an unconstrained optimization problem. The stress components were input into Hashin, and the objective function values were obtained. A smaller function value meant higher laminate strength, so the simplex method was used to find a minimum function value. Tsau found that with properly selected initial angles, the simplex method could be used to optimize the ply orientations of the four, six, and eight ply laminates. The optimization was verified by comparing the theoretical and experimental ultimate strengths. However, above eight plies the error between the predicted optimization and experimental results became too high for practical purposes. Also, even for some of the four, six, and eight ply optimizations the error was above ten percent, indicating further study is needed to control the effectiveness of this method. Barakat et. al. [20] used an energy based failure criterion developed by Abu-Farsakh and Abdel-Jawad for the strength optimization of the laminate. Analytical results showed that the laminate configuration was sensitive to the loading case, that the minimum weight occurred at an orientation where the stresses were maximum, and that the optimal laminate configuration was always non-symmetric.

The ply orientation of CFRP composites can alter interlaminar and shear strength properties, as well as material properties such as modulus. Shi, C.H. [21] examined the interlaminar stresses near the circular hole of open-hole carbon/epoxy symmetric laminate coupons. Coupons with ply angles of  $0^\circ$ ,  $15^\circ$ ,  $30^\circ$ ,  $45^\circ$ , and  $90^\circ$  were subjected to in-plane shear. While the maximum interlaminar shear values for the  $45^\circ$  ply angle

laminates are generally the lowest, shear results varied largely depending on the location around the hole and the diameter to thickness ratio. Oh et. al. [8] examined a  $[0_2/\theta_3/90_2]_s$  layup for carbon-epoxy and glass-epoxy laminates where  $\theta$  was  $(0/90)$ ,  $\pm 15^\circ$ ,  $\pm 30^\circ$ , and  $\pm 45^\circ$  ply orientations. Oh et. al. found that  $[0_2/(\pm 45)_3/90_2]_s$  was highest in ultimate bearing strength, and that  $[0_2/(\pm 15)_3/90_2]_s$  had the highest modulus. Treasurer [22] varied the longitudinal ( $0^\circ$ ) plies in traditional laminates for several loading scenarios to be non-traditional  $\pm 5^\circ$  or  $\pm 10^\circ$ . In traditional laminates the stress concentration was redistributed after longitudinal matrix splitting, however in the non-traditional laminates this splitting was suppressed causing a reduction in strength and modulus in open-hole tension and compression. However, non-traditional laminates increased the strength of single-shear bearing due to increased bearing resistance. Muc [23] analytically investigated the effect of ply orientations for uniaxial and biaxial compression on the maximum buckling load. Muc used the Love-Kirchhoff hypothesis to obtain the optimized laminate, which depended on the buckling mode, geometry, and material properties.

Much less literature is available for the influence of ply orientation on open-hole tension strength than for the influence of LSS or layup on strength. The literature that has examined ply orientation is typically confined to a few angles [21, 22], and optimization programs are also limited by number of plies [19]. The focus of the current study is to select optimized non-traditional laminates from all possible ply orientation angles ( $0^\circ$  to  $90^\circ$ ), and to produce a large volume of experimental strength and strain field results.



## 1.5: Failure Criteria

Failure criteria have been used in literature to predict the strength of CFRP composites. Dano et. al. [13] observed that when the shear stress is large, the Hashin criterion is more conservative than the maximum stress criterion. However, when the shear stress strain curve is linear, maximum stress criterion predicts higher and more accurate strength than Hashin Criterion. Dano et. al proposed a mixed failure criteria and associated table of degradation rules for pin-loaded composites.

Park [6,7] uses Ye-delamination failure criterion based on layerwise finite element contact stress analysis to predict the bearing failure strengths of mechanically fastened joints. The predicted strengths are lower than experimental strengths, a difference which increases as clamping pressure increases. The predicted and measured values stay within 6-10%.

A recently proposed strain invariant failure theory (SIFT) shows promise for modeling progressive damage and predicting strength in carbon fiber-epoxy specimens, and may avoid many problems associated with other failure criteria. Previous methods have used failure criteria to degrade properties in the stiffness matrix of failed FEA elements. These material property degradation methods (MPDMs) are generally high in computational demand<sup>3</sup>, and their solutions do not always converge. Gosse [24] used SIFT with a damage function methodology to obtain the maximum energy retention (MER, which corresponds to maximum load capacity of the strained specimen). This theory is based on the idea that solids fail by excessive deformation. The MER methodology was successfully demonstrated with normalized data from open-hole

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<sup>3</sup> When using an MPDM, commercial codes such as ABAQUS must recalculate entire meshes iteratively when properties are degraded causing a high computation demand.

compression, open-hole tension, filled-hole tension, large-notch tension, and large-notch compression tests.

Tay et. al. [5] used the SIFT based Element-Failure Method (EFM) to avoid problems associated with the MPDM and to present a simpler finite element approach. The Element Failure Method (EFM) modifies nodal forces in a finite element model to reflect the general state of damage and loading. Tay et. al. found that nodal force modification in the thickness direction is needed to correctly predict interlaminar damage, supporting the notion that composite damage progression is a complex three dimensional process. Tay's finite element model was in-house due to user constraints in the commercial code ABAQUS (Tay et. al., [25]). Implementation of the SIFT-MER-EFM approach has been demonstrated for many test scenarios and allows for both mechanical and environmental loading (thermal residual strains).

### 1.5.1: Mode-based Failure Criteria

The Maximum Stress and Maximum Strain Criteria are dependent on the failure strength of the lamina [26]. The Maximum Stress Criterion is based on the inequalities

$$s_1^{(-)} < \sigma_{11} < s_1^{(+)} \quad (1.5.1)$$

$$s_2^{(-)} < \sigma_{22} < s_2^{(+)} \quad (1.5.2)$$

$$|\tau_{12}| < s_{12}, \quad (1.5.3)$$

where  $s_1$ ,  $s_2$ , and  $s_{12}$  are the ultimate longitudinal, transverse, and shear strengths, respectively, in tension (+) and compression (-), and  $\sigma_{11}$ ,  $\sigma_{22}$ , and  $\tau_{12}$  are the in-plane stresses.

Similarly, the Maximum Strain Criterion is based on the strength of the lamina in terms of engineering strain:

$$e_1^{(-)} < \varepsilon_{11} < e_1^{(+)} \quad (1.5.4)$$

$$e_2^{(-)} < \varepsilon_{22} < e_2^{(+)} \quad (1.5.5)$$

$$|\gamma_{12}| < e_{12}, \quad (1.5.6)$$

where  $e_1$ ,  $e_2$ , and  $e_{12}$  are the ultimate longitudinal, transverse, and shear strains, respectively, in tension (+) and compression (-), and  $\varepsilon_{11}$ ,  $\varepsilon_{22}$ , and  $\gamma_{12}$  are the in-plane strains. In these ply-by-ply failure criteria, analysis is confined to in-plane fracture. Possible interaction between stress components, crack propagation between plies, and delamination is neglected in the two dimensional form of the criteria.

Hashin [27] proposed a piecewise criterion to account for possible failure modes of the composite. For plane stress, the fiber-mode failure in tension ( $\sigma_{11} > 0$ ) is

$$\left( \frac{\sigma_{11}}{s_1^{(+)}} \right)^2 + \left( \frac{\tau_{12}}{s_{12}} \right)^2 = 1. \quad (1.5.7)$$

For fiber-mode failure under compression ( $\sigma_{11} < 0$ ) this becomes

$$\left( \frac{\sigma_{11}}{s_1^{(-)}} \right)^2 = 1. \quad (1.5.8)$$

Similar equations are defined for the matrix-mode failure, and for plane stress under tension ( $\sigma_{22} > 0$ ) this is

$$\left( \frac{\sigma_{22}}{s_1^{(+)}} \right)^2 + \left( \frac{\tau_{12}}{s_{12}} \right)^2 = 1. \quad (1.5.9)$$

For matrix-mode failure under compression ( $\sigma_{22} < 0$ ) this becomes

$$\left( \frac{\sigma_{22}}{2s_{23}} \right)^2 + \left[ \left( \frac{s_2^{(-)}}{2s_{23}} \right)^2 - 1 \right] \left[ \left( \frac{\sigma_{22}}{s_2^{(-)}} \right) + \left( \frac{\sigma_{12}}{s_{12}} \right) \right]^2 = 1. \quad (1.5.10)$$

### 1.5.2: Quadratic Interaction Failure Criteria

Quadratic criteria differ from limit criteria because they use quadratic terms to account for the interaction of stress components. While these terms are meant to mimic the stress interaction in the composite, it is important to remember that these criteria are purely empirical. Hill [28] was one of the first to suggest such a criterion. He modified the von Mises Criterion for isotropic metals to include anisotropy in the equation

$$A(\sigma_{22} - \sigma_{33})^2 + B(\sigma_{33} - \sigma_{11})^2 + C(\sigma_{11} - \sigma_{22})^2 + 2D\tau_{23}^2 + 2E\tau_{31}^2 + 2F\tau_{12}^2 = 1, \quad (1.5.11)$$

where A, B, C, D, E, and F are constants based on yield strength, and defined in [28]. The Hill Criterion was modified by Tsai [30] to assume plane stress in a transversely isotropic lamina as

$$\frac{\sigma_{11}^2}{s_1^2} - \frac{\sigma_{11}\sigma_{22}}{s_1^2} + \frac{\sigma_{22}^2}{s_2^2} + \frac{\tau_{12}^2}{s_{12}^2} = 1, \quad (1.5.12)$$

and is referred to as the Tsai-Hill Criterion [26]. Generalization of the von Mises criterion for orthotropic material lacks analytic basis [30], resulting in 6 independent strength components instead of 9. Hoffman [31] added linear stress terms to the Tsai-Hill Criterion in order to account for unequal stresses in tension and compression [26, 32], and the result was

$$\frac{\sigma_{11}^2}{s_1^{(+)}s_1^{(-)}} - \frac{\sigma_{11}\sigma_{22}}{2\sigma_{11}^{(+)}\sigma_{11}^{(-)}} + \frac{\sigma_{22}^2}{s_2^{(+)}s_2^{(-)}} + \left(\frac{\tau_{12}}{s_{12}}\right)^2 + \left(\frac{1}{s_1^{(+)}} - \frac{1}{s_1^{(-)}}\right)\sigma_{11} + \left(\frac{1}{s_2^{(+)}} - \frac{1}{s_2^{(-)}}\right)\sigma_{22}. \quad (1.5.13)$$

Tsai and Wu [30] proposed a tensor polynomial theory for anisotropic materials of the form of

$$F_i\sigma_i + F_{ij}\sigma_i\sigma_j = 1. \quad (1.5.14)$$

This equation accounts for tensile and compressive stress induced failures, allowing the failure curve to be represented in a single function. Under plane stress the Tsai-Wu Criterion becomes

$$F_{11}\sigma_{11}^2 + F_{22}\sigma_{22}^2 + F_{66}\tau_{12}^2 + F_1\sigma_1 + F_2\sigma_{22} + 2F_{12}\sigma_{11}\sigma_{22} = 1, \quad (1.5.15)$$

where the  $F_{ij}$  and  $F_i$  terms are found from uniaxial and shear strength tests and are

$$F_{11} = \frac{1}{s_1^{(+)}s_1^{(-)}}, \quad F_1 = \frac{1}{s_1^{(+)}} - \frac{1}{s_1^{(-)}}, \quad F_{22} = \frac{1}{s_2^{(+)}s_2^{(-)}},$$

$$F_2 = \frac{1}{s_2^{(+)}} - \frac{1}{s_2^{(-)}}, \quad F_{66} = \frac{1}{s_{12}^2}. \quad (1.5.16)$$

$F_{12}$  was modified by Tsai and Hahn [33] to be

$$F_{12} = -\frac{(F_{11}F_{22})^{1/2}}{2} \quad (1.5.17)$$

to avoid a complicated biaxial test and optimization procedure required by the original criterion. Yamada-Sun simplified the Tsai criterion for unidirectional materials [32] to

$$\left(\frac{\sigma_{11}}{s_1^{(+)}}\right)^2 + \left(\frac{\tau_{12}}{s_{12}}\right)^2 = 1. \quad (1.5.18)$$

Quadratic criteria are fundamentally curve fits, with quadratic terms manipulated to represent stress interactions. It is important to note that these terms aren't based on specific failure modes, but rather experimental data.

### 1.5.3: Failure Criteria versus Experimental Results

Failure criteria have shown useful results for some loading scenarios, and inadequate results for others. Kam and Sher [34] studied the first ply failure and nonlinear behavior of cross-ply laminated composite plates, and examined the energy based von Karman Plate theory, the Maximum Stress, Maximum Strain, Hoffman, Tsai-Hill, and Tsai-Wu

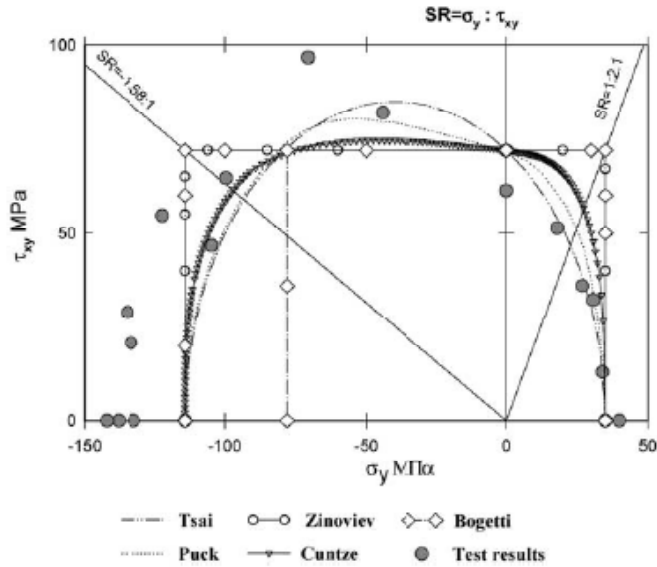
Criteria. Matrix cracking was found to be a major factor leading to stiffness reduction, and inconsistent results indicated that the strength-of-material approach to first-ply failure prediction was insufficient. Treasurer [22] found that the Tsai-Wu Criterion predicted experimental results for non-traditional laminates better than the Hashin or Maximum Stress Criteria.

Soden et. al. [1] organized a World Wide Failure Exercise (WWFE) to test the validity of 19 leading composite strength theories. Of these, the top 5 theories were evaluated and recommendations on their applicability were made. The top 5 included the theories of Zinoviev [35], Bogetti [36], Puck [37], Cuntze [38], and Tsai [39]. Theoretical and experimental agreement for initial failure was considered poor, partly due to lack of reliable experimental data, disagreement between definitions of initial failure, and not enough attention given to residual thermal stresses<sup>4</sup>. Bogetti and Zinoviev ranked best for initial failure in multidirectional laminates. Puck partially allowed for thermal stresses, and his theory yielded conservative predictions. For final failure, none of the theories in the WWFE could predict within  $\pm 10\%$  of strength for more than 40% of the test cases. In general, the theories were less accurate when shear stress and matrix behavior were significant, and where large deformation occurred before failure. However, some theories did well for predicting some test cases, as shown in Figures 1.5.1-1.5.3. The WWFE findings were published in 2004, and this was one of the largest failure criteria studies to date. The WWFE was useful in determining the weaknesses of current failure theories, and the direction that composite research should pursue. One of the weaknesses addressed was the lack of theoretical and experimental definitions for initial (first ply) failure, and corresponding lack of reliable experimental data. It will be found in the

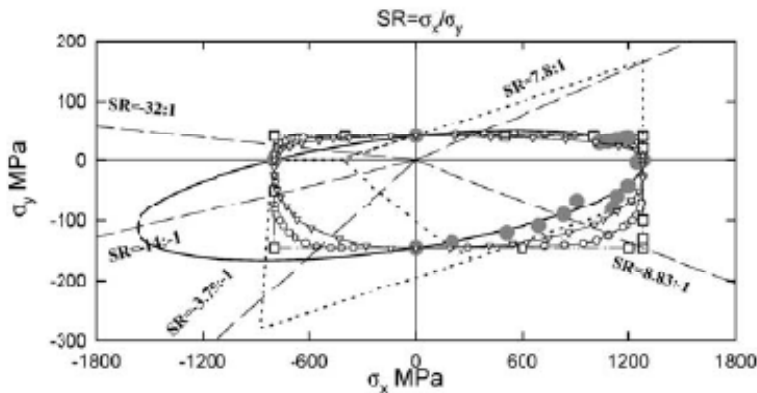
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<sup>4</sup> Residual thermal stresses can be introduced in curing, and when moisture exists.

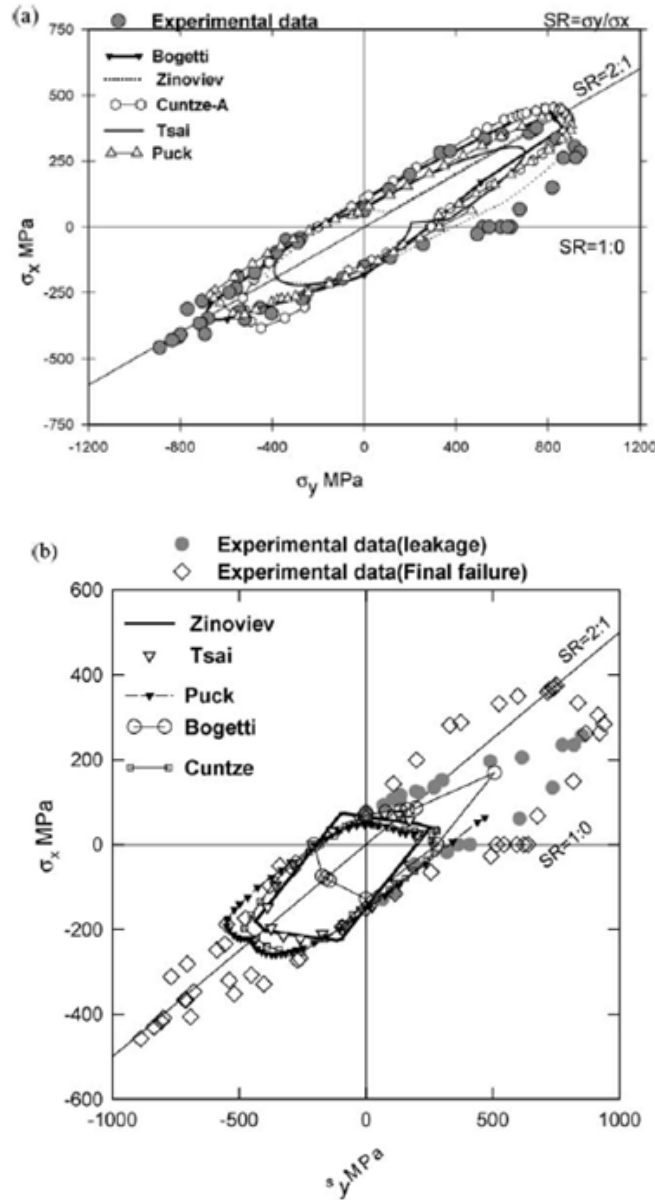
current study that the digital image correlation method (DICM) can be used to quantify first ply failure in open-hole tension laminates, and that the results compared well to some failure criteria.



**Figure 1.5.1:** Comparison between the predicted and measured failure stresses for a unidirectional fiber-reinforced lamina made of GRP material E-Glass/Ly556/HT907/Dy063 and subjected to combined shear and normal stresses perpendicular to the fibers (Test Case no. 1). [1]



**Figure 1.5.2:** Comparison between the predicted and measured biaxial failure envelope for a unidirectional fiber-reinforced GRP lamina under combined normal stresses in directions parallel ( $\sigma_x$ ) and perpendicular ( $\sigma_y$ ) to the fibers. Material: E-glass/MY750 epoxy (Test Case No. 3). [1]



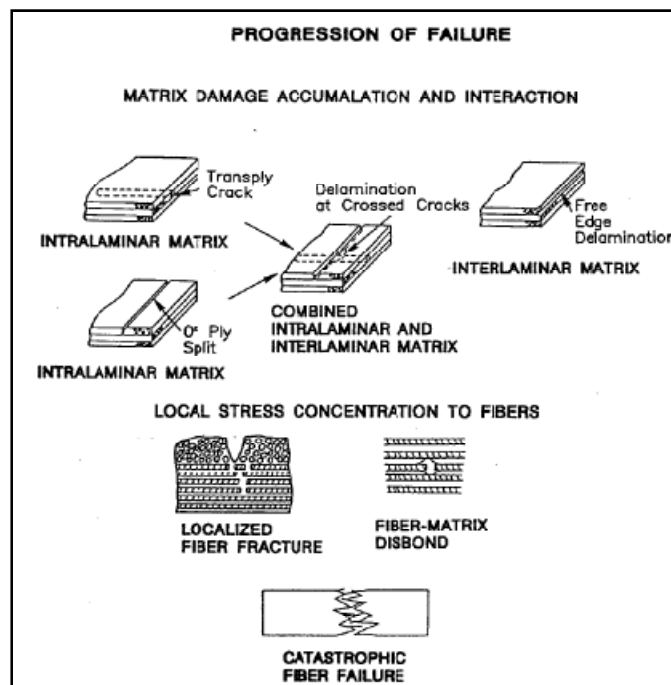
**Figure 1.5.3:** (a) Comparison between the predicted and measured final failure stresses for ( $\pm 55^\circ$ ) E-glass/MY750 laminates subjected to biaxial loads (Test Case No. 9). (b) Comparison between the predicted and measured ‘initial’ failure stresses for ( $\pm 55^\circ$ ) E-glass/MY750 laminates subjected to biaxial loads (Test Case No. 9). [1]

From these results it is clear that failure criteria must be examined in non-traditional laminates to understand their applicability, as well as their limits. As shown above, a failure criterion may predict strength well in one loading scenario, but be overly-conservative or unrealistic in others.



## 1.6: Failure Progression

Failure in multidirectional laminates is a progressive and three-dimensional process. First ply failure often occurs in the form of intralaminar matrix cracks such as matrix splitting. Intralaminar cracks do not always degrade strength, but may promote other forms of damage. Under fatigue loading, these cracks may propagate into adjacent plies. When the cracks span the thickness of several off-axis plies, they can cause a stress concentration in the load bearing  $0^\circ$  plies, resulting in a reduction of tensile strength [2]. Intralaminar cracks may also couple with interlaminar matrix failure (delamination), to completely isolate a ply or group of plies. Delamination is prone to initiate at the free-edges of the coupon. Various forms of tensile damage are shown in Fig. 1.6.1. Even when matrix cracking does not degrade strength, it may alter properties such as thermal expansion, liquid permeability, and oxidative stability.



**Figure 1.6.1:** Failure mechanisms for laminates loaded in tension [2]

Damage becomes more complicated with the introduction of a hole due to a bolt or rivet. Matrix damage and fiber matrix splitting at the hole may actually increase the ultimate tensile strength because the stress is redistributed. This often occurs in “hard laminates,” which have an increased number of  $0^\circ$  plies. During the stress redistribution, material stiffness is reduced [2]. Treasurer [22] also found this softening effect during the stress redistribution of hard laminates. The decrease in material stiffness was verified in the current study, as will be shown in Chapter 5. Treasurer [22] also found that replacing longitudinal plies with off-axis plies ( $\pm 5^\circ$  and  $\pm 10^\circ$ ) in open-hole tension suppressed this stress redistribution and decreased strength. However, Treasurer found a 23% and 25% strength increase for the  $\pm 5^\circ$  and  $\pm 10^\circ$  longitudinal ply replacements in single-shear bearing. The slightly off-axis plies added bearing resistance compared to the traditional layups.

### **1.7: Conclusion**

As shown in the literature review, much research has been done considering fastener design. Studies may also be found focusing on the effect of stress concentrations in composites. However, much of this work has concentrated on the interaction of the bolt and the hole. Research that concerns itself with ply orientation generally only considers changing the number of plies in standard layups, or comparing standard layups with one another. Less work attempted to tailor the ply orientation to increase strength, which is the focus of the current study.

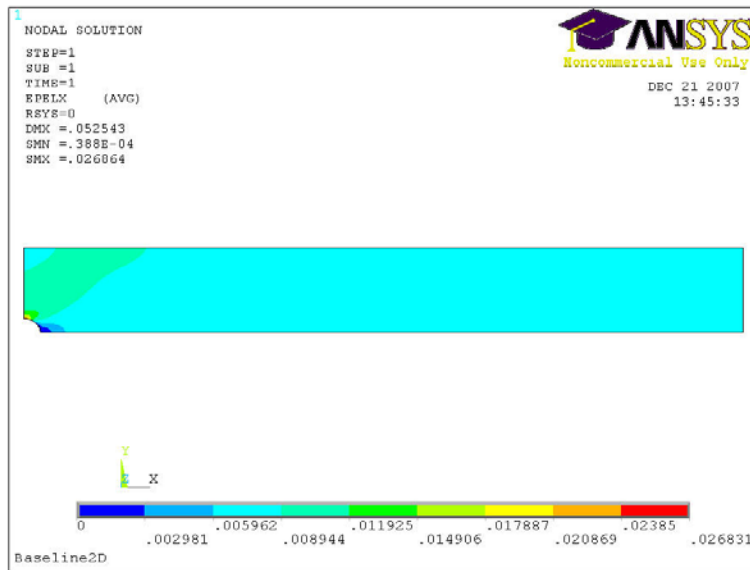
The results in literature for open-hole tension are often limited by proprietary constraint, lack of theoretical and experimental comparison, or limits in reliability (as in the case of initial failure). The current study examines numerous non-traditional

laminates in open-hole tension and the corresponding interaction between the LSS, layup, and ply orientation near the hole. Experimental results are compared to theoretical predictions, as well as to results available in literature. It will be shown that the laminate may be tailored to increase strength using tools readily available in industry, and that the digital image correlation method is a valuable tool for locating first ply failure in experimental research.

## 2: Finite Element Analysis

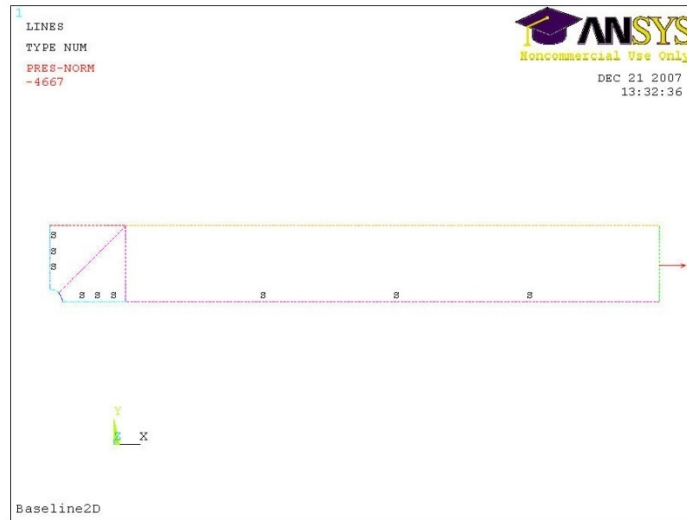
### 2.1: Finite Element Model

A two-dimensional finite element model (FEM) was created in Ansys to describe the strain around the hole of non-traditional laminates (Fig. 2.1.1). Two optimization methods were considered: the minimization of the strain concentration,  $k_t$ , and the maximization of the first ply failure load, FPF. The strain concentration,  $k_t$ , is the ratio of the maximum strain to the nominal strain. Using these methods, the ply orientations were varied and the optimal layups were found.



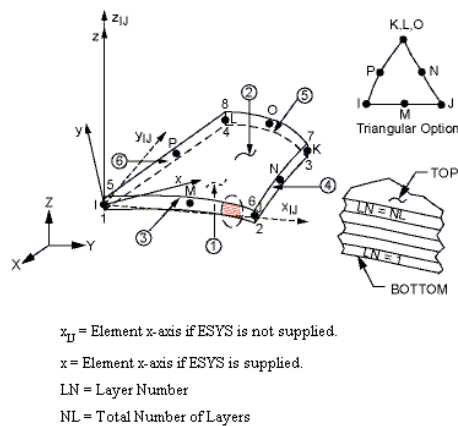
**Figure 2.1.1:** 2D Finite Element Model

A quarter of the open-hole tension coupon was modeled, allowed by symmetry along the longitudinal and transverse centers (Fig. 2.1.2). The quarter model reduced the computation time and the number of elements required. A 4.67 kip/in line pressure was applied to the right edge of the coupon, corresponding to a 66.7 ksi tensile stress for a coupon with a thickness of 0.07 inches and a width of 1.5 inches.



**Figure 2.1.2:** Boundary Conditions

The finite element model was constructed using a linear Shell99 element (Fig. 2.1.3). This element allows the input of either the ply orientations or the ABD matrix (from classical lamination theory) to describe the layup. Shell99 is designed for layered applications of two-dimensional structural shell models, appropriate for thin laminates. The element has eight nodes (four corner, four midside), one element through the thickness, and six degrees of freedom per node. Shell99 has less element formulation time than other shell elements (Shell91) because it doesn't have nonlinear capabilities.



**Figure 2.1.3:** Shell99 Element (© Ansys 2006)

T600: 125-33 carbon/epoxy material properties, found using in-plane lab tests, were input as an orthotropic material model in Ansys, and are given in Table 2.1.1 (as well as Appendix A Table A.1), with the corresponding standard deviations. The out-of-plane material properties ( $\nu_{23}$ ) and compressive strengths were estimated from AS4/3501-6 [40], where the standard deviation was not reported in the referenced literature. The tests were performed under tensile loading using a biaxial extensometer to record strain and using a load cell to record load. Five specimens of  $[0]_6$ ,  $[90]_{16}$ , and  $[\pm 45]_{2s}$  were tested in order to find the corresponding modulus, Poisson ratio, and strength (Table 2.1.1).

**Table 2.1.1:** T600: 125-33 Composite Material System (psi) <sup>5</sup>

$E_1 = 19.73 \times 10^6 \pm 0.65 \times 10^6$	$E_2 = 1.38 \times 10^6 \pm 0.021 \times 10^6$	$E_3 = E_2$
$G_{12} = 0.81 \times 10^6 \pm 0.022 \times 10^6$	$G_{13} = G_{12}$	$\nu_{12} = 0.298 \pm 0.019$
$\nu_{23} = 0.35$	$\nu_{13} = \nu_{12}$	$s_L^{(+)} = 332.8 \times 10^3 \pm 16.2 \times 10^3$
$s_T^{(+)} = 6.28 \times 10^3 \pm 0.63 \times 10^3$	$s_{LT} = 9.03 \times 10^3 \pm 0.37 \times 10^3$	$s_L^{(-)} = 209 \times 10^3$
$s_T^{(-)} = 33 \times 10^3$		

The standard laminate coordinate axis and ply angles were used, defined by Fig.

1.1.1. Modulus was defined as the slope in the elastic region of the longitudinal, transverse, and shear stress-strain curves for the  $[0]_6$ ,  $[90]_{16}$ , and  $[\pm 45]_{2s}$  layups, respectively. As can be concluded from the definition of a ply angle (Fig. 1.1.1), the longitudinal stress and transverse stresses for the  $[0]_6$  and  $[90]_{16}$  layups were equivalent to the loading direction (applied) stresses in each test, and the longitudinal and transverse strains were equivalent to the loading direction strain. For  $[\pm 45]_{2s}$ , the shear stress was found as half the applied stress using the stress transformation [26]

$$\begin{Bmatrix} \sigma_1 \\ \sigma_2 \\ \tau_{12} \end{Bmatrix} = [T] \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} \quad (2.1.1)$$

<sup>5</sup> Properties were found from in-plane tests performed at the WSU Environmental Exposure Facility by Matt Shultz.

where the transformation matrix  $[T]$  is given by

$$[T] = \begin{bmatrix} c^2 & s^2 & 2cs \\ s^2 & c^2 & -2cs \\ -cs & cs & c^2 - s^2 \end{bmatrix} \quad (2.1.2)$$

with  $c=\cos(\theta)$   $s=\sin(\theta)$ . Substituting  $\sigma_x=\tau_{xy}=0$  for the  $[(\pm 45)_2]_s$  loading scenario

$$\tau_{12} = -\cos(45^\circ)\sin(45^\circ)\sigma_x = -\frac{\sigma_x}{2}. \quad (2.1.3)$$

Similarly, the shear strain is found using the strain transformation [26]

$$\begin{Bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \gamma_{12}/2 \end{Bmatrix} = [T] \begin{Bmatrix} \varepsilon_x \\ \varepsilon_y \\ \gamma_{xy}/2 \end{Bmatrix}, \quad (2.1.4)$$

and solving for the shear strain we obtain

$$\frac{\gamma_{12}}{2} = -\cos(45^\circ)\sin(45^\circ)\varepsilon_x + \cos(45^\circ)\sin(45^\circ)\varepsilon_y + (\cos^2(45^\circ) - \sin^2(45^\circ))\frac{\gamma_{xy}}{2}. \quad (2.1.5)$$

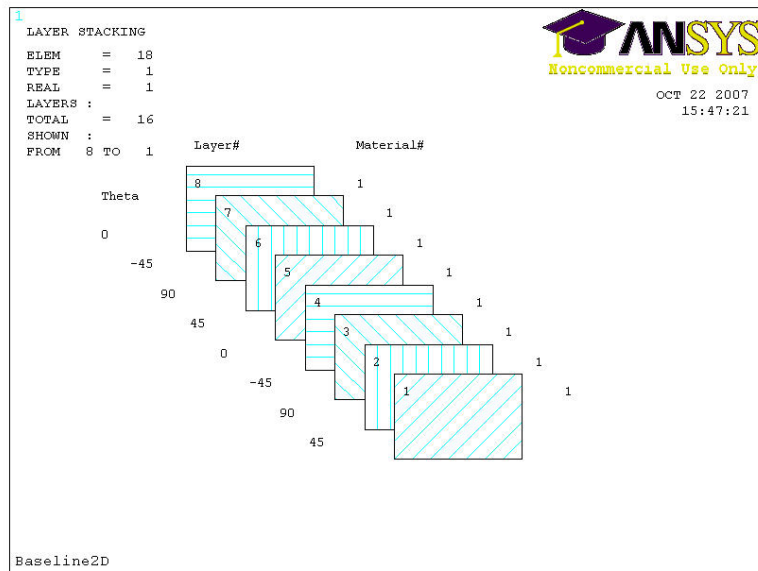
This reduces to

$$\gamma_{12} = -\varepsilon_x + \varepsilon_y, \quad (2.1.6)$$

which shows that the shear strain for the  $[\pm 45]_{2s}$  test can be found from the sum of the longitudinal and transverse strains, measured by the biaxial extensometer. The longitudinal and transverse strengths were defined as the ultimate tensile strength (Appendix A Table A.1). The shear strength was defined as the stress corresponding to 50,000  $\mu\varepsilon$ , according to ASTM D3518 (discussed further in Section 5.3).

The dimensions for the baseline coupon are those of a standard open-hole tension coupon (Appendix B), with a baseline layup of  $[(45/90/-45/0)_2]_s$ . This baseline layup is shown ply-by-ply for half the laminate in Fig. 2.1.4 (the other half of the laminate is

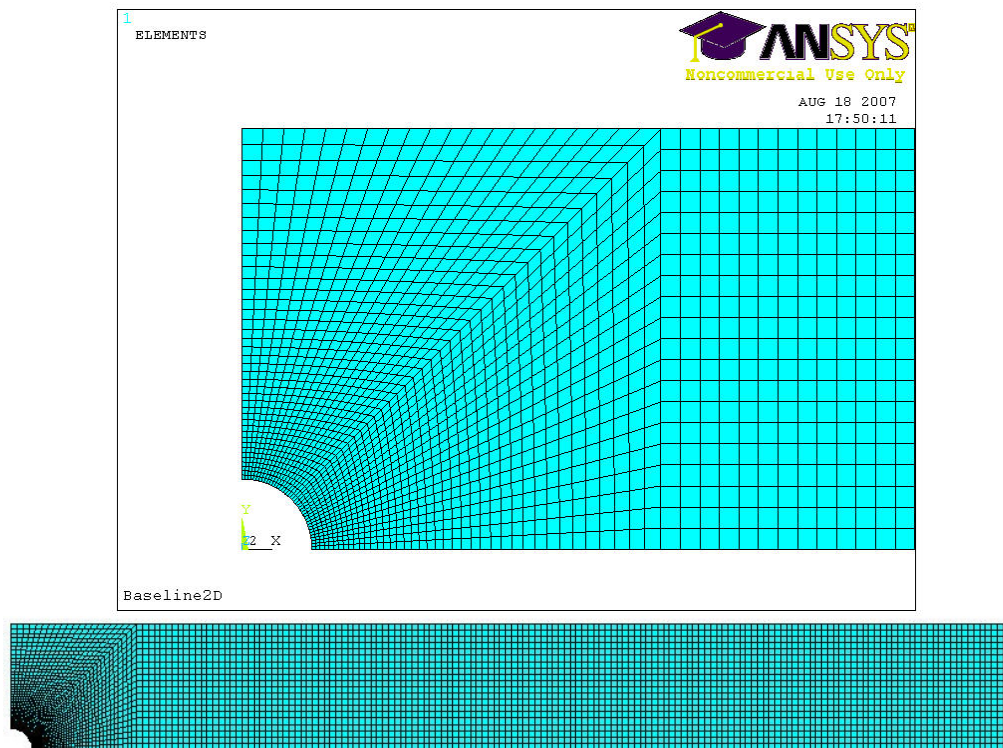
symmetric to this). The 2D FEA in Ansys models these plies at the same location, so that through-the-thickness effects are not accounted for. The plies can be easily altered for a given layup, and the computation is under a minute. Thus, the 2D FEA is ideal for analysis of a large number of layups, as in optimization considering all ply angles from 0° to 90°.



**Figure 2.1.4:** Ply orientations in Ansys

The final mesh geometry was found to be optimal during the development of the FEM, and contains 4,600 elements (Fig. 2.1.5). Other mesh shapes and configurations distorted the strain contours near the hole, thus the triangular mesh areas at the hole were extended to 0.75 inches from the hole center. At this mesh boundary, the stress concentration of the hole had dissipated enough so that the strain contours are not adversely affected by the mesh boundary. The convergence study (Appendix C) showed that further mesh refinement is not necessary, as the solution was not altered by adding more elements. The mesh was more refined at the hole due to the high strain gradients at the stress concentration, and gradually became less refined away from the hole.

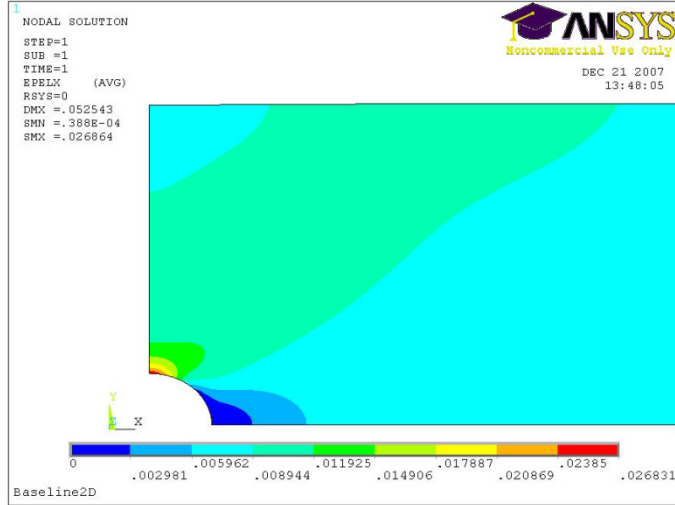




**Figure 2.1.5:** Final Mesh

## **2.2: Optimization Methods**

Two optimization methods were used: the reduction of the strain concentration factor,  $k_t$ , in order to increase the strength of the laminate, and the use of the Maximum Strain Criterion to increase the first ply failure load, FPF. The optimization curves for these methods are shown in Appendix D and are described in Section 2.4. To define the stress concentration, the maximum strain was taken at the top of the hole (0, 0.125), and the nominal strain as taken at a point away from all edges where the strain reaches a constant level (Fig. 2.2.1).



**Figure 2.2.1:** Strain Concentration

The stress concentration was defined as

$$k_t = \frac{\sigma_{\max}}{\sigma_{\text{nom}}} \quad (2.2.1)$$

where

$$\sigma_{\text{nom}} = \frac{P}{A_{\text{net}}} \quad (2.2.2)$$

and

$$A_{\text{net}} = A_{\text{gross}} - A_{\text{hole}} \quad (2.2.3)$$

Since Ansys reports a gross area stress as

$$\sigma_{\text{FEA},\text{nom}} = \frac{P}{A_{\text{gross}}} \quad (2.2.4)$$

a correction factor must be applied to get the net area stress or strain. Thus,  $k_t$  was

found from FEA results using

$$k_t = \frac{\sigma_{\max}}{\sigma_{\text{FEA},\text{nom}}} \frac{A_{\text{net}}}{A_{\text{gross}}} = \frac{\varepsilon_{\max}}{\varepsilon_{\text{FEA}}} \frac{A_{\text{net}}}{A_{\text{gross}}} \quad (2.2.5)$$

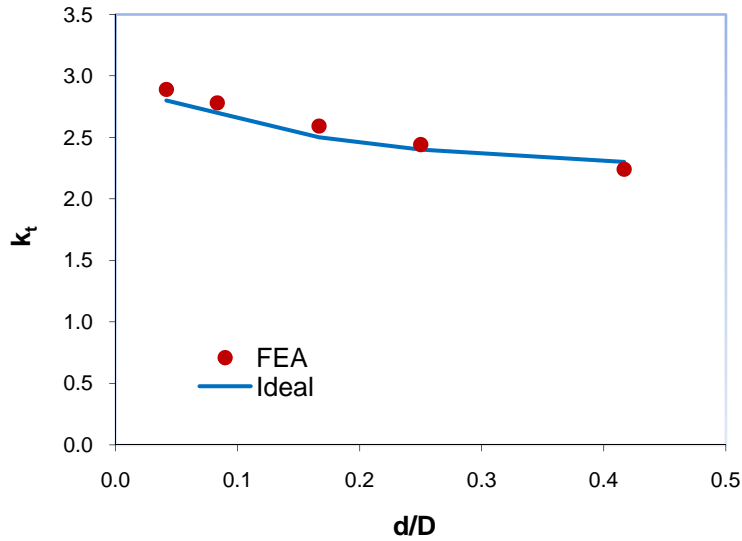
An alternative method for calculating  $k_t$ , sometimes used in industry, is to apply a width correction factor to a plate of infinite width. This is a more generally applicable method because a width correction factor can be applied to the infinite plate results for any number of geometries. If the appropriate correction factor is applied for the current geometry, the infinite plate  $k_t$  is approximately equivalent to the  $k_t$  calculated from eq. 2.2.5. The dimensions of the open-hole tension coupon did not change for the current study, so eq. 2.2.5 was used because it incorporates the coupon geometry and edge effects.

In two dimensional FEA, the stress depends on the ply orientation and changes for each ply, whereas the strain remains constant in the absence of through-the-thickness effects. Therefore, a stress-based  $k_t$  would require an average of the  $k_t$  for each ply, whereas a strain-based  $k_t$  would be equivalent to the  $k_t$  of any ply. Thus, the strain-based  $k_t$  was preferred for the current study so that only the strain data from the surface ply was needed.

### **2.3: Model Verification**

Aluminum material properties (Appendix A) were input into the FEM, and the  $k_t$  was found to be 2.59 for the given coupon geometry (Appendix B). This is near the analytical value of 2.50 for an isotropic material with the same hole/width ( $d/D$ ) ratio [41]. The width was then varied and compared to analytical values, as shown in Fig.

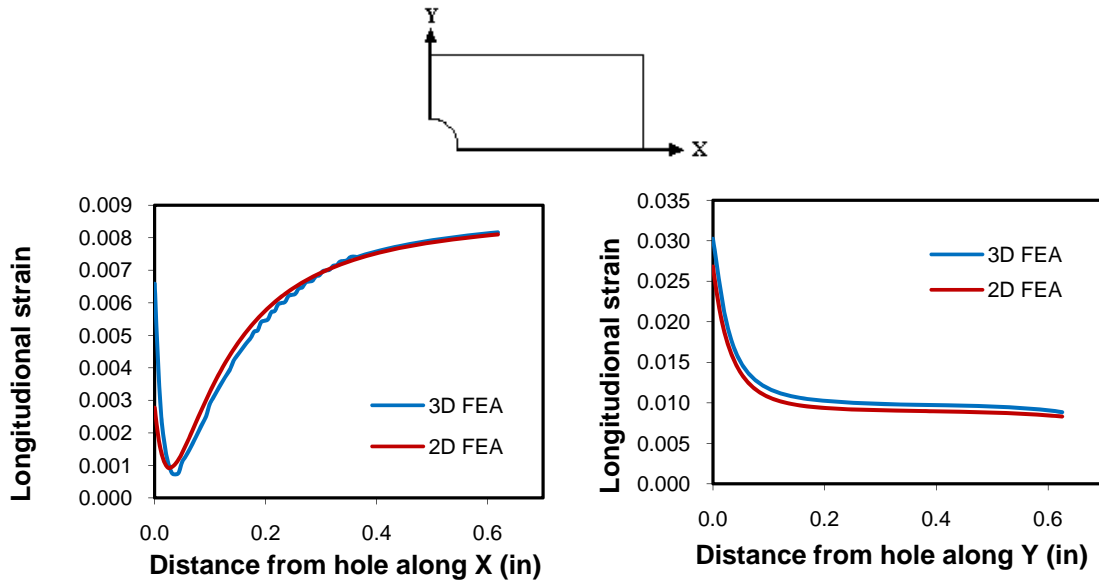
2.3.1.



**Figure 2.3.1:**  $k_t$  versus the hole-to-width ratio ( $d/D$ ) for an isotropic material

The 2D FEA was compared to the 3D FEA made by Kothidar [42]. The strain is compared along the longitudinal and transverse centerlines in Fig. 2.3.2. For the strains along X, the difference between the 2D and 3D solutions increases near the hole because of the free-edge and through-the-thickness effects, which are not accounted for in the 2D model. This causes the 3D strain to become 137% higher than the 2D at the minimum strain location (0.125, 0), and 13% higher at the maximum strain location (0, 0.125). Away from the hole, the solution completely converges along the longitudinal centerline, and converges to 6% along the transverse centerline. While the 137% higher 3D strain is a substantial difference at (0.125, 0), this is the location of minimum strain and so does not significantly affect  $k_t$  or FPF prediction, as those are dependent on the maximum strain and (for  $k_t$ ) the nominal strain. While the free-edge and through-the-thickness effects are important for the prediction of strain response and strength, the 2D model may be used for a fast approximation, as the computation time is greatly reduced in 2D FEA.

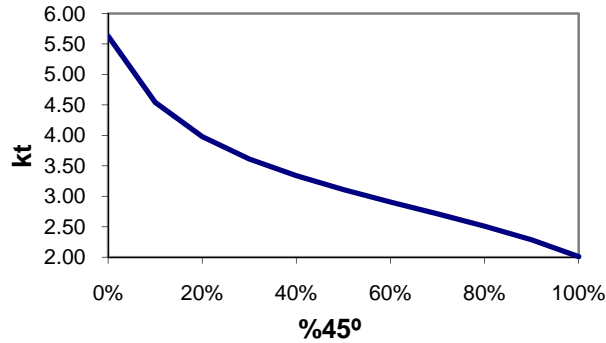
This computational time savings is pivotal for the theoretical strength optimization of a large number of non-traditional laminates.



**Figure 2.3.2:** Longitudinal strain vs. distance along X and Y

## 2.4: FEA Results

An initial FEA study was performed to increase the percentage of  $\pm 45^\circ$  plies, shown in Figure 2.4.1. The percentage of  $\pm 45^\circ$  plies was varied from 0 to 100% and the resulting  $k_t$  was plotted. The strain concentration decreased as  $\pm 45^\circ$  plies increased. This result agrees with physical intuition, as the  $\pm 45^\circ$  plies would disperse the strain concentration at strain maximum (the top of the hole in Fig. 2.2.1). This result shows that if non  $45^\circ$  plies are replaced with  $45^\circ$  plies in the baseline layup, the strain concentration may decrease.



**Figure 2.4.1:**  $k_t$  vs. the percentage of 45° plies for the baseline layup

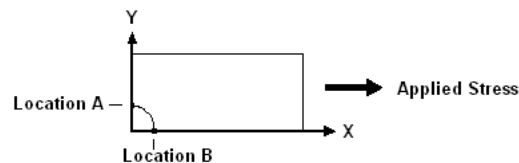
The Maximum Strain Criterion, discussed in Section 1.6, was used to predict FPF. This is because the Maximum Strain Criterion best predicted the initial fabricated set of  $k_t$ -optimized non-traditional laminates, and was thus used for the second fabricated set of FPF-optimized non-traditional laminates. The strain data from Ansys for a given layup was input into a failure criteria Matlab program (Appendix E), which included the Maximum Strain Criterion and several other common failure criteria, detailed in Section 1.6 and 1.7. The Matlab program was created with the use of Ref. [32]. The program inputs were the strain state at a locations of interest, the ply orientations of the laminate, and the load. The strain and corresponding stresses were transformed into the fiber direction coordinate system for each ply using the transformation matrix of equations 2.1.1, 2.1.2, and 2.1.4 from Ref. [26]. The stress or strain was then applied to each strength criterion for each ply. Since balanced orientations ( $\pm 45^\circ$ ) produced the same failure index, the absolute value of the ply orientations was used so that only one set of tensile and compressive strengths were needed for a given failure criterion and strain state. The in-plane allowable stresses were found in laboratory tests (Appendix A), described in Section 2.1. These strengths were divided by the appropriate modulus to get allowable strains for the strain based criteria. Nonlinear effects were ignored in the current study to simplify analysis and decrease computation time, but would have some

influence on the strain state and thus the failure criteria. Using a given failure criterion, the failure index was calculated and used to find strength by

$$s = \frac{\sigma_{\text{applied}}}{f.i.} \quad (2.4.1)$$

where  $\sigma_{\text{applied}}$  is the applied stress,  $s$  is the strength, and  $f.i.$  is the failure index. The outputs of the Matlab program were the  $k^{\text{th}}$  ply failure load and the corresponding failure mode<sup>6</sup> (for mode-based criteria).

The optimization methods were performed by varying the ply angle one degree at a time and calculating the  $k_t$  or FPF. The 2D FEM was chosen over the 3D FEM for this purpose due to the reduced computation time; the 2D model took under a minute to solve, whereas the 3D took at least a half hour. This time savings became especially important because of the very large number of layups considered during theoretical optimization (illustrated in Appendix D). The location of initial failure was found to be at Location A of Fig. 2.4.2 (as will be discussed in Chapter 5) where the maximum longitudinal strain is expected to occur. The minimum longitudinal strain was found to be at Location B. The optimized layups are given in Tables 2.4.1 and 2.4.2 with the corresponding  $k_t$  or FPF values reported at the bottom. These values were also reported as normalized to the baseline layup, to show the extent of improvement for  $k_t$  or FPF that can be achieved during optimization.



**Figure 2.4.2:** FEA Coordinates and locations of interest

<sup>6</sup> To see the failure modes in the Matlab program, the semicolons must be removed from the appropriate lines of code.

**Table 2.4.1:** Method 1: reduce  $k_t$ 

	<b>Baseline</b>	<b>Vary 45</b>	<b>Vary 0</b>	<b>Vary 90</b>	<b>Vary 0&amp;45</b>	<b>Vary 0&amp;90</b>	<b>Vary 45&amp;90</b>
	45	54	45	45	62	45	54
	90	90	90	51	90	52	54
	-45	-54	-45	-45	-62	-45	-54
	0	0	57	0	62	52	0
	45	54	45	45	62	45	54
	90	90	90	-51	90	-52	-54
	-45	-54	-45	-45	-62	-45	-54
	0	0	-57	0	-62	-52	0
Line of Symmetry -----							
<b><math>k_t</math></b>	<b>2.58</b>	<b>2.51</b>	<b>1.94</b>	<b>2.39</b>	<b>1.77</b>	<b>1.77</b>	<b>2.35</b>
<b>Normalized</b>	<b>1.00</b>	<b>0.97</b>	<b>0.75</b>	<b>0.93</b>	<b>0.68</b>	<b>0.69</b>	<b>0.91</b>

**Table 2.4.2:** Method 2: increase FPF

	<b>Baseline</b>	<b>Vary 45</b>	<b>Vary 0</b>	<b>Vary 90</b>	<b>Vary 0&amp;45</b>	<b>Vary 0&amp;90</b>	<b>Vary 45&amp;90</b>
	45	21	45	45	21	45	4
	90	90	90	0	90	0	1
	-45	-21	-45	-45	-21	-45	-4
	0	0	0	0	0	0	0
	45	21	45	45	21	45	4
	90	90	90	0	90	0	-1
	-45	-21	-45	-45	-21	-45	-4
	0	0	0	0	0	0	0
Line of Symmetry -----							
<b>FPF</b>	<b>1295</b>	<b>1476</b>	<b>1295</b>	<b>2297</b>	<b>1476</b>	<b>2297</b>	<b>5736</b>
<b>Normalized</b>	<b>1.00</b>	<b>1.14</b>	<b>1.00</b>	<b>1.77</b>	<b>1.14</b>	<b>1.77</b>	<b>4.43</b>

Some layups were redundant (such as the FPF: Vary 45 and the FPF: Vary 0&45), and only one set was manufactured. A uniaxial layup was fabricated in place of the FPF vary 45&90 because of the similarity of these layups, and in order to study the uniaxial failure mode. The  $k_t$ : Vary 0&45 and  $k_t$ : Vary 0&90 layups were not made because the initial  $k_t$ : Vary 0 tests showed that varying the primary load bearing ( $0^\circ$ ) plies only reduced strength for both FPF and UTS. The final 11 layups that were fabricated are given in Table 2.4.3, along with the optimization method and corresponding description.

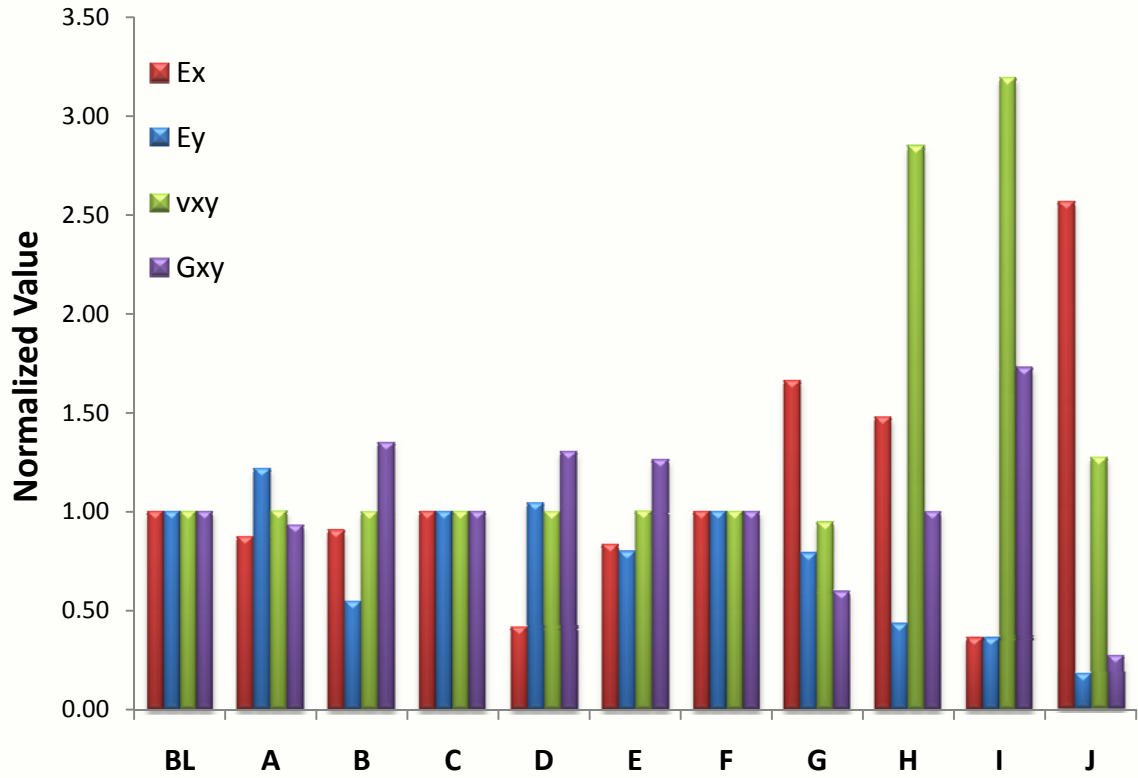


**Table 2.4.3: Fabricated layups<sup>7</sup>**

<b>Designation</b>	<b>Layup</b>	<b>Method</b>	<b>Description</b>
BL	[(45/90/-45/0) <sub>2</sub> ] <sub>s</sub>	Quasi-isotropic	Baseline
A	[(54/90/-54/0) <sub>2</sub> ] <sub>s</sub>	k <sub>t</sub>	Vary 45°
B	[(45/51/-45/0) <sub>2</sub> ] <sub>s</sub>	k <sub>t</sub>	Vary 90°
C	[(45/0/-45/90) <sub>2</sub> ] <sub>s</sub>	Stacking sequence	Vary Order 1
D	[(45/90/-45/57) <sub>2</sub> ] <sub>s</sub>	k <sub>t</sub>	Vary 0°
E	[(54/54/-54/0) <sub>2</sub> ] <sub>s</sub>	FPF	Vary 45° & 90°
F	[(45/-45/90/0) <sub>2</sub> ] <sub>s</sub>	Stacking sequence	Vary Order 2
G	[(21/90/-21/0) <sub>2</sub> ] <sub>s</sub>	FPF	Vary 45°
H	[(45/0/-45/0) <sub>2</sub> ] <sub>s</sub>	FPF	Vary 90°
I	[(±45) <sub>4</sub> ] <sub>s</sub>	Failure mode	Shear failure
J	[0] <sub>16</sub>	Failure mode	Uniaxial failure

The optimization schemes did not impose limits on the variation in material properties of the resulting non-traditional laminates compared to baseline. The predicted values of the material properties (using CLT) are given in Appendix A, and are shown in Fig. 2.4.3 normalized to baseline. The material properties of the I and J layups vary the largest, as expected for failure-mode layups that are designed to focus on a specific material property. While some layups do not vary much from baseline, others show large variation, such as the H layup. These variations will be discussed and compared to strength results in Chapter 5. Imposing limits during the optimization process (such as a limit of 10% variation in modulus that might be imposed in industry), could easily be done using CLT. However, for the current study, these limits were not imposed in order to understand the full effect of the optimization methods.

<sup>7</sup> The layups are all balanced (the same number of +/- plies on either side of the center line).

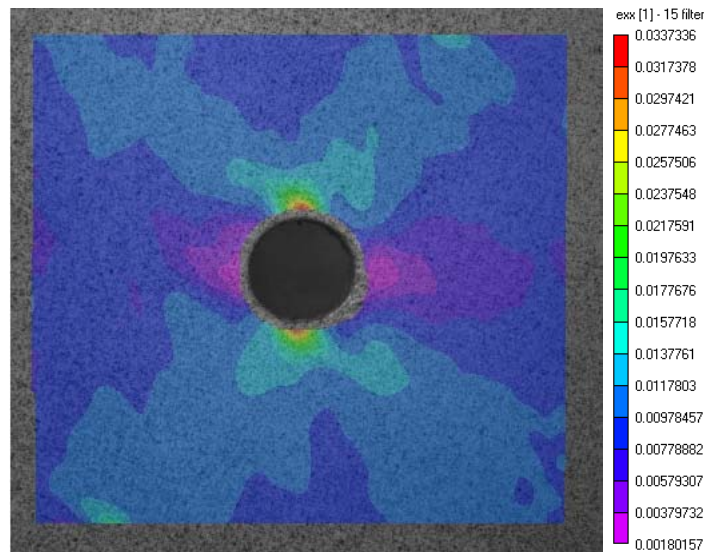


**Figure 2.4.3:** Material properties from CLT of non-traditional laminates normalized to the baseline (BL) layup

### 3: Digital Image Correlation Method

#### 3.1: Experimental Technique

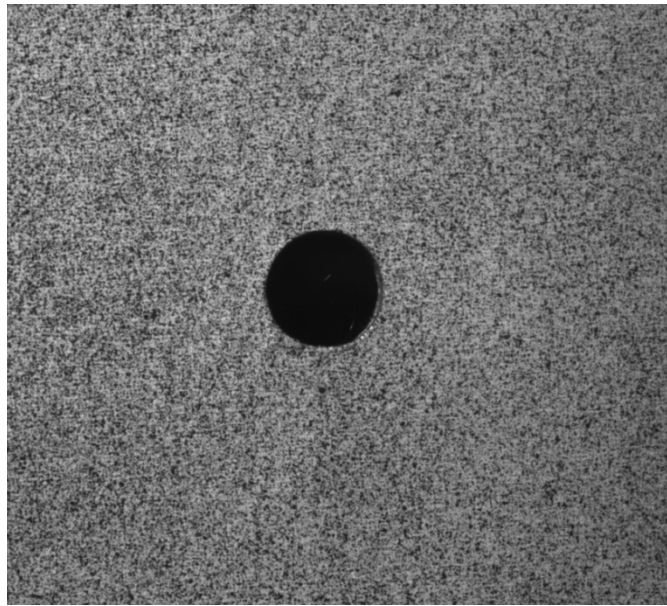
The digital image correlation method (DICM) allows displacements and strains to be measured across the entire surface of the coupon (Fig. 3.1.1). While computer based methods for optics, imaging, and metrology have developed for decades, recent advances in digital image processing have allowed DICM to become increasingly useful. While methods such as Moiré interferometry and the holographic technique have been shown to successfully measure full-field surface strains, these methods require special preparations and environmental control.



**Figure 3.1.1:** Longitudinal strain (tensile loading is horizontal)

DICM correlates images by matching the pixel grey-level values of a reference image and a deformed image. To initiate this process, a speckle pattern (Fig. 3.1.2) is applied to the coupon to create a random distribution of intensity of grey levels, which is supplied to the software. Regardless of how the speckle is applied, it must deform with the sample, and must not reinforce the specimen. The speckle pattern should be random, uniform, and

high contrast, and must not degrade during coupon deformation [43]. A satisfactory speckle can be obtained using spray paint cans (white base with black dots); however, a finer speckle pattern can be generated using a commercial air sprayer, as was done in the current study.



**Figure 3.1.2:** Speckle pattern

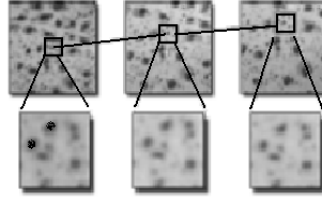
DICM was successfully used for length scales of  $10^{-9}$  m to  $10^2$  m, strains up to 400%, and time scales up to 5,000 frames per second [43]. Helm et. al. [44] used digital image correlation to measure strain in 2.4 m concrete beams reinforced with carbon/epoxy composites and loaded to 95% of the beam's ultimate load. The Digital Image Correlation Method was able to obtain continuous data, creating a more accurate shear stress plot than could be interpolated from electrical resistance strain gage data. Because of this spatial data, DICM was found to be particularly suited to study strain heterogeneities. Lagattu et. al. [45] used digital image correlation to measure the strain gradients around a hole in a composite laminate, at a crack tip in a TiAl alloy, and on a polymer neck front. For the composite with a hole, Lagattu found four quasi-symmetric

zones of high strains along the  $\pm 45$  axes. Lagattu et. al. concluded that DICM was able to characterize high strain gradient heterogeneities in materials. Ambu et. al. [46] applied DICM to static and fatigue tensile loading of graphite/epoxy laminates with circular holes for different stacking sequences. Strain distribution near the hole was found to be significantly affected by fatigue damage. Ambu et. al. [46] found that DICM could be used to study the effect of failure modes in composite laminates and that strain was redistributed around the hole during failure progression.

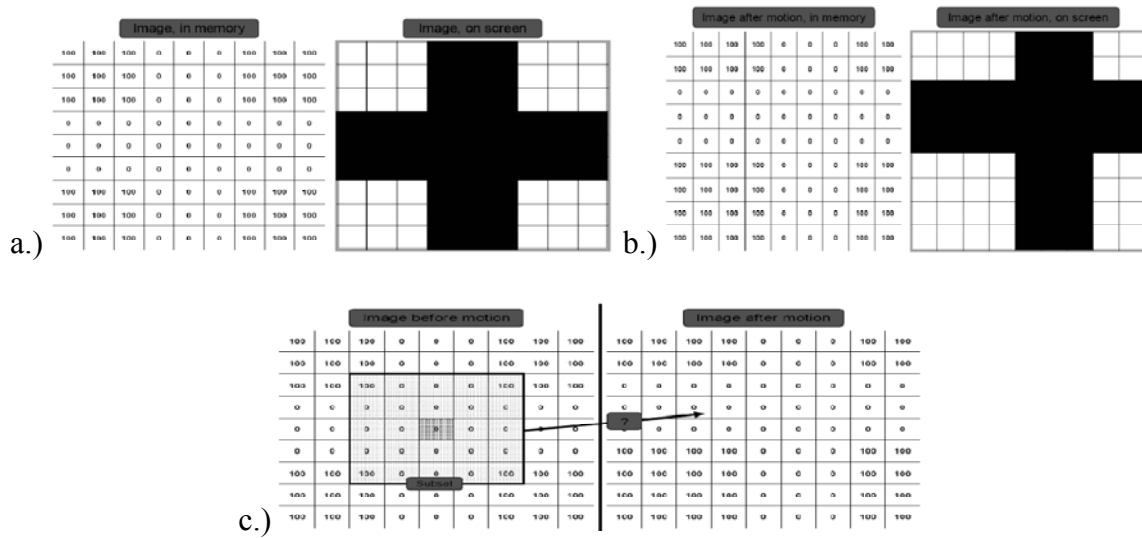
The current study used commercial DICM software from Correlated Solutions, Inc. Once the speckled images were acquired, the software discretized the digital images into subsets, which were  $N \times N$  pixel squares. The reference image subsets were matched with their equivalent subsets in the deformed images (Fig. 3.1.3), creating a displacement field over the coupon surface. Subset matching was accomplished with a correlation function  $C$ , such as sum of squared differences [43]

$$C(x, y, u, v) = \sum_{i, j = -n/2}^{n/2} (I(x+i, y+j) - I^*(x+u+i, y+v+j))^2 \quad (3.1.1)$$

where  $x$  and  $y$  are the pixel coordinates of the reference image,  $u$  and  $v$  are the displacements along the  $x$  and  $y$  axis,  $I$  and  $I^*$  are the images before and after motion describing the grey level intensity,  $n$  is the subset size, and  $i$  and  $j$  are the increments. Smaller values for the correlation function meant better similarity for the subset matching (Fig. 3.1.4).



**Figure 3.1.3:** Image correlation [43]



**Figure 3.1.4:** Image correlation showing a.) the reference image b.) the deformed image and c.) the subset matching [43]

An error function was reduced using an optimizing method, and such methods include coarse-fine search, Fast-Fourier Transforms (FFT), genetic algorithms, and iterative optimizers (newton\_Raphson methods, conjugate gradients, trust region methods). However, these methods often have drawbacks; coarse-fine search was found to be too slow, FFT methods were limited, and genetic algorithms were found to be robust but slow [43]. Bi-linear B-Spline interpolation was used for the optimization method by Correlated Solutions, Inc., which was shown in [47] to be effective for DICM. The B-spline function represents the object deformation field, and optimizes the intensity matching between two images. The equations for the B-spline function require an in-

depth discussion, and the reader is referred to Ref. [47] for a description of these and the equations for displacements and strains in DICM.

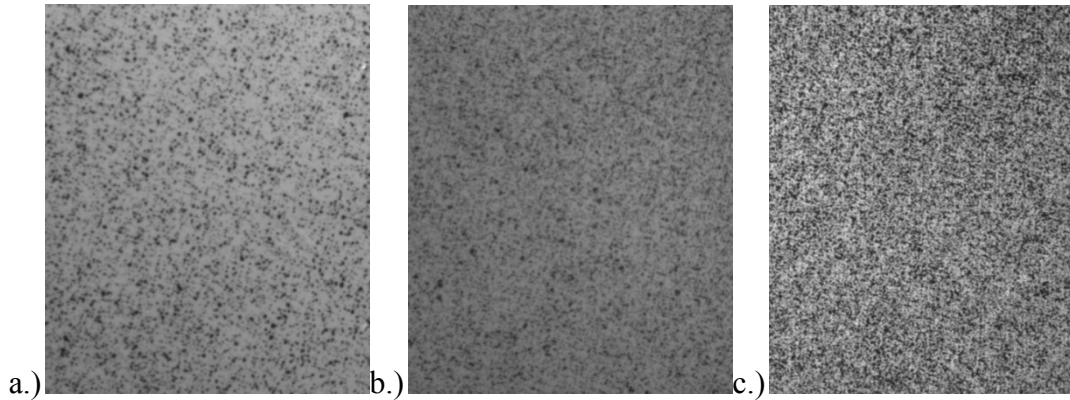
Vic3D data analysis software from Correlated Solutions was used in the present research. In Vic3D, the displacements are calculated in the center of each subset. The subset is defined as an  $n \times n$  pixel window that surrounds a data point and represents a strain window. During correlation, optimization equations are used to find out where the window is in the deformed image, and the deformation is reported. The strains can then be calculated from the deformation for each data point. The subsets are spaced by a user-defined number of pixels, or step size. Subsets may overlap for a small stepsize, so that the step size represents the increments, or steps, that the subset (strain window) is shifted during correlation. A smaller step size yields more data points and also more computation time. The displacement calculation is smoothed typically using a distortion order of 2 for the data points in the subset. The in-plane displacement vector is calculated at every subset center, and the strain is calculated from the triangle formed by three displacement points. Due to variations inherent to experimental analysis, Vic3D uses a robust correlation function, including a subset shape function from (similar to FEA) to define the deformed subset. Displacements must be transformed into a useful coordinate system, using calibration techniques [44].

As noted by Lagattu et. al. [45], measuring high strain gradients with DICM requires a balance of parameters such as the speckle pattern, subset size, studied region size, data point spacing, and measurable strain range. Once these parameters are accounted for, DICM can be a useful tool to study high strain gradients such as those found around a circular hole. As failure progresses, the redistribution of surface stress due to

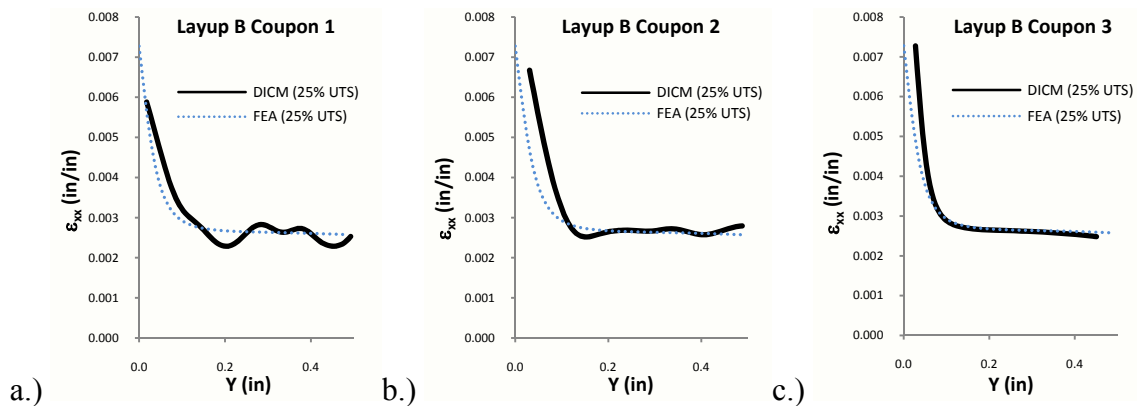
delamination or fiber pullout can be observed. Full-field strains around the hole for the entire loading range of a laminate can be compared for varied layups.

In the current study, a balance of parameters was found to be optimum during the DICM data reduction. The final speckle pattern, shown in Fig. 3.1.2, worked well because it was non-repetitive, isotropic, and high contrast. Speckle patterns with large white or black areas did not correlate as well because larger dots required larger subsets. Speckle patterns with a grey background also did not correlate well because of the lack of contrast. The subset could not be smaller than the largest black or white region, or the correlation function would have no variation in values to match between subsets. Fig. 3.1.5 shows different speckle patterns, and Fig. 3.1.6 shows the corresponding strain profiles. The strain profiles in Fig. 3.1.6 are more consistent for the more isotropic and higher contrast speckle patterns, as shown by the smoother lines. This trend in correlation was shown for nearly all of the coupons tested, and thus the speckle pattern was found to be one of the most important aspects of the test setup. Calibration was also clearly important, and better correlation was found when the two cameras recorded the same field of view. This could be achieved by locating dots in the speckle pattern, noting their distance to the edges of the image, and matching the distance of the two camera images.





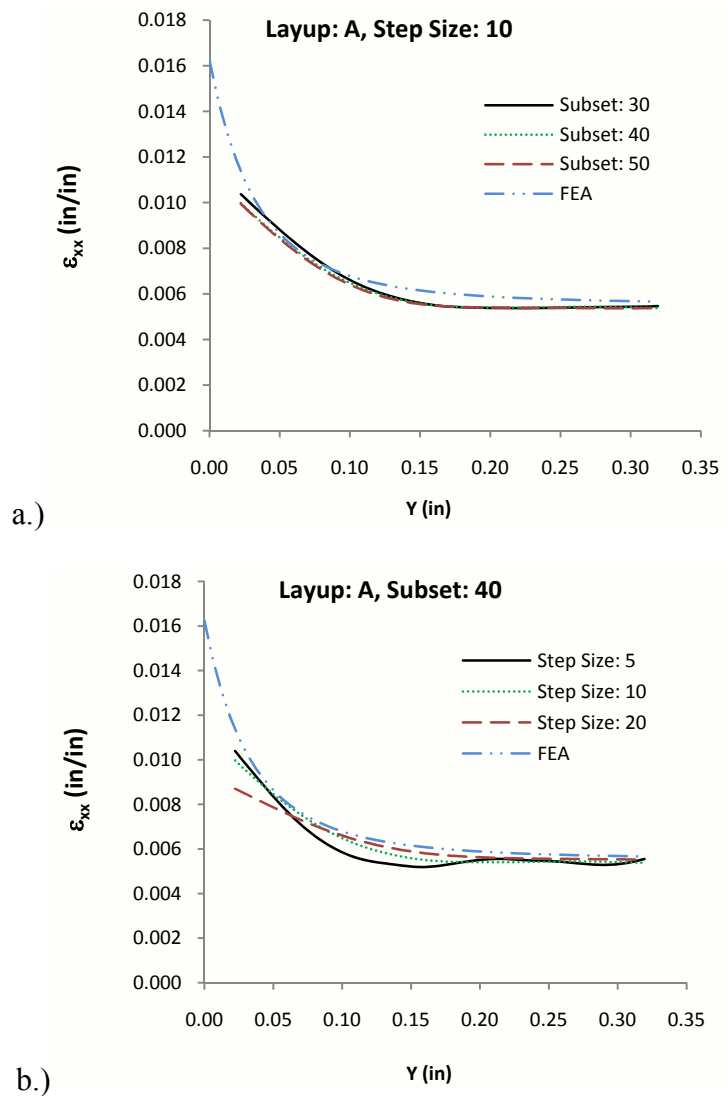
**Figure 3.1.5:** Speckle patterns for Layup B showing a.) a mediocre pattern (coupon1), b.) a better pattern (coupon 2), and c.) a good pattern (coupon 3)



**Figure 3.1.6:** Strain profiles for Layup B for a.) coupon 1, b.) coupon 2, and c.) coupon 3, corresponding to the speckle patterns of Fig. 3.1.5a-c and showing improved correlation for speckle patterns that are isotropic, high contrast, and with smaller areas of black and white.

A subset size of 40 was used in the current study. A larger subset would be needed if larger black or white areas existed in the speckle pattern, but was not necessary because a larger subset of 50 produced the same solution as 40 (Fig. 5.1.7a). DICM can only go half a subset size from the edge of the hole, which was 0.022 inches for a subset of 40, using CCD cameras (4 mega pixel resolution) and 50mm Schneider lenses zoomed to their maximum. The step size was 10, chosen because a larger size reduced detail but a smaller size significantly increased the computation time, from 10 minutes to over an hour (Fig.

3.1.7a). A smaller step size (5) did not significantly improve the gradient near the hole (at 0.022 inches), and a larger step size (20) showed poor correlation near the hole.

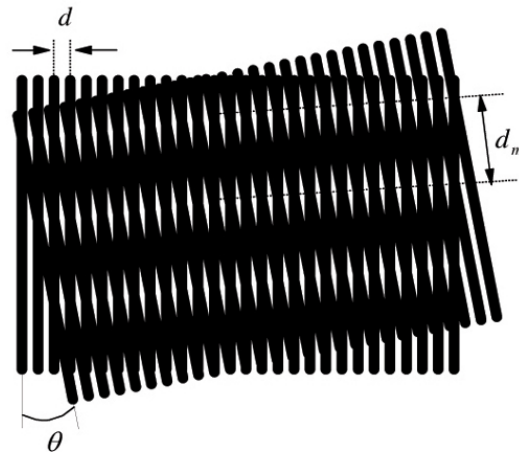


**Figure 3.1.7:** DICM parameter study for a.) Subset, and b.) Step Size

### 3.2: Moiré Interferometry

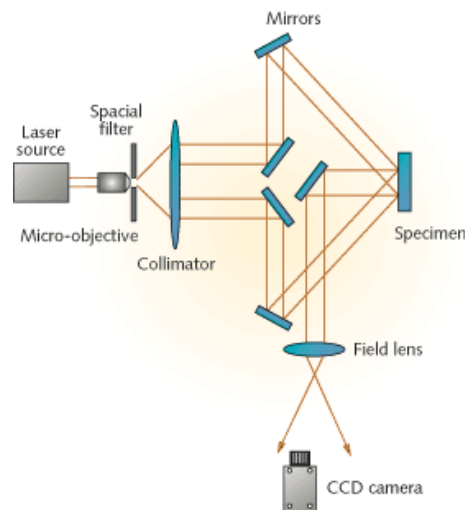
Moiré Interferometry (MI) is a well developed technique for the measurement of full-field surface strains, and is useful for the verification of DICM. The strain field is measured at static loads within the ranges allowed by the cameras, diffraction grating, and experimental setup. Strain contour surfaces can typically be acquired at any

wavelength in the range of  $10\ \mu\text{m}$  to  $100\ \mu\text{m}$ . The diffraction grating is applied to the surface of the coupon resulting in a fringe pattern, shown in Fig. 3.2.1.



**Figure 3.2.1:** Moiré Interferometry fringe pattern [55]

The fringe pattern is illuminated with a laser light source, which is split into two horizontal and two vertical beams. When the specimen is loaded, the deformation of the specimen also deforms the fringe pattern at the surface, changing the diffracted beams. The incident angle of the four beams result in  $\pm 1$  diffraction orders that recombine normal to the specimen, shown in Fig. 3.2.2.



**Figure 3.2.2:** Moiré Interferometry measurement technique [56]

The recombined beams consist of interference patterns representing the deformation of the coupon surface. The fringe patterns thus correspond to the in-plane displacements. In order to measure longitudinal deformation, two horizontal beams are blocked using a filter, and the two vertical beams interfere. This generates a vertical deformation field, or V-Field. Similarly, in order to measure transverse deformation, two vertical beams are blocked using a filter, and two horizontal beams interfere. This generates a horizontal deformation field, or U-Field. The deformation fields can be related to in-plane strains using [48, 49]

$$\varepsilon_x = \frac{\partial U}{\partial x} = \frac{1}{f} \left[ \frac{\partial N_x}{\partial x} \right] \quad (3.2.1)$$

$$\varepsilon_y = \frac{\partial V}{\partial y} = \frac{1}{f} \left[ \frac{\partial N_y}{\partial y} \right] \quad (3.2.2)$$

$$\gamma_{xy} = \frac{\partial U}{\partial y} + \frac{\partial V}{\partial x} = \frac{1}{f} \left[ \frac{\partial N_x}{\partial y} + \frac{\partial N_y}{\partial x} \right] \quad (3.2.3)$$

where  $f$  is the virtual grating frequency,  $N_x$  is the horizontal fringe order, and  $N_y$  is the vertical fringe order. The fringe orders are retrieved from the unwrapped phase distribution of the corresponding Moiré interferometric fringe pattern.

MI was used to measure deformation in composite multi-span beam shear specimens in [50]. Analysis of composite beams under 5-point and 3-point flexure showed free-edge effects produced shears on the edge of the beams, and that MI performed well for whole-field strain analysis of composites. A fourier-transform method for analyzing fringe patterns of Moiré Interferometry was performed by [51]. Surface strain measurements of

carbon-fiber/PEEK composite laminates were successfully achieved up to 37.9% the theoretical limit of the fringe pattern.

Moiré and DICM are compared in Chapter 5. While both accurately measure strains in the elastic region of composite laminates, Moiré was not able to obtain strains in the damaged region. DICM has the advantage of continuous measurement and correlation, while Moiré generally has higher accuracy, and can measure strains from 0.002 to 2%. DICM has limits of measurement at extremely low strains due to the limit of the CCD cameras, as will be shown in Chapter 5. However, Moiré can only take measurements at static loads, and precautions must be taken to minimize vibrations from the equipment and the building. Thus, DICM is more useful for measuring the surface strain of many laminates over their entire load range.

## **4: Manufacturing**

### **4.1: Prepreg**

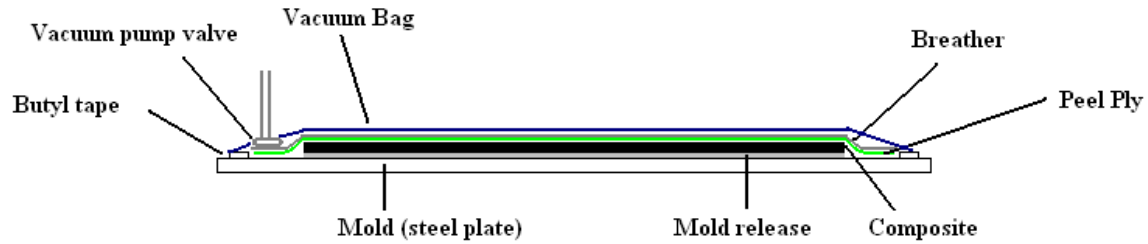
Prepreg, short for preimpregnated, is a combination of fiber and resin made ready for the fabrication and final curing of a composite part. In the manufacturing of the prepreg, the resin and curing agent (mixed thoroughly and called the matrix) are impregnated into the fiber. The fiber may be carbon, graphite, fiberglass, organic, etc., chosen for the desired strength and stiffness properties. The prepreg is made into either woven fabrics, roving, or unidirectional tape. Fabrics and tapes are made into rolls which can vary in size up to 72 inches wide and several hundred feet long. Roving is used for filament winding, which is not discussed here. Once the resin has been impregnated onto the fibers and made into a final roll, the prepreg must be stored in a freezer until fabrication to slow the resin cure. The prepreg is supplied to the fabricator who completes the cure using heat, pressure, and vacuum bagging to remove excess resin.

Thermoset or thermoplastic matrices can be used. The thermoset prepreg is partially cured (B-stage cure) before fabrication. Resin in a thermoplastic prepreg is melted polymer or polymer in solvent. Prepregs are convenient because of a well controlled fiber to resin ratio, easy storage, controlled resin flow during curing, and the possibility of hand fabrication of layups.

### **4.2 : Fabrication**

Fabrication (Fig. 4.2.1) consisted of the layup, mold, and autoclave. Creating a layup, such as the baseline  $[45/90/-45/0]_s$ , involved cutting and laying the correct angles. Care was taken to use gloves and sterilize the mold (a steel plate). A mold release agent was first spread across the mold, and the plies were then added to make the layup. On top of

the layup, peel ply was added which allowed the resin to bleed through during cure, and did not stick to the composite. Breather cloth was put on top, and the vacuum nozzle was placed on top of this. Butyl tape was placed in a rectangle surrounding the part, and vacuum bagging material was sealed on top of everything, with a slit for the vacuum tube.



**Figure 4.2.1:** Fabrication setup

The autoclave was set to a specific cure cycle, as prescribed in literature. This cure cycle starts with a ramp period, in which temperature and pressure were increased at a constant rate. The part was soaked at a specific temperature and pressure for a given time, followed by a ramp down period. For the current study, open-hole tension coupons were fabricated from 16 plies of carbon/epoxy prepreg (T600/125-33). All coupons were autoclave cured at 350 °F and 90 psig. After a 10 minute ramp period, the coupons were then soaked at this temperature and pressure for 45 minutes and then slowly cooled (under a constant pressure of 90 psig) for an additional 45 minutes. Coupons were machined from fabricated plaques using a waterjet system and an aerospace quality abrasive mixture. The holes were drilled with a drill press, as would be encountered in industry. The effect of this process is discussed in Chapter 5.

## 5: Results

### 5.1: Introduction

Linear finite element analysis was applied to kt- and FPF-optimization methods to tailor non-traditional laminates in open-hole tension (Chapter 2). Some layups were not fabricated because they were either repetitive or irrelevant (Section 2.4). The final eleven layups, shown in Table 5.1.1, were fabricated according to the techniques explained in Chapter 4 and the dimensions given in Appendix B. The coupons were speckled and loaded in an MTS load frame (10 kip load cell) during which DICM surface strain measurements (Chapter 3) were taken.

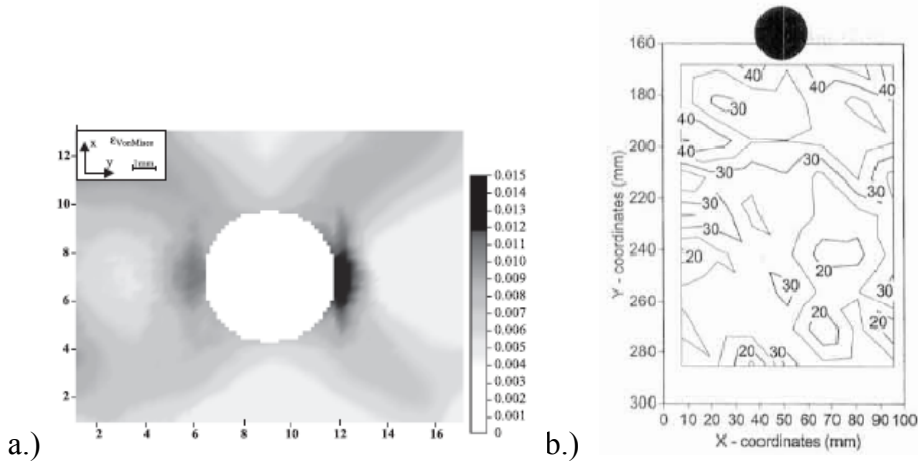
**Table 5.1.1:** Fabricated layups<sup>8</sup>

Designation	Layup	Method	Description
BL	[(45/90/-45/0) <sub>2</sub> ] <sub>s</sub>	Quasi-isotropic	Baseline
A	[(54/90/-54/0) <sub>2</sub> ] <sub>s</sub>	k <sub>t</sub>	Vary 45°
B	[(45/51/-45/0) <sub>2</sub> ] <sub>s</sub>	k <sub>t</sub>	Vary 90°
C	[(45/0/-45/90) <sub>2</sub> ] <sub>s</sub>	Stacking sequence	Vary Order 1
D	[(45/90/-45/57) <sub>2</sub> ] <sub>s</sub>	k <sub>t</sub>	Vary 0°
E	[(54/54/-54/0) <sub>2</sub> ] <sub>s</sub>	FPF	Vary 45° & 90°
F	[(45/-45/90/0) <sub>2</sub> ] <sub>s</sub>	Stacking sequence	Vary Order 2
G	[(21/90/-21/0) <sub>2</sub> ] <sub>s</sub>	FPF	Vary 45°
H	[(45/0/-45/0) <sub>2</sub> ] <sub>s</sub>	FPF	Vary 90°
I	[(±45) <sub>4</sub> ] <sub>s</sub>	Failure mode	Shear failure
J	[0] <sub>16</sub>	Failure mode	Uniaxial failure

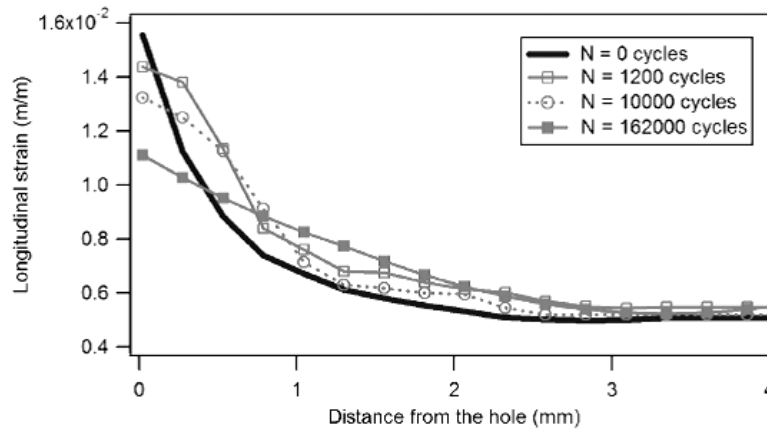
DICM has been used by others to quantitatively and qualitatively characterize surface strain. DICM can be used to observe the strain concentration near a hole, as shown in Fig. 5.1.1, where the strain contours of a composite laminate (5.1.1.a) and a wood specimen (5.1.1.b) are given. Fig. 5.1.2 shows the strain profile along the net section of a composite laminate in fatigue. The strain contours can be used to understand strain patterns around the hole, and the strain profile can be used to quantify those observations.

<sup>8</sup> The layups are all balanced (the same number of +/- plies on either side of the center line).





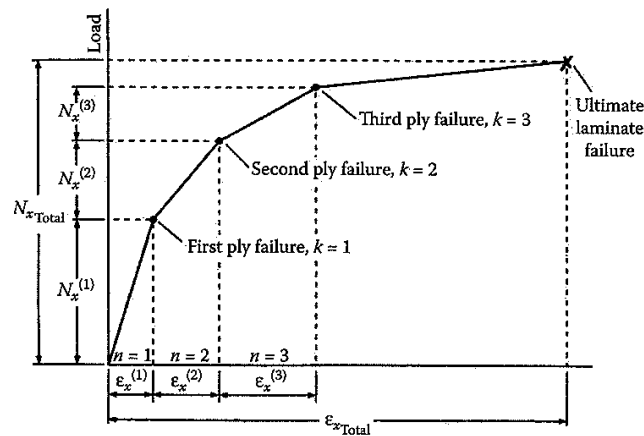
**Figure 5.1.1:** a.) Von Mises strain map measured by DICM around a 5 mm hole on a thermoplastic matrix composite,  $\sigma_{\text{applied}}=67\% \sigma_{\text{failure}}$  (vertical loading) [45] b.) Transverse displacement ( $\text{mm} \times 10^{-3}$ ) contour plots by DICM near a bolt (vertical load) [52].



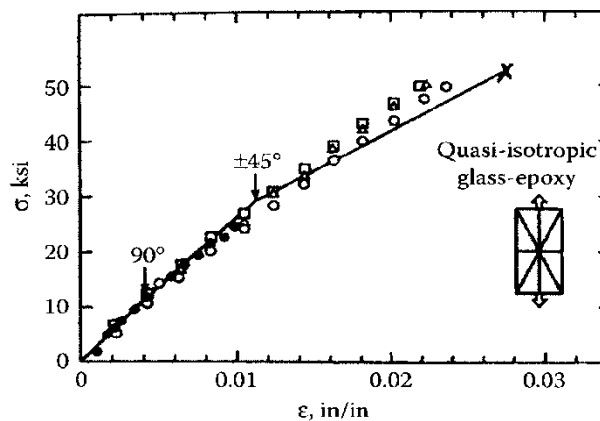
**Figure 5.1.2:** Longitudinal strain versus distance from the hole at the net section of a fatigued  $[0/\pm 45/90]_{2s}$  graphite/PEEK sample; maximum load=75% of the ultimate static load [46].

Experimental measurement of FPF can be a difficult task. There is no generally accepted experimental method for determining FPF, such as the 2% offset rule for yield in some metals. The idealized load-strain curve displays several piecewise linear sections separated by “knees” that correspond to the  $k$ th ply failure (Fig. 5.1.3). In reality, the changes in stiffness are often more subtle for first ply failure (Fig. 5.1.4), and may be

nonlinear for subsequent ply failures. In the World Wide Failure Exercise [1], results for initial failure were poor because of a shortage of reliable experimental data, and because of discrepancies between experimental and theoretical definitions of initial failure. Experimental analysis of FPF is further complicated by the inherent variations in manufacturing. Stress-strain plots for coupons of the same layup will vary as damage progresses, as shown by the diverging data points of Fig. 5.1.4. These issues will be discussed, and experimental methods will be suggested, in Section 5.4.



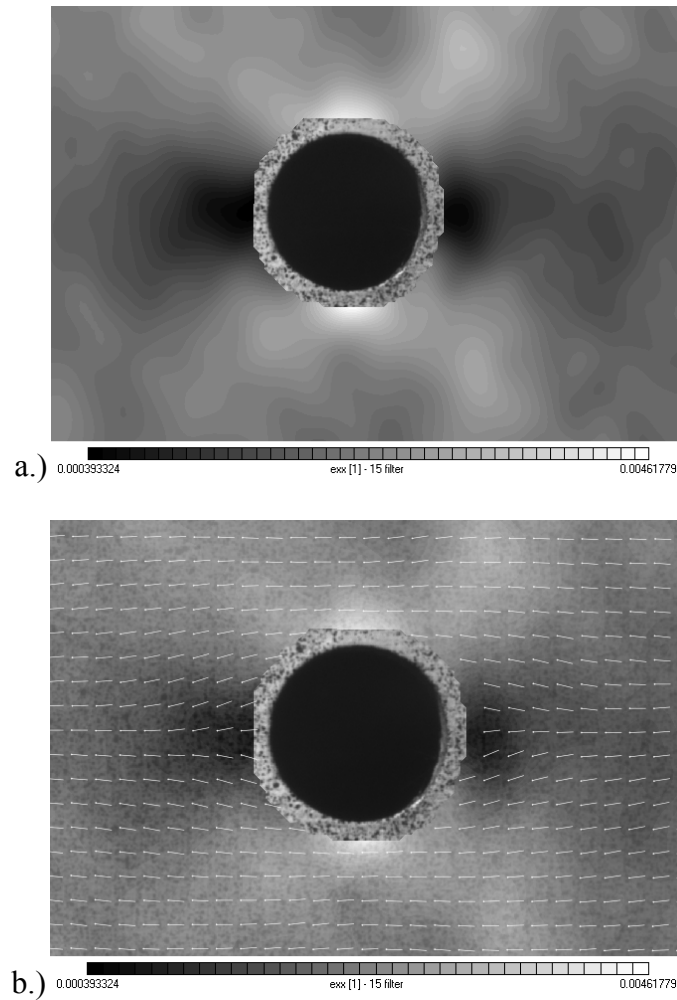
**Figure 5.1.3:** Idealized load-strain curve for uniaxially loaded laminate showing multiple ply failures leading up to ultimate laminate failure. [26]



**Figure 5.1.4:** Comparison of predicted and measured stress-strain response of  $[0\pm45/90]_s$  glass/epoxy laminate. [53]

## 5.2: Strain Profile

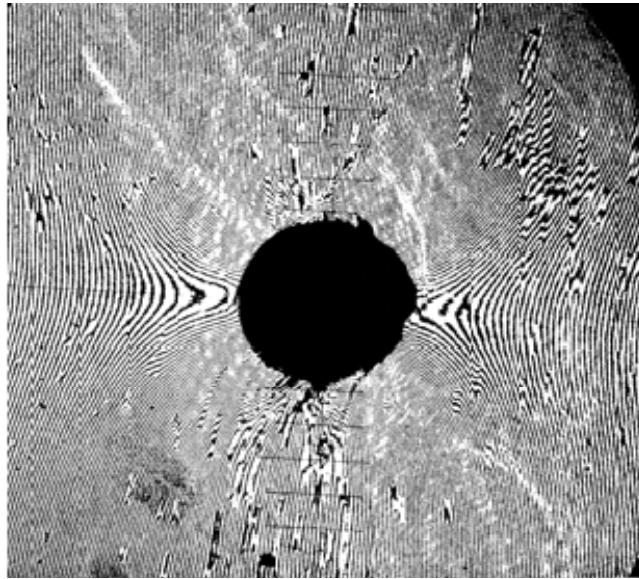
The strain profile near the hole for the eleven layups of Table 5.1.1 was generated using DICM. Several coupons were tested for each layup, and the surface strain was correlated at 6 second increments until failure. Due to the large volume of strain data that was produced, only a representative portion can be presented here. The DICM strain contour plot for the Baseline layup is shown in Fig. 5.2.1a, where the longitudinal strain and tensile loading directions are horizontal. The contours clearly show maximum and minimum strain locations at Location A and B (Fig. 2.4.1). Maximum strain zones can be seen stemming out from the maximum strain location (Location A) at approximately a  $\pm 45^\circ$  angle. These contour patterns are representative of the non-traditional layups tested, and selected examples are shown in Appendix F Fig. F.1 and Fig. F.2. The only exception to this general pattern occurred in the G laminate (App. F Fig. F.3), where the maximum strain location was shifted slightly, even in the linear region. This occurs because the G layup has  $21^\circ$  plies at the surface, so that the surface strain measured by the DICM was influenced by the strain state of the  $21^\circ$  surface ply. Additional selected contour plots are given in Appendix F. In Fig. 5.2.1b, the maximum principle strain is shown around the hole. As expected, the strain flows around the hole from the minimum zone to the maximum zone, causing the strain concentration to occur at Location A (Fig 2.4.1).



**Figure 5.2.1: a.)** Longitudinal strain DICM contours and **b.)** Principle strain vectors for the Baseline laminate at 19 Ksi (26% UTS), where tensile loading is horizontal.

Moiré Interferometry deformation contours are shown in Fig. 5.2.2 for the Baseline layup. For Moiré interferometry, lines closer together indicate high deformation zones and correspond to the high strain zones of DICM shown in Fig 5.2.1. Thus, the high strain zones at the top and bottom of the hole and the low strain zones at the sides of the hole of Fig. 5.2.1 from DICM correspond well to the high and low deformation zones of Moiré Interferometry shown in Fig. 5.2.2. However, the quality of contours at the top and bottom of the hole in Fig. 5.2.2 degrades, perhaps due to damage introduced by drilling the hole. As noted by [10], the drilling process may introduce delimitation, which would

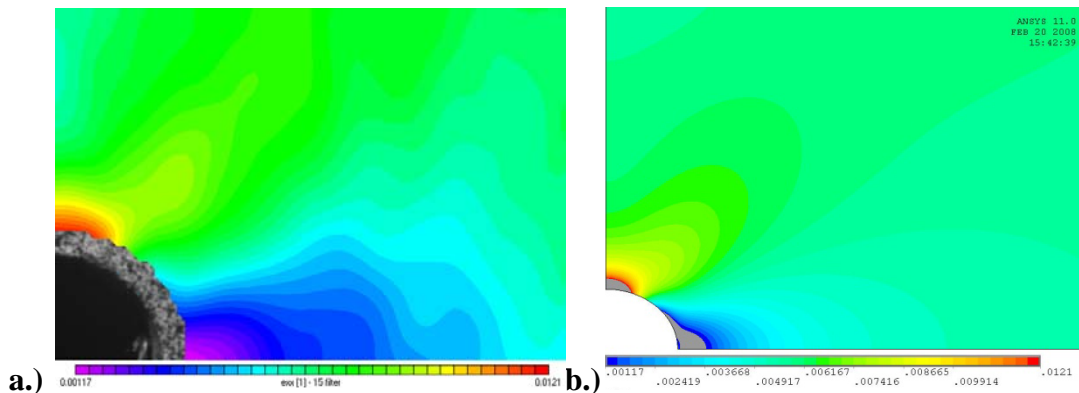
vary the strain concentration from ideal theoretical predictions. This effect would not be seen by the DICM strain contours since Moiré Interferometry goes nearly all the way to the hole edge. A standard machine shop drill press was used during fabrication, similar to the process that would be used in industry, and so Fig. 5.2.2 represents a more realistic engineering situation. Further contour comparisons for Layups BL and B for DICM and Moiré Interferometry under longitudinal and transverse deformation are given in Appendix G.



**Figure 5.2.2:** Moiré Interferometry longitudinal deformation contour for the baseline layup at 19 Ksi (26% UTS), where tensile loading is horizontal

The DICM strain contours of layup A were compared to those predicted by FEA in Fig 5.2.3. The tensile loading was horizontal, so that the maximum strain occurs at the top of the hole and the minimum strain at the side of the hole. The contour shapes for DICM and FEA compare well, with zones of maximum and minimum strain extending out from the hole, as in Fig. 5.1.1. As discussed in Chapter 3, the DICM strain field has an offset from the edge of the hole (Fig. 5.2.3a). The DICM maximum strain will be less

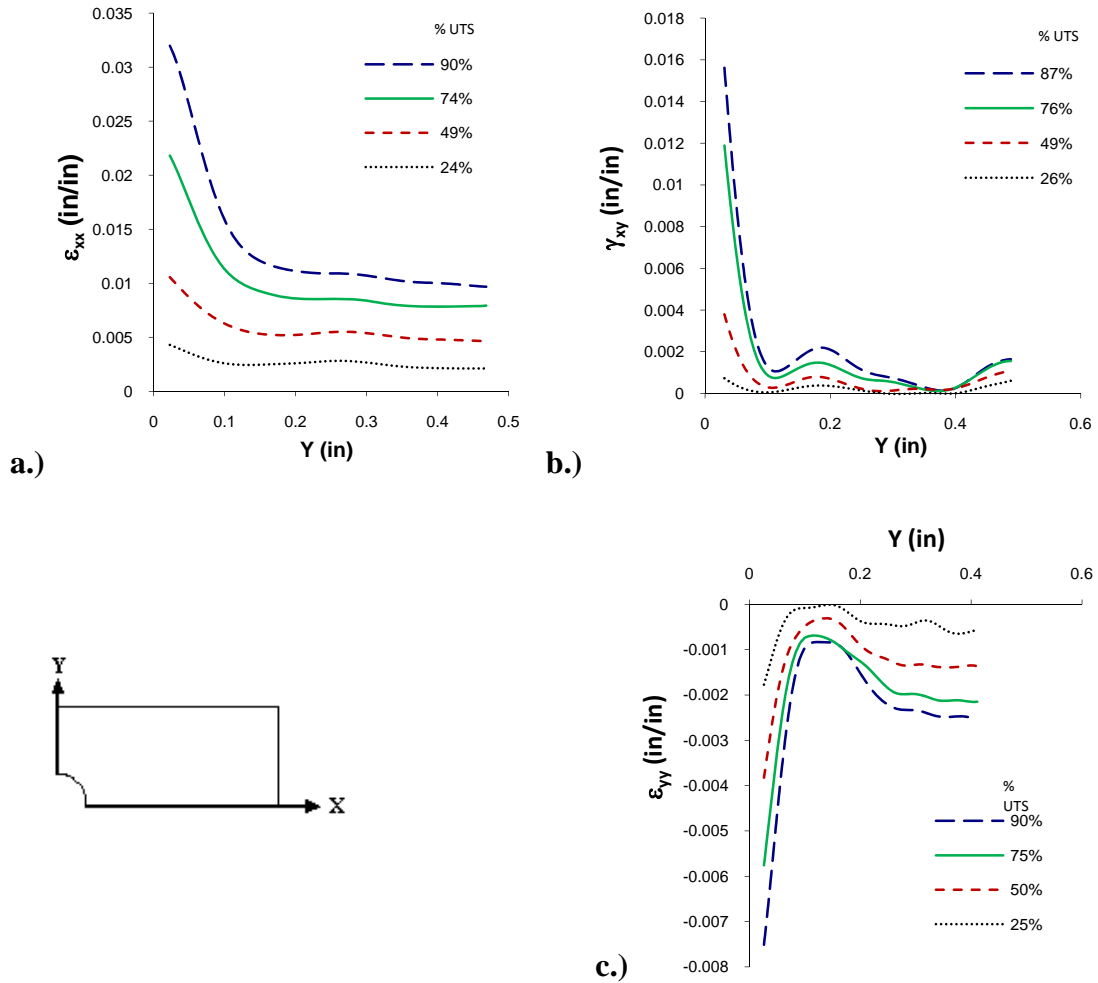
than that predicted by FEA because the DICM strain field does not go to the edge of the hole. The plotted strain range from the DICM contours was used for the FEA contours (Fig. 5.2.3.b). The gray areas near the top and side of the hole in Fig. 5.2.3.b represent strains above or below this range. The FEA gray area peaks should correspond to the DICM maximum and minimum strain locations. This is approximately true for the maximum strain, with FEA peak and DICM offset locations of 0.022 and 0.024 inches from the hole. However, the locations do not compare as well at the minimum strain with FEA peak and DICM offset locations of 0.056 and 0.026 inches from the hole. This discrepancy reveals that DICM correlates well with linear FEA for higher strains but less well for very low strains, as would be seen at the minimum strain zone of Location B (Fig. 2.4.1).



**Figure 5.2.3:** Strain contours for layup A at 38 Ksi (53% UTS) in a strain range of 0.00117 in/in to 0.0121 in/in for **a.)** DICM and **b.)** 2D linear FEA. The gray band for **b.)** near the top and side of the hole represents strains above and below the strain range.

Fig. 5.2.4 shows the development of longitudinal, shear, and transverse strain profiles along the net-section of selected layups, at loads of (or near) 25%, 50%, 75%, and 90% UTS. As expected, the strain is maximum at the hole due to the strain concentration induced by the absence of material. The transverse strain is negative due to the Poisson effect. These graphs represent the general trend for the strain profiles of the

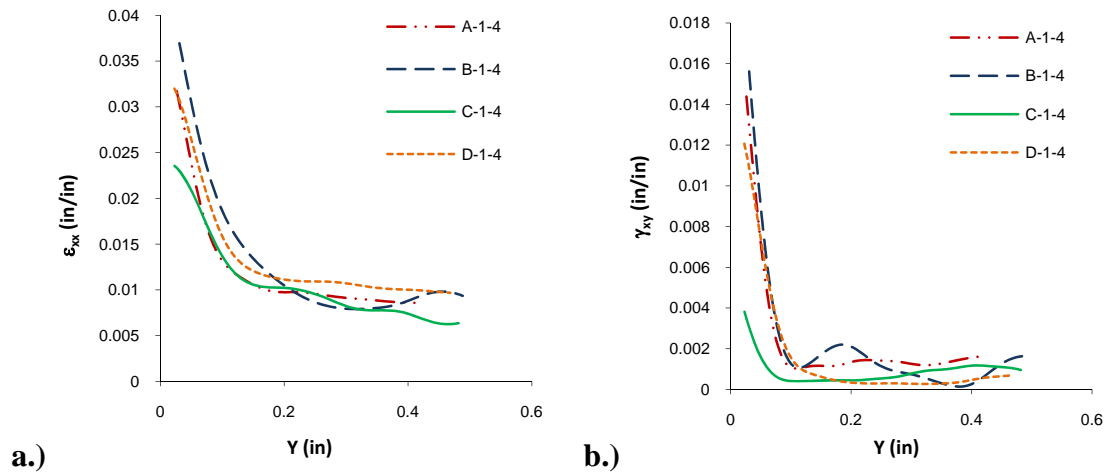
non-traditional layups tested, and the corresponding strain ranges. The trends are also illustrated by the contour plots of App. F., F.1-F.6.



**Figure 5.2.4:** Strain vs. Y distance from the hole using DICM for **a.)** longitudinal strain: layup D, **b.)** shear strain: layup B, and **c.)** transverse strain: layup A.

The longitudinal and shear strain profiles are compared in Fig. 5.2.5 for selected layups. The general trend remains consistent for all the layups, with the maximum at the hole and a decline in strain for the rest of the net section. Variations can be seen in magnitude of the maximum strain and the existence of “peaks” and “valleys” along the curve. The variation in the maximum strain is due to the specific layup’s strain response to the applied load (90% UTS in Fig. 5.2.5), and the peaks and valleys are due to a

combination of factors including the layup, load, speckle pattern, and coupon irregularities introduced during fabrication. From the magnitude of the maximum strains near the hole in Figs. 5.2.4 and 5.2.5, it seems reasonable that failure will be primarily dependent on the longitudinal strain and shear strain. Indeed, this is supported by the Maximum Strain Criterion, in which first and second ply failure modes are always either due to the longitudinal strain or the shear strain, which will be discussed further in Section 5.3 and 5.4.

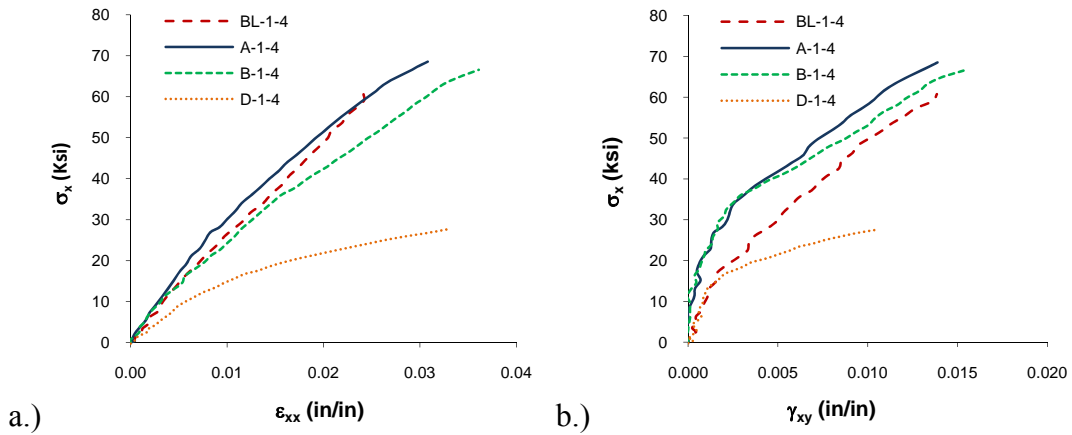


**Figure 5.2.5:** Strain vs. Y distance from the hole plots of selected coupons at 90% UTS for a.) longitudinal strain and b.) shear strain.

The stress-strain curves are plotted at Location A (Fig. 2.4.1) for selected layups in Fig. 5.2.6. The curves are clearly non-linear, indicating damage at the hole. “Knee” points represent ply failure and can be seen throughout the curves, which will be discussed in Section 5.3 and 5.4. The position of the curves relative to one another may change for longitudinal or shear stiffness, as shown by the layup A and B coupons. As damage progresses, some curves may soften more than others, as seen by comparing the A and B coupons, as well as the BL and D coupons. All the non-traditional laminates



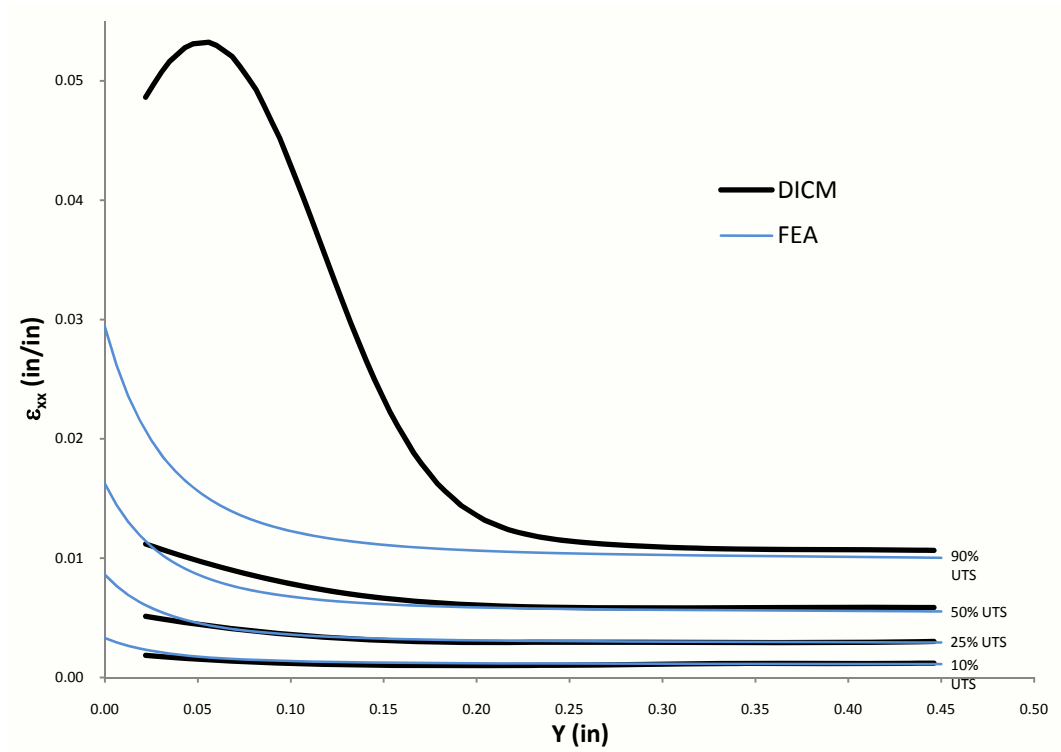
showed decreasing stiffness as damage progressed. Figs. 5.2.4-5.2.6 demonstrate the comparisons that can be generated using DICM.



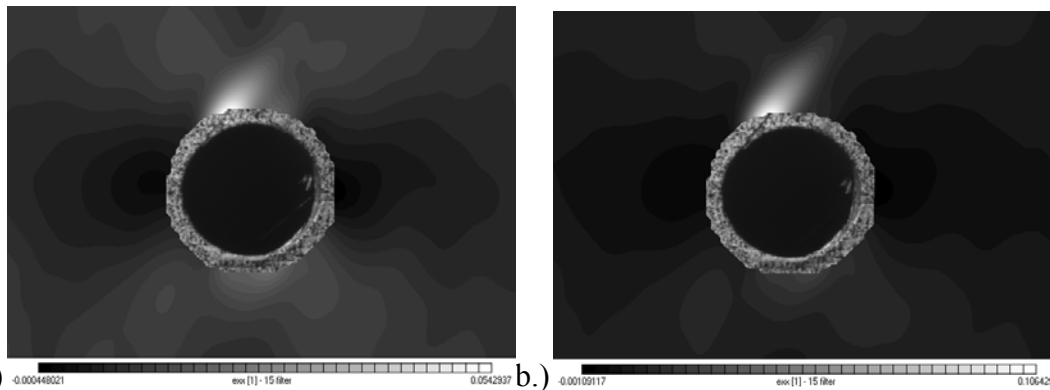
**Figure 5.2.6:** Stress vs. strain plots of selected coupons for a.) longitudinal strain and b.) shear strain.

To quantitatively compare DICM to the 2D FEA model, the axial strain was plotted as a function of the distance from the hole (normal to the loading direction) as shown in Fig. 5.2.7. DICM is not valid at the edge of the specimen (Fig. 5.2.1), so the DICM data in Fig. 5.2.7 begins at 0.022 in from the hole. For mid-range loading (10 to 50% UTS) the DICM solution compares reasonably well to linear FEA. The coupon used in Fig. 5.2.7 had a good speckle pattern (isotropic, high contrast, Fig. 3.1.5c), with camera images set closely to each other to improve calibration and correlation. For coupons with the speckle patterns of Fig. 3.1.5a-b (about two-thirds of the coupons tested), correlation was found to be reasonably good for 25 to 50% UTS. For all the coupons at high loads (90% UTS), non-linear effects become large, and the comparison deteriorates because the 2D FEA is linear. At low loads (<25% UTS), the comparison again deteriorates because DICM correlates poorly for low strains, as shown in Fig. 5.2.3. The DICM strain profile begins to decline near the hole for 90% UTS. This is due to the shifting of the strain concentration to the left under high damage, as shown in Fig. 5.2.8. This shift can be seen

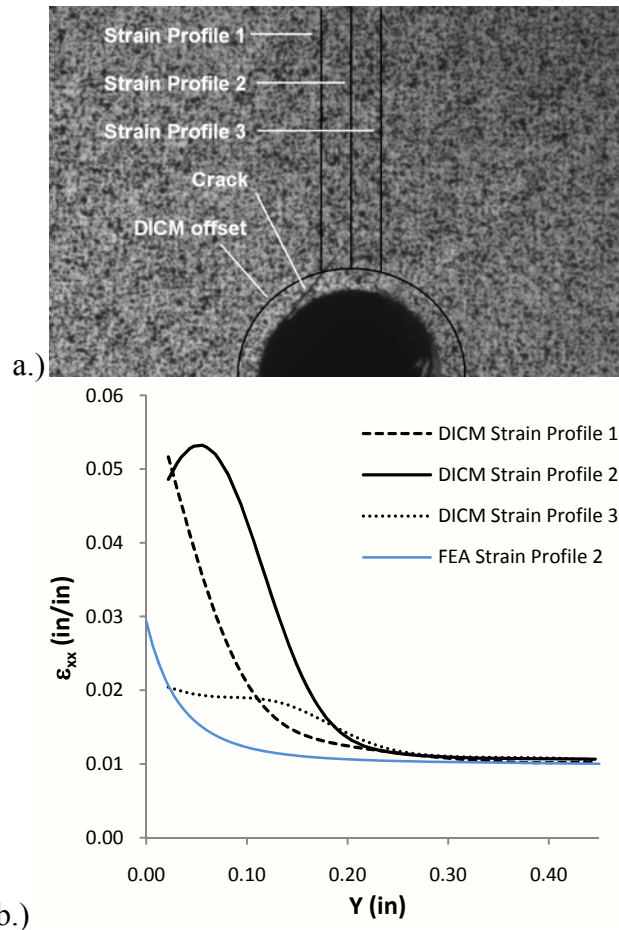
to occur due to the ply splitting along the 45° angle during damage progression, as shown by Fig. 5.2.9. The highest strain occurred at the crack, shown by Strain Profile 1 of Fig. 5.2.9. As the strain profile locations were moved away from the crack and away from Location A, the strain profiles decreased as shown by Strain Profiles 2 and 3.



**Figure 5.2.7:** Longitudinal strain of layup A as a function of distance from the hole for DICM and 2D FEA

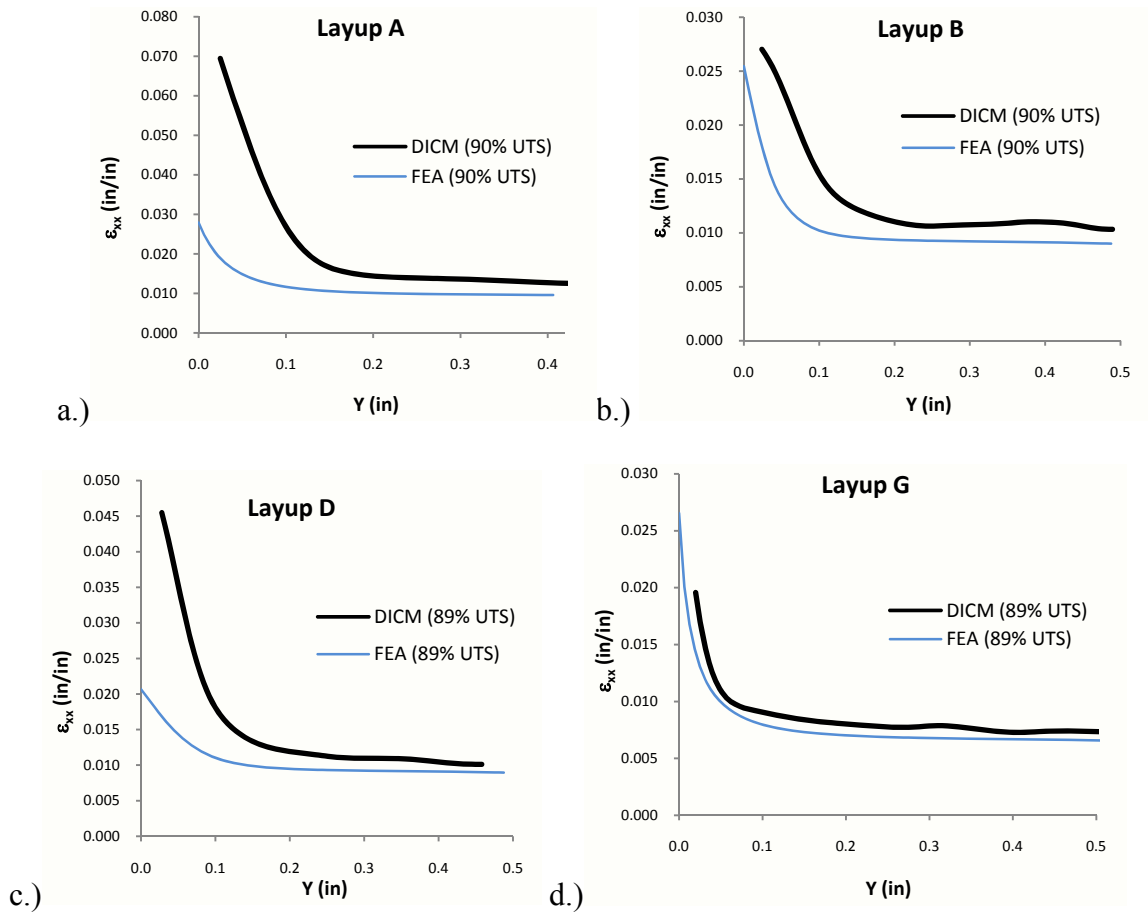


**Figure 5.2.8:** Longitudinal strain contours for layup A at a.) 90% UTS and b.) 98% UTS showing the maximum strain at the left of the net-section during high damage



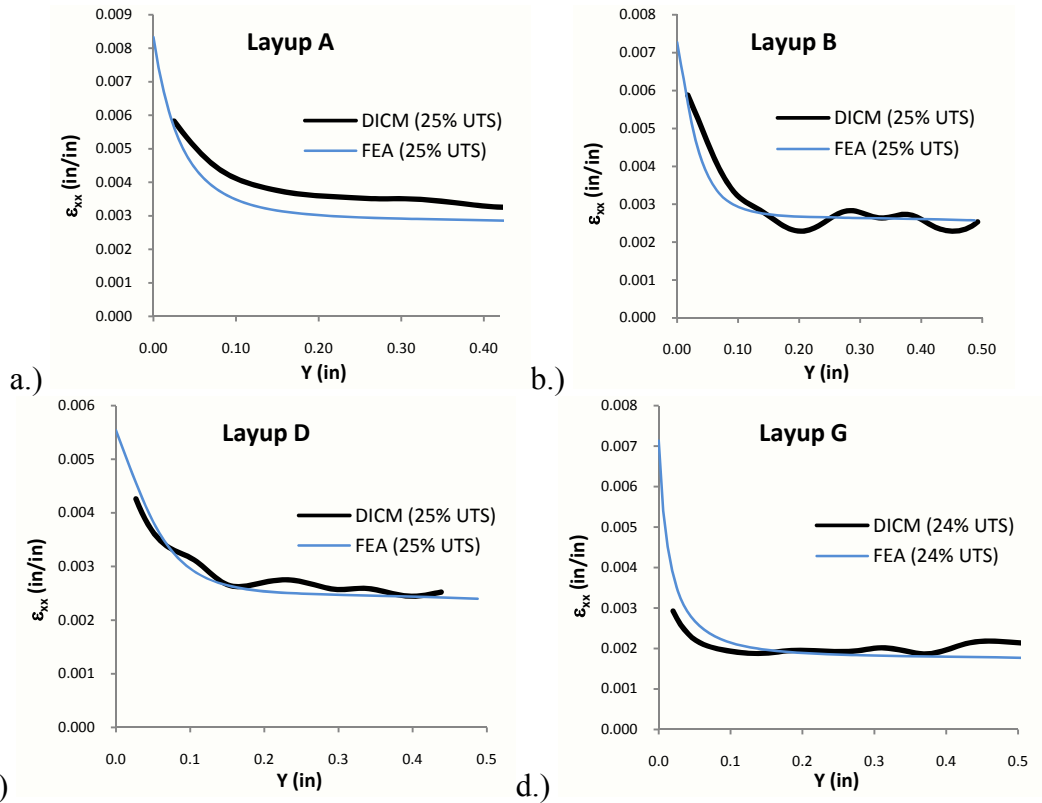
**Figure 5.2.9: a.)** Diagram showing the crack and the location of the strain profiles, and **b.)** strain profiles for Layup A at 90% UTS

The shift of the strain concentration in Fig.'s 5.2.7 -5.2.9 was unique to the damage of this coupon, as shown Fig. 5.2.10a in which the strain profile for another Layup A coupon does not decline near the hole. This illustrates the variability that can occur during damage progression due to coupon irregularities introduced during fabrication. Fig. 5.2.10b-d shows the strain profile along the net section of several other layups at 90% UTS, compared to linear FEA. This shows that the DICM curve is consistently higher than the linear FEA curve. This is expected, since the modulus should decrease during softening causing the strain to increase for the same applied stress.

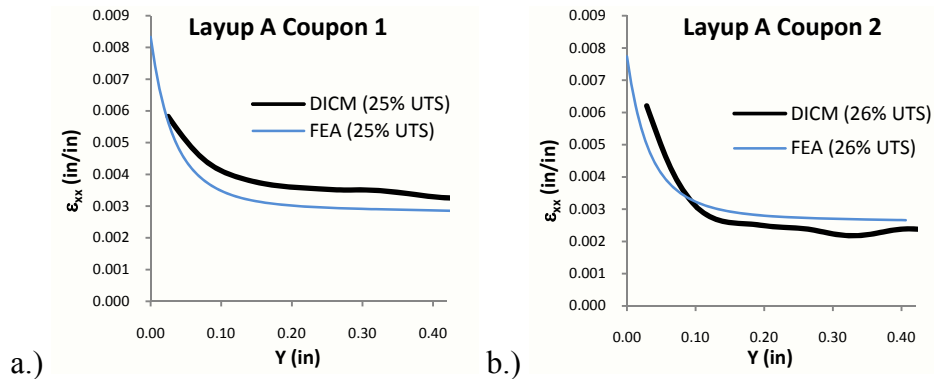


**Figure 5.2.10:** Longitudinal strain vs. distance from the hole for DICM and 2D FEA at 90% UTS for a.) Layup A, b.) Layup B, c.) Layup C, and d.) Layup D

For lower loads, less damage resulted in less nonlinearity, and the DICM strain profiles were closer to the linear FEA solutions (Fig. 5.2.11). However, due to the lower strains, the solutions are less smooth for 25% UTS. Furthermore, experimental variation was seen between coupons of the same layup, shown in Fig. 5.2.12. The general trend of Fig.'s 5.2.10-12 shows that when damage exists the DICM solution is higher than the FEA solution due to softening of the stiffness, and that variability exists due to experimental variation and low strains.

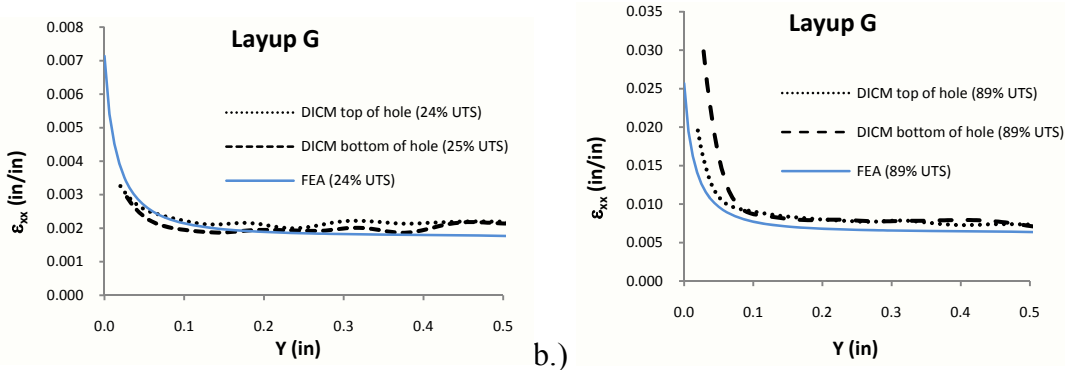


**Figure 5.2.11:** Longitudinal strain vs. distance from the hole for DICM and 2D FEA at 25% UTS for a.) Layup A, b.) Layup B, c.) Layup C, and d.) Layup D



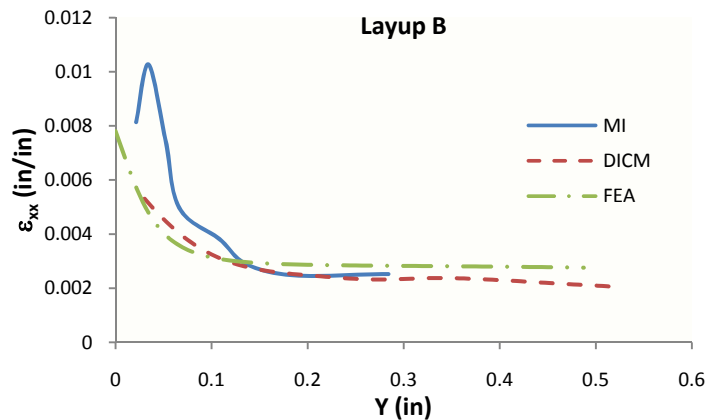
**Figure 5.2.12:** Strain profile for Layup A showing experimental variation at 25% UTS for a.) coupon 1, and b.) coupon 2

Some experimental variation can also be seen in Fig. 5.2.13, where the strain profile is shown at both sides of the hole for high and low loads. Near the hole at high loads (Fig. 5.2.13a), the strain profiles are somewhat different due to the individual damage occurring along either side of the hole. At lower loads (Fig. 5.2.13b), the solutions are much closer.



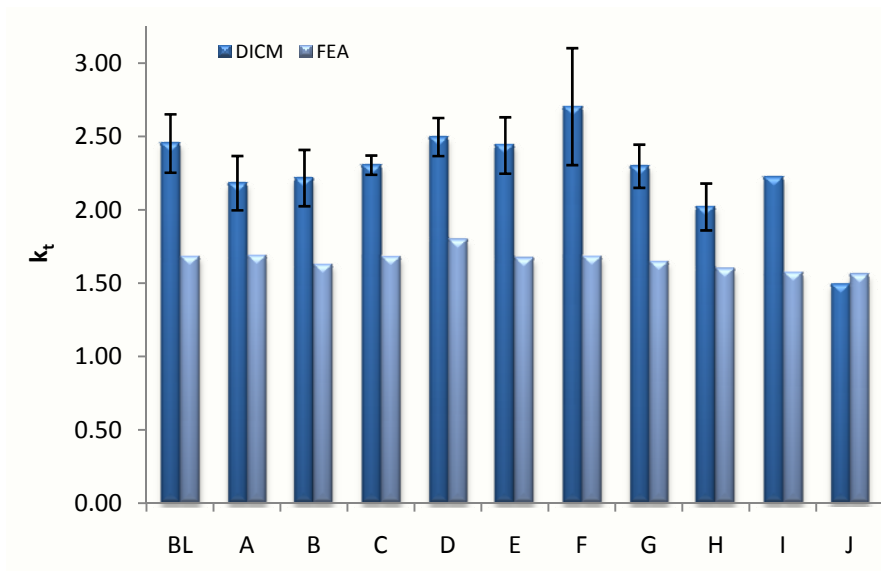
**Figure 5.2.13:** Strain Profiles at the top and bottom of the hole along the net section for Layup G at **a.)** 24% UTS, and **b.)** 89% UTS

The trends of Fig.'s 5.2.10-12 were also supported by Moiré Interferometry (MI), as shown in Fig. 5.2.14, where the strain profile along the net section was compared for MI, DICM, and FEA. Near the hole, the DICM and MI solutions rise above the FEA solution due to damage and stiffness softening. The MI solution begins to decline near the hole due to damage probably caused by drilling or shipping & handling (the Moiré analysis was performed at the University of Utah). The experimental results near the hole for DICM and MI may differ due to coupon irregularities and experimental variation. This difference in experimental results may also be due to limits in the strain measurement abilities of MI, as DICM shows a better approximation to linear FEA.

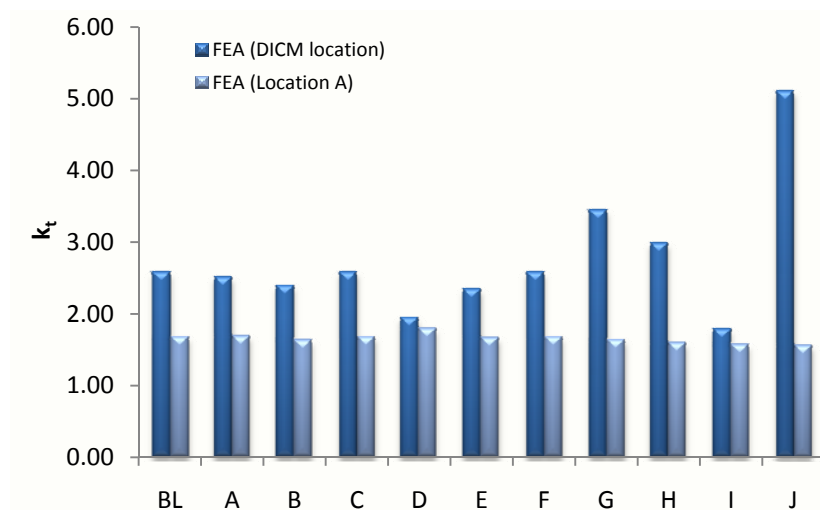


**Figure 5.2.14:** Longitudinal strain vs. distance from the hole at 19 Ksi (25% UTS) for Layup B using Moiré Interferometry, DICM, and FEA

The experimental and theoretical strain concentrations were compared in Fig. 5.2.15 using DICM and linear 2D FEA, respectively. The experimental  $k_t$  was found as the slope of the  $\epsilon_{\max}$  vs.  $\epsilon_{\text{nom}}$  DICM plot in the linear region, with a correction factor applied as explained in Section 2.2. To obtain an accurate comparison, the  $k_t$ 's from the FEA were compared to DICM at the DICM offset location, instead of Location A from Fig. 2.4.1. A representative offset of 0.022 inches was used, due to the fact that the same or nearly the same subset was used for the over 50 coupons tested resulting in approximately the same offset for all the coupons. The DICM nominal strain was not a true nominal strain due to the limits of the measureable area of the DICM camera. The DICM nominal strain location was taken at (0.5 in, 0.375 in), as was the FEA nominal strain to obtain an accurate comparison. This location was taken in order to avoid the minimum strain zone stemming from Location B (Fig. 2.4.1), and also to avoid free-edge effects, and thus be as close to the true experimental nominal strain as possible. The DICM- and FEA-predicted  $k_t$ 's did not compare well, due to the experimental variation near the hole shown by Fig.'s 5.2.10-5.2.13. Fig. 5.2.16 shows the difference between the adjusted DICM location  $k_t$ , used in Fig. 5.2.15, and the original Location A based  $k_t$ , used in Chapter 2. The large variation in  $k_t$  between these locations is due to the large strain gradient that exists near the hole. This strain gradient, along with the experimental variation of Figs. 5.2.10-5.2.12, explains the variation in experimental and theoretical results of Fig. 5.2.15. This variation suggests that DICM can only be used as an indication of the strain concentration near the hole.



**Figure 5.2.15:** Comparison of experimental (DICM) and theoretical (2D linear FEA) results for the strain concentration,  $k_t$ , at the DICM location ( $\epsilon_{\max}$  at 0.022 inches from the hole and  $\epsilon_{\min}$  at (0.5, 0.375) inches from the coupon center)

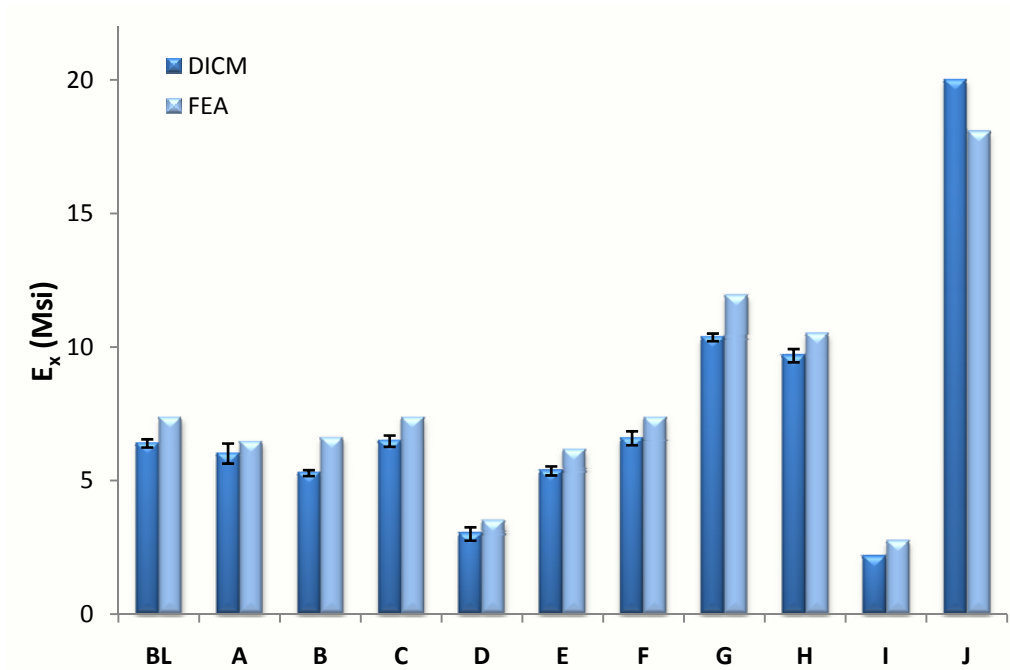


**Figure 5.2.16:** Comparison of the  $k_t$  from the DICM location ( $\epsilon_{\max}$  at 0.022 inches from the hole and  $\epsilon_{\min}$  at (0.5, 0.375) inches from the coupon center) to the  $k_t$  from Location A ( $\epsilon_{\max}$  at Location A, Fig. 2.4.1, and  $\epsilon_{\min}$  at (4, 0) from the coupon center)

Fig. 5.2.17 compares the modulus calculated using DICM and FEA at the (0.5, 0.375) nominal location. The modulus was calculated using the nominal strain and the applied stress. As mentioned for Fig. 5.2.15, the nominal location was taken to avoid maximum strain zones and be as close to the true nominal as the DICM strain field would allow.



Fig. 5.2.17 shows that the theoretical modulus was near but consistently higher than the DICM modulus. This may be due to through-the-thickness effects caused by the interlaminar interaction between plies of varying angles, not accounted for in 2D FEA. Fig. 5.2.17 shows that a reasonable experimental and theoretical comparison exists for the modulus in the linear region.

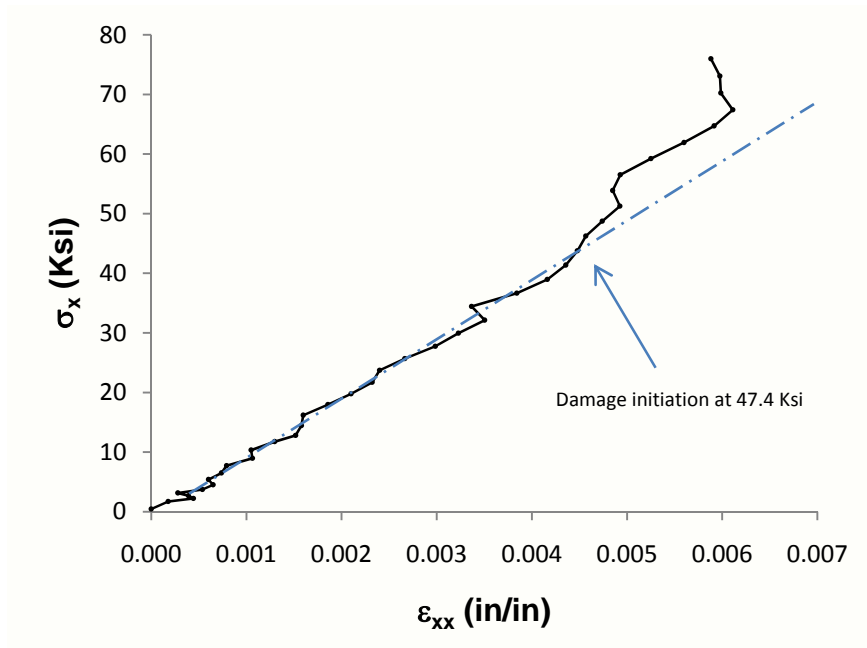


**Figure 5.2.17:** Comparison of experimental (DICM) and theoretical (FEA) results for the longitudinal modulus,  $E_x$

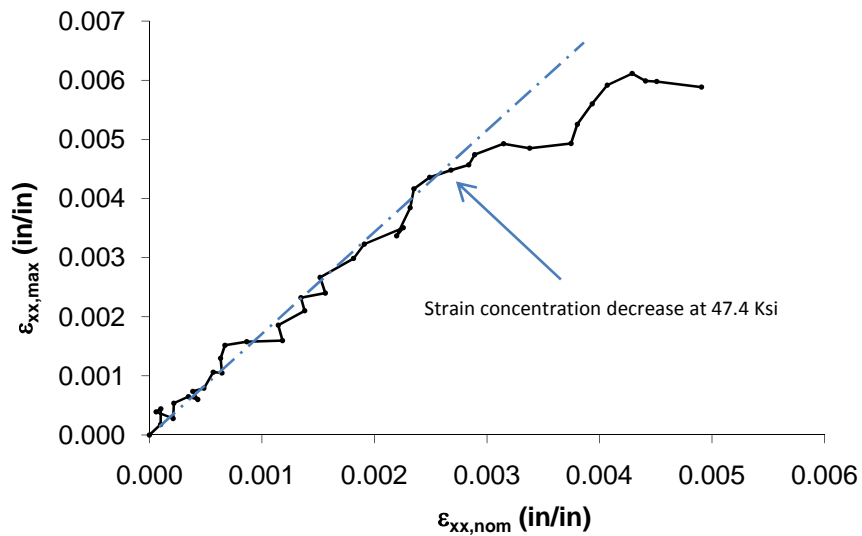
### 5.3: Failure Modes

This section discusses Layups I and J, which were  $[(\pm 45)_4]_s$  and  $[0]_{16}$ , respectively (Table 5.1.1). These layups were fabricated to study the corresponding shear and longitudinal failure modes, in order to better understand failure in the non-traditional laminates. In Fig. 5.3.1, the uniaxial applied stress-longitudinal strain curve slopes upward after the proportional limit, indicating an increase in stiffness after damage initiation. This increase in stiffness is unique to the uniaxial layup, as the stiffness decreases near the hole during damage progression for common hard layups (those with high numbers of  $0^\circ$  plies), noted by References [2] and [22]. The decrease in stiffness noted in the literature was supported in the current study by the stress-strain curve of the hard laminate H, shown at the end of Appendix H.1. The stiffness increases for Layup J is due to the physical nature of the uniaxial layup, where the stress concentration is redistributed due to longitudinal matrix splitting. This stress redistribution is shown in Fig. 5.3.2, where the slope equals the strain concentration (without the correction factor of Section 2.2 applied, since the variable of interest is only the change in slope). The slope of Fig 5.3.2 decreases at the load where the stiffness of Fig. 5.3.1 increases (47.4 ksi), indicating that stress redistribution has occurred. This stress redistribution causes an increase in stiffness in the uniaxial layup due to the stress concentration being effectively “cut-off”, and the coupon being divided into three sections, as shown in Fig. 5.3.3d. This division is suppressed in most hard laminates due to secondary load bearing plies (such as the  $\pm 45^\circ$  plies of Layup H), which cause the stiffness to decrease after stress redistribution. The decrease in stiffness after stress redistribution is much less in hard

laminates (Layup H), than in less hard laminates (Layup D), as would be expected. This trend can be seen in App. H and will be discussed in Section 5.4.



**Figure 5.3.1:** Stress vs. longitudinal strain for a uniaxial coupon (Layup J) showing an increase in stiffness after damage initiation (at Location A, Fig. 2.4.1)



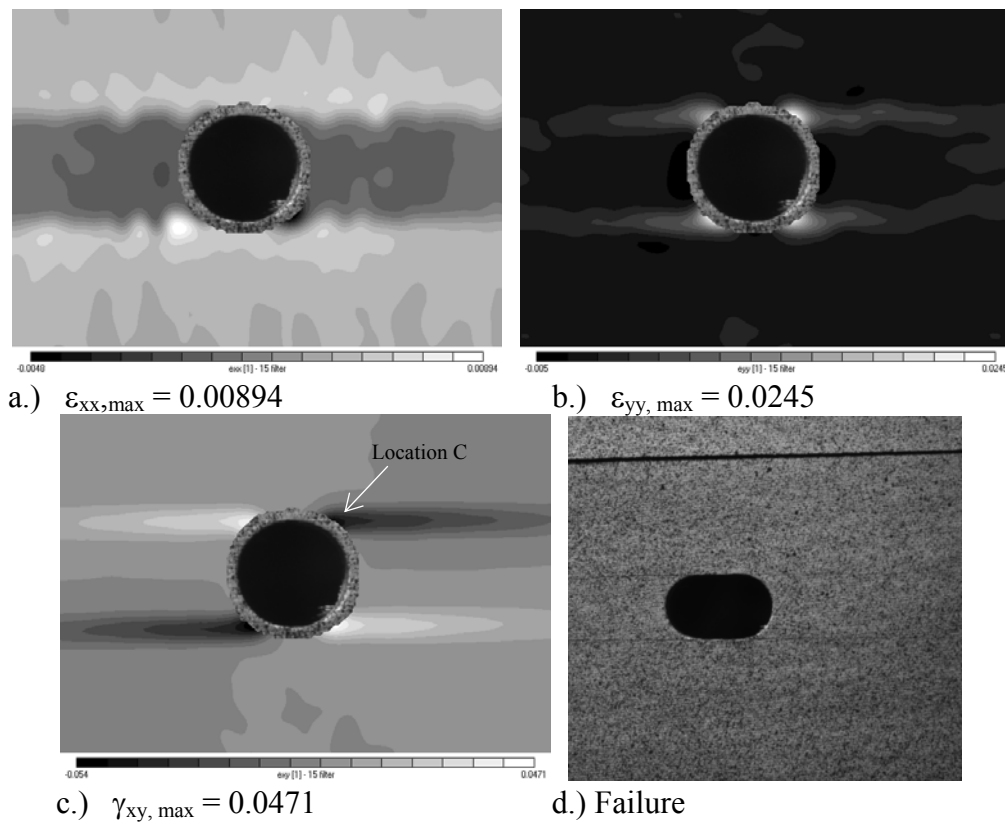
**Figure 5.3.2:** Maximum vs. nominal strain for layup J (uniaxial), indicating a stress redistribution where the slope (representing the strain concentration) decreases, corresponding to an increase in stiffness in Fig. 5.3.1 (at Location A, Fig 2.4.1)

FPF is located at the first “knee” point of the stress vs. strain curve (Fig. 5.1.3), and corresponds to the proportional limit of this curve [26]. This knee was found for the traditional  $[0/\pm 45/90]_s$  laminate in Ref. [53], and occurred in the  $90^\circ$  plies. The  $90^\circ$  ply is expected to fail first because the matrix resists the applied stress and not the fibers, causing FPF due to the transverse matrix failure mode. For open-hole tension, this failure mode is expected to occur at Location A (Fig. 2.4.1) where the maximum longitudinal strain occurs. However, the  $90^\circ$  plies were altered for some of the non-traditional laminates (Table 5.1.1), and no  $90^\circ$  plies existed in the failure mode laminates (Layups I and J), so FPF could theoretically occur anywhere along the circumference of the hole, and due to any failure mode (shear, transverse, or longitudinal) for these layups. For the non-traditional layups, the question of where FPF occurs will be resolved in Section 5.4. For the failure mode layups, this question is resolved in the following discussion.

Wherever FPF occurs, it should correspond to a slope change in the experimental stress-strain curve at this location because the resulting damage should soften (reduce) the stiffness. Depending on the failure-mode (longitudinal, transverse, or shear), this slope change may be best observed using the stress-longitudinal strain, stress-transverse strain, or stress-shear strain curve. FPF cannot be observed experimentally except under CT scanning, which was not available for the current study, but will be examined by Kothidar [42]. Thus, the method in the current study for locating FPF was to locate the change in stiffness of the stress-strain curve (the “knee” point of Fig. 5.1.3), and compare this to linear FEA and failure criteria.

The Maximum Strain Criterion predicts a longitudinal failure mode for the uniaxial Layup J at Location A, which would indicate fiber breakage and thus catastrophic failure.

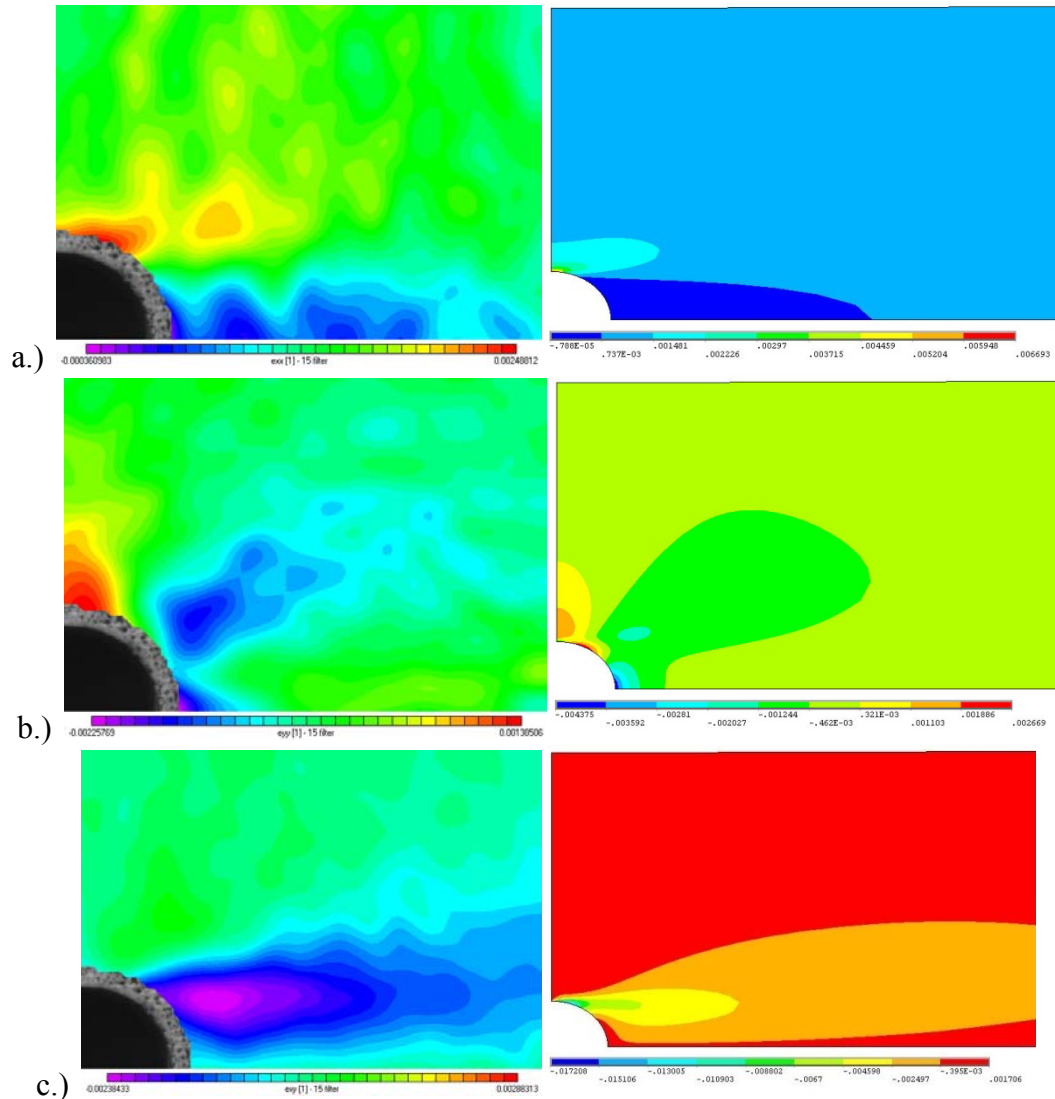
This failure mode would suggest that either FPF equals UTS for Layup J, or that FPF is occurring somewhere else in the coupon. For Layup J, Fig. 5.3.3 shows that high transverse and shear strains develop at the hole edge at approximately  $\pm 60^\circ$  from the longitudinal direction, defined now as Location C (Fig. 5.3.3.c). Since the longitudinal failure index at Location A was found to be small (App. A Table A.5), and since the transverse strains at Location C are much higher than usual (Fig. 5.3.3), it is reasonable to suggest that failure may initiate at Location C instead of Location A for Layup J.



**Figure 5.3.3:** Strain contours at 90% UTS for Layup J showing a.) longitudinal strain, b.) transverse strain and, c.) shear strain. Image “d” shows the uniaxial coupon failed due to matrix splitting at the sides of the hole.

The experimental DICM strain contours qualitatively compare well with the theoretical linear FEA strain contours for Layup J at 25% UTS (Fig. 5.3.4) at Location A

and Location C. Thus, the FPF from the experimental stress-strain curve (the “knee” point of Fig. 5.1.3), should compare well to linear FEA and failure criteria.



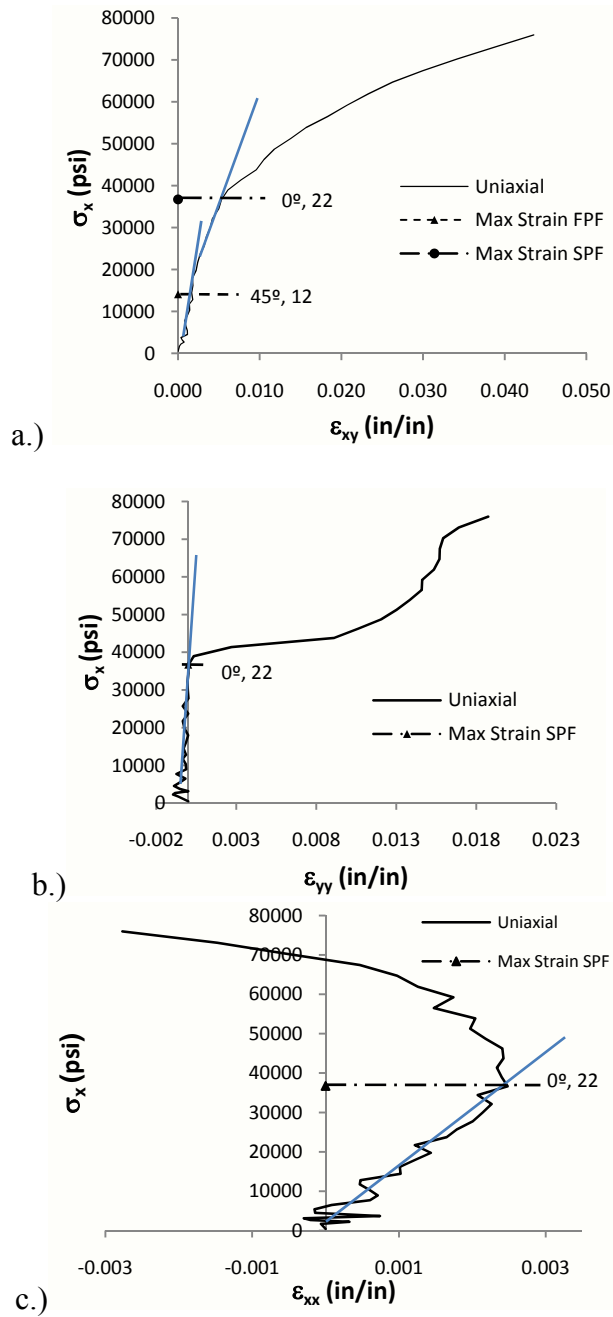
**Figure 5.3.4:** DICM (left image) and FEA (right image) strain contours at 25% UTS for a.) longitudinal strain, b.) transverse strain, and c.) shear strain

Using Location C and comparing experiment to theory, good correlation was found for FPF (Fig. 5.3.5a). The Maximum Strain prediction for FPF in Fig. 5.3.5a is shown by a horizontal line indicating the stress, and text indicating the failure ply ( $45^\circ$ ) and failure mode (shear, 12). Using the second highest failure mode (transverse) for second ply failure (SPF)<sup>9</sup>, and comparing this to the second proportional limit in the shear, transverse, and longitudinal stress-strain curves (Fig. 5.3.5a-c), a strong correlation was also found. These results indicate that if the stress-strain curve is close to linear, then linear FEA and the Maximum Strain Criteria can predict FPF and SPF. However, SPF was not pursued in this paper due to its dependence on nonlinear damage. Furthermore, as shown by Refs. [14] and [15], after damage initiation strain is no longer constant through-the-thickness, which would affect the surface strain measurements of DICM. This affect would need to be quantified using a nonlinear 3D FEA model before SPF could be understood, and is left for future research.

From these comparisons it is suggested that FPF in uniaxial coupons with center holes occurs due to the shear failure-mode at Location C, and that final failure is primarily due to matrix splitting at the sides of the hole (Fig. 5.3.3d). The longitudinal (fiber failure) mode was predicted to occur last, but the catastrophic failure that normally accompanies this failure mode was not seen. Thus, shear and transverse failure modes may have caused the matrix splitting that lead to final failure. However, some fiber breakage may still exist inside the laminate.

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<sup>9</sup> Second ply failure for a uniaxial laminate is a misnomer here, since it is actually the second failure mode of the same  $0^\circ$  ply. However, in the non-traditional laminates SPF always occurred in a different ply than FPF, so the nomenclature is maintained here.

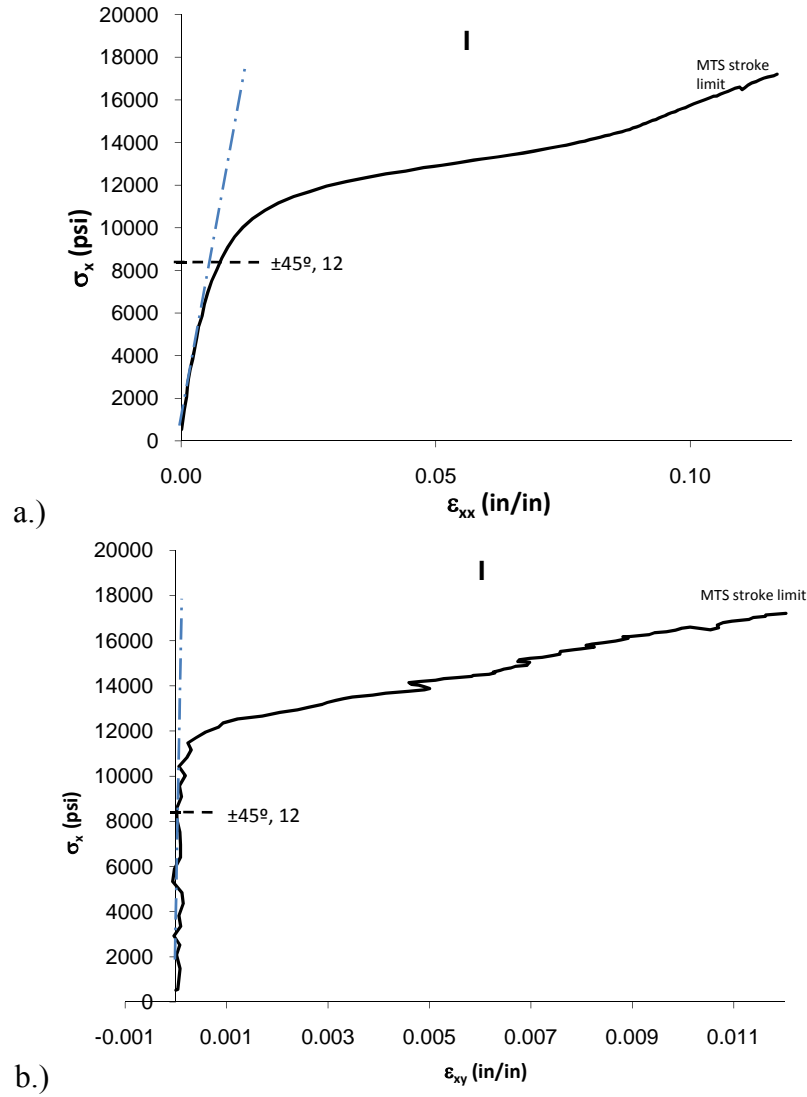


**Figure 5.3.5:** Stress-strain plots for a.) shear, b.) transverse, and c.) axial strain at Location B of the uniaxial “J” laminate showing FPF and SPF

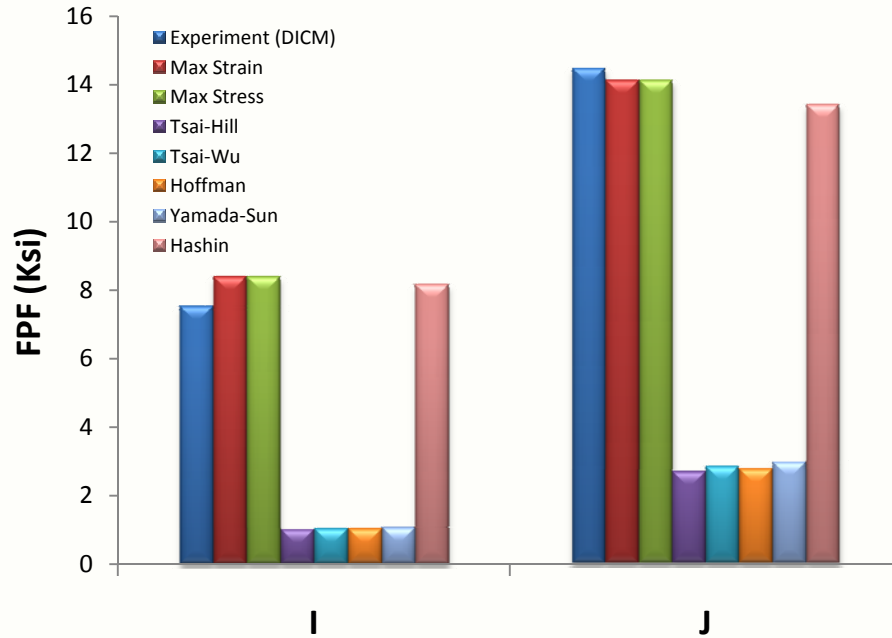


There is some debate as to how shear strength should be defined. Shear strength may be dependent on thickness, and a scale factor has been suggested for thick laminates [54]. However, more recently, Wang [4] suggested that shear strength in the  $\pm 45^\circ$  tension test does not strongly depend on thickness. ASTM D3518 defines shear strength as the stress corresponding to  $50,000 \mu\epsilon$ . The ASTM standard for shear strength was used in the present study to predict FPF, and a scale factor was not applied. The longitudinal stress-strain plot of the  $\pm 45^\circ$  I layup is shown in Fig. 5.3.6a, which shows reasonable correlation between the proportional limit and the FPF predicted by the Maximum Strain Criteria. Using the stress-shear strain plot, a better correlation is found (Fig. 5.3.6b). Thus, the ASTM standard was used in this study, as were stress-longitudinal strain and stress-shear strain plots to locate FPF, which will be discussed further in Section 5.4.

The proportional limit from DICM was compared to the FPF predicted by common failure criteria in Fig. 5.3.7. Good correlation was found between experiment and the Maximum Strain and Maximum Stress Criteria for the failure mode layups I and J. Hashin appeared to show good correlation, but only because it was reduced to the Maximum Stress criteria for the given strain states. The remaining quadratic interaction criteria did not approximate FPF well.



**Figure 5.3.6:** Stress-strain plots of the  $\pm 45^\circ$  failure-mode layup I for a.) longitudinal strain and b.) shear strain comparing the proportional limit to the FPF predicted by the Maximum Strain Criterion



**Figure 5.3.7:** Experimental FPF using DICM and the proportional compared to FPF predicted by common failure criteria.

As mentioned above, Location C was used for Layup J for the strain state input into failure criteria and for the location of the DICM stress-strain curves. For the remaining layups (BL through I, Table 5.1.1), Location A was used. A theoretical study was performed that examined FPF at the maximum strain locations for the shear, transverse, and longitudinal strains of each laminate. For layups BL-I, FPF always occurred at Location A, so this location was chosen for experimental FPF analysis.

#### 5.4: Experimental and Theoretical FPF for Non-Traditional Laminates

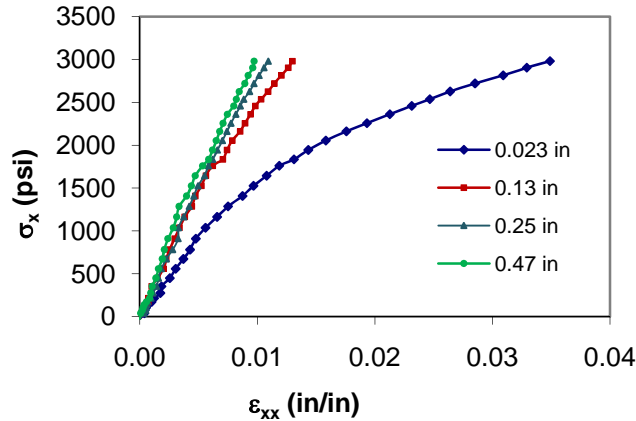
The experimental results generated by DICM were compared to predictions of FEA and failure criteria. According to Maximum Strain, first ply failures for the layups defined in Table 5.1.1 were generally transverse to the fiber, indicating matrix cracking or matrix splitting, but in some cases were in shear as shown by Table 5.4.1. In this table 11, 22, and 12 represent longitudinal, transverse, and shear failure modes. The failure

mode observations made here are theoretical (based on Maximum Strain), compared to experimentally observed changes in stiffness (equivalent to the “knees” of Fig. 5.1.3).

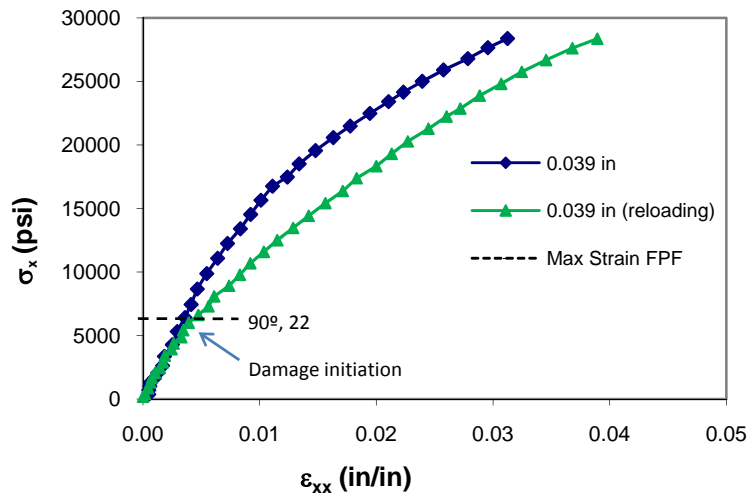
**Table 5.4.1:** Failure Modes for FPF as predicted by Maximum Strain Criterion

Max Strain	BL	A	B	C	D	E	F	G	H	I	J
Ply	90°	90°	45°	90°	90°	54°	90°	90°	45°	45°	45°
Direction	22	22	12	22	22	12	22	22	12	12	12

Stress-strain behavior for composites is often thought of as being linear until failure. This is true away from the hole, as would be measured by a strain gage or extensometer. However, near the hole the stiffness becomes nonlinear (Fig. 5.4.1), indicating the usefulness of DICM. The polymer behavior of the epoxy matrix could account for some of this nonlinearity, as polymers can show nonlinearity even in their elastic region. However, the reloading curve of Figure 5.4.2 shows that damage is in fact occurring. The “knee” can be seen in the loading and reloading curves of Fig. 5.4.2. This knee indicates experimental damage initiation, and corresponds to the theoretical FPF as predicted by maximum strain. It is important to note that the reloading curve of Fig. 5.4.2 does not follow the final slope of the original loading curve, as would be expected with ideal damage. This could be due to cracks being sealed upon unloading, or due to nonlinear effects in the coupon itself. Since the 0° plies of the baseline layup were replaced with 57° plies to make Layup D, this layup is essentially an in-plane shear coupon, and nonlinear effects would be expected in Fig. 5.4.2. The Maximum Strain FPF for Fig. 5.4.2 is shown by a horizontal line indicating the stress, and text indicating the failure ply (90°) and failure mode (transverse, 22).



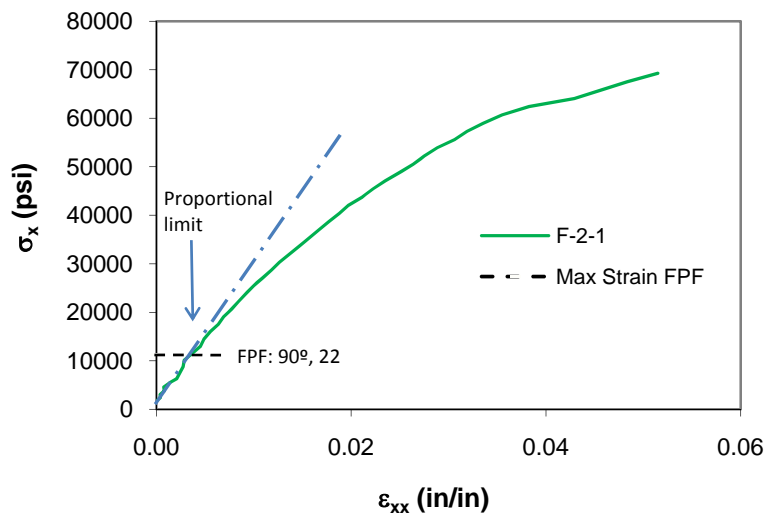
**Figure 5.4.1:** Stress vs. longitudinal strain for locations from the hole along Y (Fig. 2.4.1) for Layup D



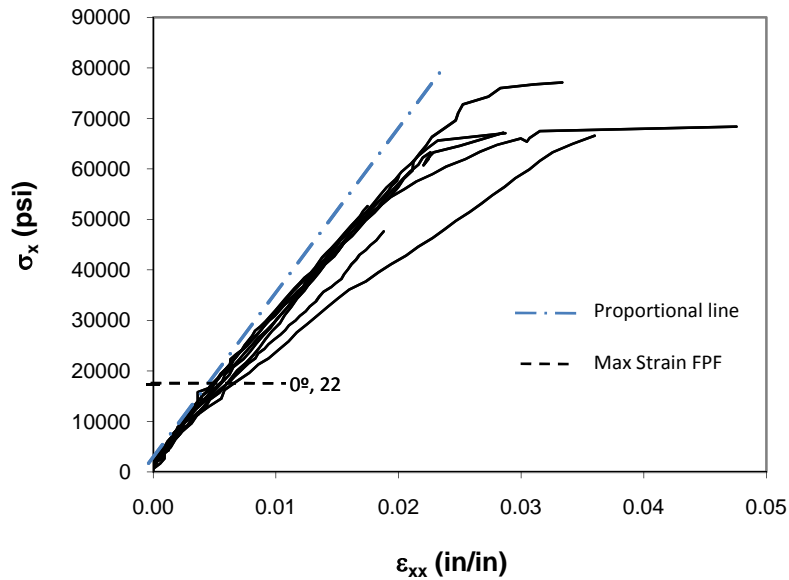
**Figure 5.4.2:** Stress vs. longitudinal strain for loading and reloading up to 90% UTS, compared to Maximum Strain predicted FPF (Layup D)

Fig. 5.4.3 shows that the proportional limit approximately corresponds to the FPF predicted by Maximum Strain for Layup F. The proportional limit is more subtle for some other layups, as only a slight change in stiffness follows FPF. However, FPF can still be seen clearly as a trend when comparing several coupons of the same layup, as seen for Layup B in Fig. 5.4.4. Plots similar to Fig. 5.4.4 for all the non-traditional layups are given in Appendix H. Appendix H shows the applied stress vs. longitudinal strain and the applied stress vs. shear strain curves of several coupons for each of the eight non-

traditional layups (as well as the baseline layup) defined in Table 5.1.1. In general, FPF can be clearly seen from these curves, as well as SPF and  $k^{\text{th}}$  ply failure in many instances. However to accurately understand ply failure after the proportional limit, 3D FEA and nonlinear damage progression analysis is needed, which is beyond the scope of the current study. Thus, the plots of Appendix H were used to locate FPF at the proportional limit. Scatter can be seen between these stress-strain curves as damage progresses (Fig. 5.4.4). This may be due to material defects and inconsistencies inherent to the fabrication process, as well as experimental variation caused by the speckle pattern and correlation. The curves agree in the linear region where there is little or no damage, as expected. The curves begin to diverge from the proportional line at the proportional limit, suggesting that this is a good definition of FPF.



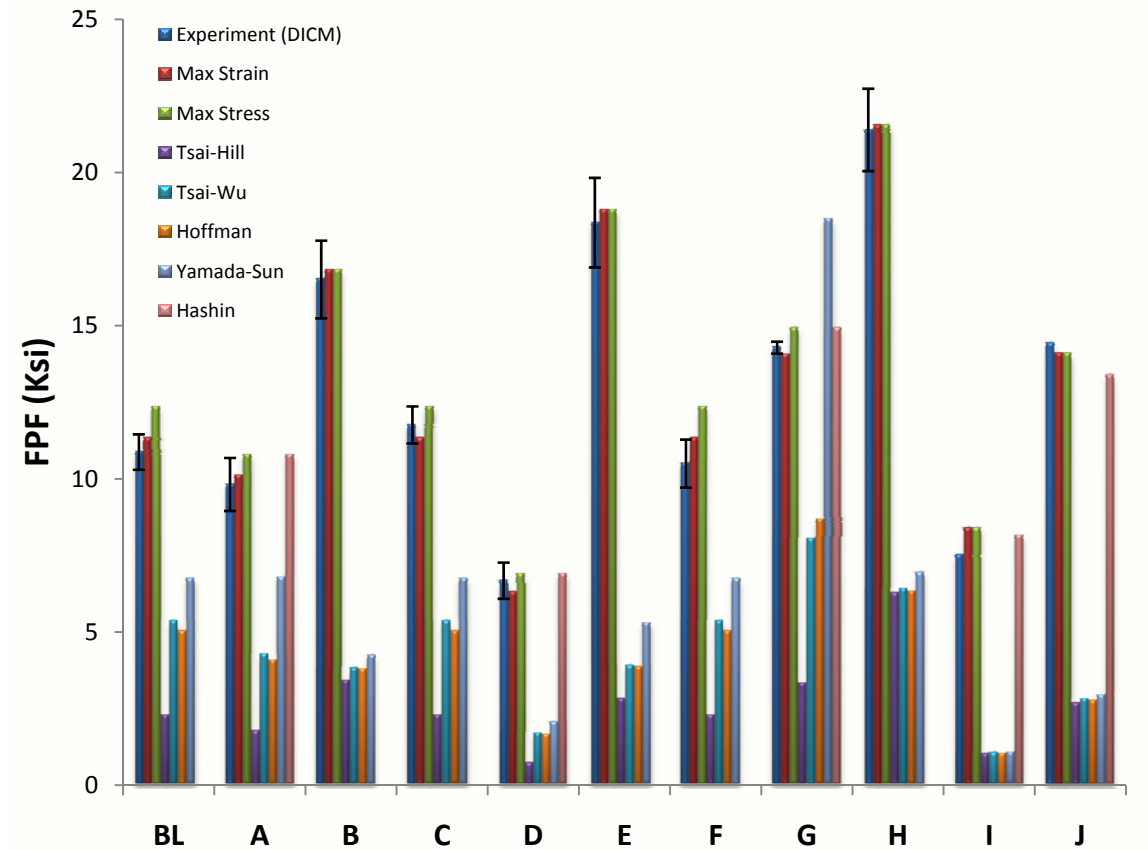
**Figure 5.4.3:** Stress vs. longitudinal strain showing the proportional limit compared to Maximum Strain predicted FPF (layup F)



**Figure 5.4.4:** Stress vs. longitudinal strain for layup B, comparing the proportional limit to Maximum Strain predicted FPF.

Fig. 5.4.5 compares the experimental FPF (from DICM using the proportional limit) to predictions of FPF from common failure criteria. Table 5.4.2 shows how many coupons of each layup were tested. This number varies due to the availability of material for coupon fabrication, as well as DICM difficulty with the speckle pattern in a few of the coupons tested. The error bars represent one standard deviation. The Maximum Strain Criterion was best at describing FPF, showing that it was an appropriate choice for optimization. Similarly, the Maximum Stress Criterion also gave consistently close approximations to the proportional limit. Hashin Criterion produced values of zero for all the layups except A, D, and G, where the strain state caused the Hashin Criterion to reduce to the Maximum Stress Criterion. The remaining quadratic interaction criteria all poorly predicted FPF. Fig. 5.4.5 and Table 5.4.1 indicate that the Maximum Strain and Maximum Stress Criteria work well for matrix splitting and shear failure modes. This

result is important because these criteria have generally only been applied to fiber dominated strength [26].



**Figure 5.4.5:** Experimental FPF (proportional limit, DICM) for non-traditional laminates, compared to common failure criteria.

**Table 5.4.2:** Number of DICM coupons experimentally tested for Fig. 5.4.5

Name	BL	A	B	C	D	E	F	G	H	I	J
#	8	8	8	5	9	5	3	2	3	1	1

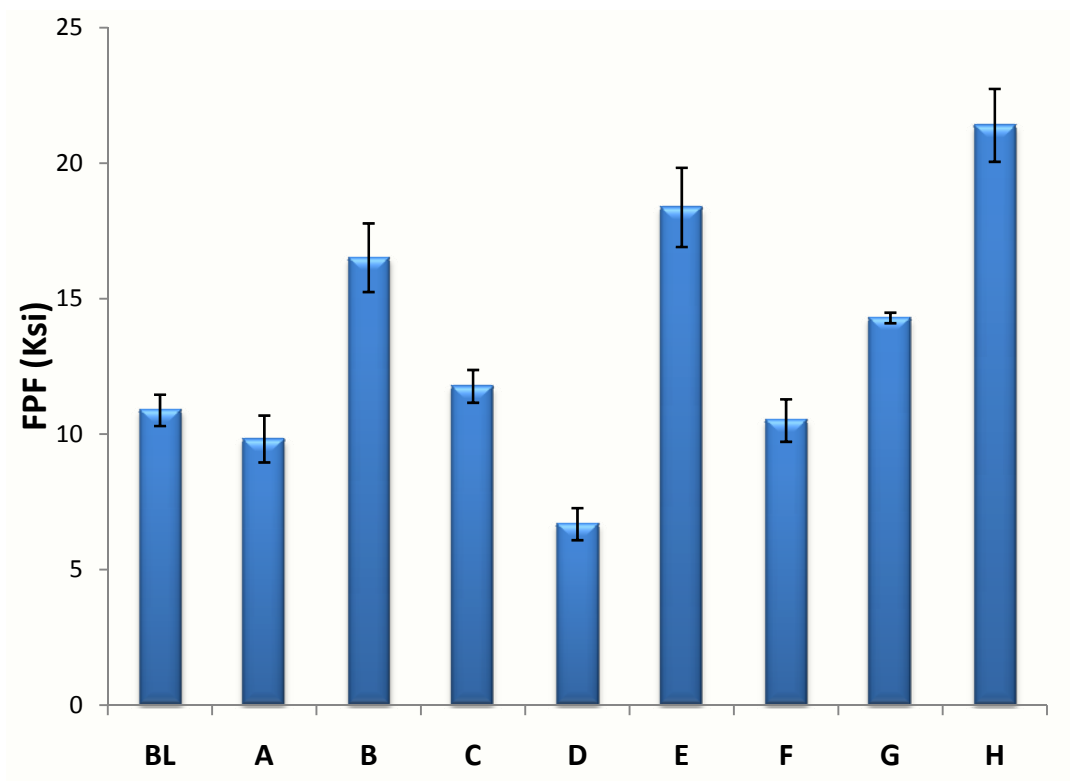
### 5.5: Strength

First ply failure (FPF) and ultimate tensile strength (UTS) were examined in the current research. For FPF, a strength increase was found for the non-traditional layups B, C, E, G, and H, as shown in Fig. 5.5.1. Table 5.5.1 shows the corresponding number of coupons tested for each layup. For layups G and H, this increase was expected, as these layups were optimized for FPF using the Maximum Strain Criterion. The largest increase



in strength was for layup H, which replaced the Baseline 90° plies with 0° plies during optimization. More surprising were the FPF increases found for layups B, C, and E, which were  $k_t$ -optimized. Referring to the FPF failure-modes discussed in Section 5.3 (Table 5.3.1), it is seen that the FPF failure-mode for layups B and E switched from the Baseline failure ply of 90° in the 22 direction, to a shear failure mode of the  $\pm 45^\circ$  ply (or  $\pm 54^\circ$  for layup E) in the 12 direction. This was also seen for the FPF-optimized layup H, whose failure ply was  $\pm 45^\circ$  in the 12 direction. In fact, it can be seen that layups B, E, and H, with the corresponding shear failure modes, have the highest increase in FPF over the Baseline. This is due to the 90° ply that has been varied to optimize for either  $k_t$  or FPF. The 90° ply was replaced with 51°, 54°, and 0° for layups B, E, and H, respectively, resulting in change of failure mode and the observed increase in FPF. Also notable was the slight increase in FPF gained by simply varying the order of the Baseline plies, as in layup C. In this layup, the 0° and 90° ply orders were reversed and thus the 90° ply was moved further inside the laminate and away from the free edge effects of the surface of the coupon. A large FPF decrease was found for the  $k_t$ -optimized layup D, whose 0° plies were replaced with 57° plies. From Table 2.4.1, layup D had the lowest  $k_t$  out of the single angle variations, but also showed the lowest strength, indicating that removing the primary load bearing plies negatively effects FPF, regardless of the beneficial reduction of  $k_t$ . The replacement of the 0° plies with 57° plies essentially made Layup D an in-plane shear coupon, resulting in the large non-linear behavior exhibited by this laminate. The  $k_t$ -optimized Layup A, in which the  $\pm 45^\circ$  plies were replaced with  $\pm 54^\circ$  plies, also showed an FPF decrease. Thus, by replacing the  $\pm 45^\circ$  plies with plies whose angles are further away from the load bearing direction, FPF decreased, supporting the conclusions

drawn for layup D. This is due to the increase in longitudinal stress resulting from diverting plies away from the longitudinal axis (0.0181 in/in for BL compared to 0.0202 in/in for A and 0.0324 in/in for D)<sup>10</sup>, resulting in a lower FPF for the transverse failure mode of the 90° plies. When the ±45° plies were moved closer to the load bearing axis (as in the ±21° plies of layup G), or when the number of primary load bearing plies was increased (90° replaced with 0° plies for layup H), the FPF increased.



**Figure 5.5.1:** Experimental first ply failure for non-traditional layups using DICM and the proportional limit

**Table 5.5.1:** Number of DICM coupons experimentally tested for Fig. 5.5.1

Name	BL	A	B	C	D	E	F	G	H
#	8	8	8	5	9	5	3	2	3

<sup>10</sup> Calculated at the longitudinal strain maximum (Location A, Fig. 2.4.1) by FEA at 66.7 Ksi.

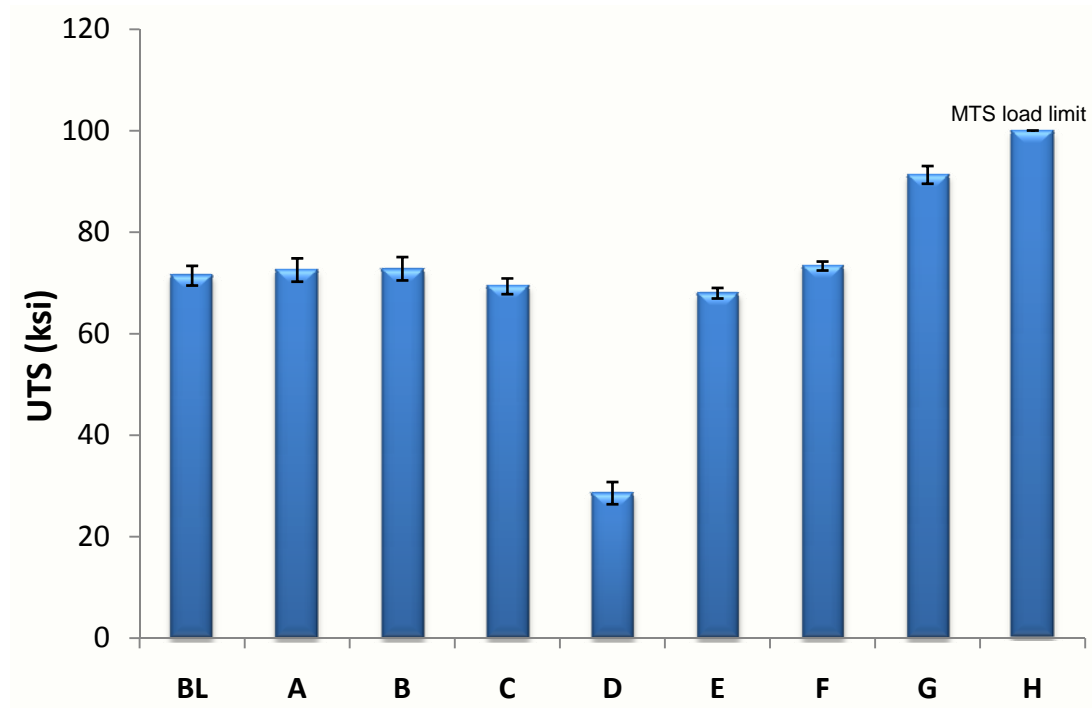
Increases in UTS were found in some of the non-traditional laminates, as shown by Fig. 5.5.2. The number of coupons tested for each laminate are given in Table 5.5.2. Selected photographs of coupons at or near failure are shown in Appendix I. The shear failure of Fig. I.1 and the catastrophic failure of Fig. I.2 are representative of the non-traditional laminates tested. The largest strength increase was found in layup H (Fig. 5.5.2), whose UTS was above the load capabilities of the MTS tester, and far above the Baseline. A similar result was obtained for UTS and FPF of Layup H – as the primary load bearing plies were increased, strength was increased, as expected. The increase in UTS comes with a material property trade-off, as seen by the reduction of the transverse modulus of the H layup by over a factor of two (material properties are given in App. A, Table A.4 and are shown normalized to baseline in Fig. 5.5.3). This reduction in modulus could be limited during the optimization process using CLT for industrial applications of this method.

A large increase in UTS was also obtained for layup G, with much less material property trade off than for layup H (Fig. 5.5.3). For layup G, the  $\pm 45^\circ$  Baseline plies were replaced with  $\pm 21^\circ$  plies, which resulted in a UTS increase because ply angles were shifted closer to the primary load bearing axis (the same reason layup G obtained an FPF increase). Similarly, as plies were shifted away from the longitudinal axis (as for Layup D and E), the UTS was decreased. Layup D saw the largest decrease in UTS because it was the only layup without  $0^\circ$  plies.

Strength was increased or decreased depending on the location of the  $0^\circ$  plies. Layup C switched the order of the Baseline  $0^\circ$  and  $90^\circ$  plies, thus moving the  $0^\circ$  ply outward resulting in a UTS decrease. Layup F varied the order and moved the  $90^\circ$  ply further

inward, resulting in a UTS increase. This result is consistent with those predicted by Tay et. al. [2], and underscores the importance of stacking sequence.

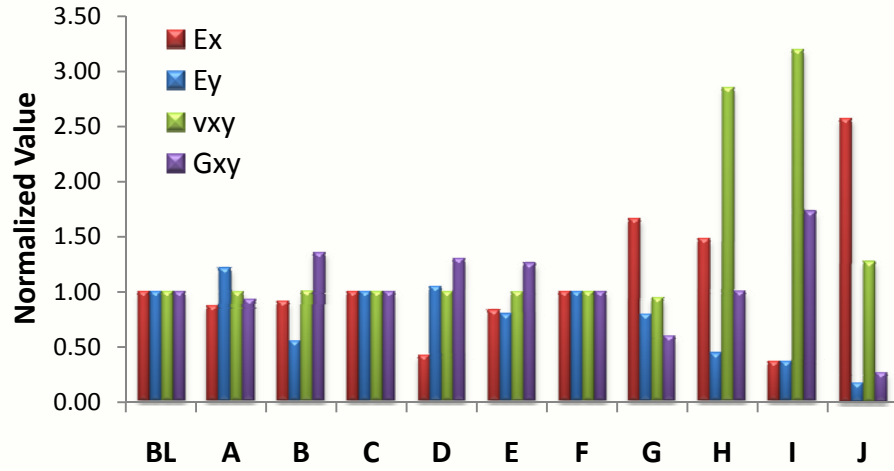
A UTS increase was also found for layups A and B, which were optimized for  $k_t$ . This shows that the strain concentration is an important variable in damage progression and in designing non-traditional laminates. Specifically, altering the  $90^\circ$  and  $\pm 45^\circ$  plies can lead to beneficial strength results, while altering the  $0^\circ$  load bearing plies should be avoided.



**Figure 5.5.2:** Experimental ultimate tensile stress for non-traditional layups

**Table 5.5.2:** Number of DICM coupons experimentally tested for Fig. 5.5.2

Name	BL	A	B	C	D	E	F	G	H
#	11	11	11	8	11	6	3	3	3



**Figure 5.5.3:** Material properties of non-traditional laminates normalized to the baseline (BL) layup

## 6: Conclusion

Non-traditional laminates provide the opportunity for increased control over strength and material properties. These laminates can be tailored to meet specific needs and loading scenarios commonly experienced in engineering structures. This study has shown the potential for layup tailoring using tools readily available in industry, such as linear two-dimensional finite element analysis, failure criteria, and classical lamination theory. These tools can be used to obtain improvements in FPF, as shown by the spatial strain measurements of DICM and Moiré Interferometry. Furthermore, strength can be increased, sometimes with little change in material properties.

DICM was found to be a valuable tool for the measurement of full-field surface strain. DICM was verified using FEA and Moiré Interferometry. The usefulness of this method lies in the ease in which strain data can be produced; where Moiré Interferometry can only be used at static loads and requires extensive setup, DICM can be used for the entire loading of the coupon and only requires a speckle pattern and camera calibration. Like all measurement techniques, DICM has limits, and is currently not optimal for very low strains (below 0.001 in/in). However, DICM worked well for strains above 0.001 in/in for T600: 125-33 laminates in open-hole tension. The strain profile and stress-strain plots revealed the development of the surface strain during damage progression. This was used to quantify FPF, defined as the proportional limit, which compared well to the Maximum Strain Criterion. The importance of DICM is underscored by the World-Wide Failure Exercise [35], in which initial failure data was either not available or of poor quality. The DICM results presented in Chapter 5 and Appendix G show that the FPF can be found by locating trends in the stiffness change at the proportional limit.

The Maximum Strain and Maximum Stress Failure Criteria performed far better than any of the other criteria tested for predicting FPF. While the failure mode predictions remain to be verified by x-ray [42], the results agree with expectations – the majority of first ply failures were in the transverse direction of the 90° plies due to the large longitudinal strain at the strain concentration of the hole. This failure mode was predicted to change to  $\pm 45^\circ$  (or near  $\pm 45^\circ$ ) shear failure only for laminates with the 90° plies replaced with near 45° plies. The corresponding increase in FPF for these laminates is also as expected, and was consistent with DICM results.

The FPF load capacity was improved for Layups B, C, E, G, and H, and the UTS load capacity was improved for Layups A, B, F, G, and H. This improvement was achieved with little or no variation from the baseline elastic properties for layups A, C, E, and F. Generally speaking, material properties declined when the ply orientation dominating that property changed if quasi-isotropic response is desired. For Layup D, the variation in the 0° ply was detrimental to both FPF and UTS due to the large decrease in the longitudinal modulus, even though the largest strain concentration reduction was achieved for this layup. Layup H achieved the largest increase in both FPF and UTS, but the replacement of the 90° ply also caused the largest decrease in the transverse modulus. Thus, Layup D and H are not recommended, and show that the load bearing 0° plies should not be altered, and that the 90° should not be drastically changed. For other layups, the replacement of a given ply did not greatly reduce the corresponding material properties. This occurred during the replacement of the 90° plies with 51° plies for layup B, and the  $\pm 45^\circ$  plies with  $\pm 21^\circ$  plies for layup G. While these replacements resulted in a decrease in the transverse and shear modulus for Layup B and G, respectively, large increases in strength were

found for a relatively small material property trade-off. This suggests that the  $90^\circ$  and  $\pm 45^\circ$  ply orientations could be tailored for a specific loading scenario, with limits placed on the extent of this variation. If no limits were imposed, Layups B and G are suggested to increase both FPF and UTS in the open-hole tension loading scenario. Another significant strength improvement was achieved for Layup C, where the FPF strength increased simply by switching the order of the  $0^\circ$  and  $90^\circ$  plies. Similarly, for Layup F, simply moving the  $90^\circ$  ply inward caused an increase in UTS.

This study has shown that non-traditional laminates can be tailored to provide an increased load capacity for a given loading scenario. The opportunity clearly exists for strength optimization, though limits should be imposed on the extent of the material property trade-off. The non-traditional laminates fabricated in this study were optimized for open-hole tension, and improvements in FPF and UTS were found. The methodology presented here could be easily adopted in industry using FEA, CLT, and the Maximum Strain Criteria. Recommendations for further research include CT scans to verify FPF, nonlinear FEA with damage progression to understand  $k^{\text{th}}$  ply failures, and analysis of through-the-thickness effects to account for interlaminar ply interactions.



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## Appendix A: Tables

**Table A.1:** T600: 125-33 Composite Material System (psi)<sup>11</sup>

$E_1 = 19.73 \times 10^6 \pm 0.65 \times 10^6$	$E_2 = 1.38 \times 10^6 \pm 0.021 \times 10^6$	$E_3 = E_2$
$G_{12} = 0.81 \times 10^6 \pm 0.022 \times 10^6$	$G_{13} = G_{12}$	$\nu_{12} = 0.298 \pm 0.019$
$\nu_{23} = 0.35$	$\nu_{13} = \nu_{12}$	$S_L^{(+)} = 332.8 \times 10^3 \pm 16.2 \times 10^3$
$S_T^{(+)} = 6.28 \times 10^3 \pm 0.63 \times 10^3$	$S_{LT} = 9.03 \times 10^3 \pm 0.37 \times 10^3$	$S_L^{(-)} = 209 \times 10^3$
$S_T^{(-)} = 33 \times 10^3$		

**Table A.2:** Aluminum Alloy 7075<sup>12</sup> (psi)

$E_x = 10.3 \times 10^6$	$\nu = 0.33$
--------------------------	--------------

**Table A.3:** Fabricated laminates

Designation	Layup	Description	Method
BL	[(45/90/-45/0)2]s	baseline	Baseline
A	[(54/90/-54/0)2]s	vary 45	kt
B	[(45/51/-45/0)2]s	vary 90	kt
C	[(45/0/-45/90)2]s	vary order 1	kt
D	[(45/90/-45/57)2]s	vary 0	kt
E	[(54/54/-54/0)2]s	vary 45&90	kt
F	[(45/-45/90/0)2]s	vary order 2	literature
G	[(21/90/-21/0)2]s	vary 45	FPF
H	[(45/0/-45/0)2]s	vary 90	FPF
I	[(±45)4]s	shear failure	failure mode
J	[0]16	uniaxial failure	failure mode

**Table A.4:** Material properties using classical lamination theory

Property	BL	A	B	C	D	E	F	G	H	I	J
$E_x$ (Msi)	7.7	6.7	7.0	7.7	3.2	6.4	7.7	12.8	11.4	2.8	19.7
$E_y$ (Msi)	8	9	4	8	8	6	8	6	3	3	1
$\nu_{xy}$	0.23	0.23	0.23	0.23	0.23	0.23	0.23	0.22	0.67	0.75	0.30
$G_{xy}$ (Msi)	3	3	4	3	4	4	3	2	3	5	1

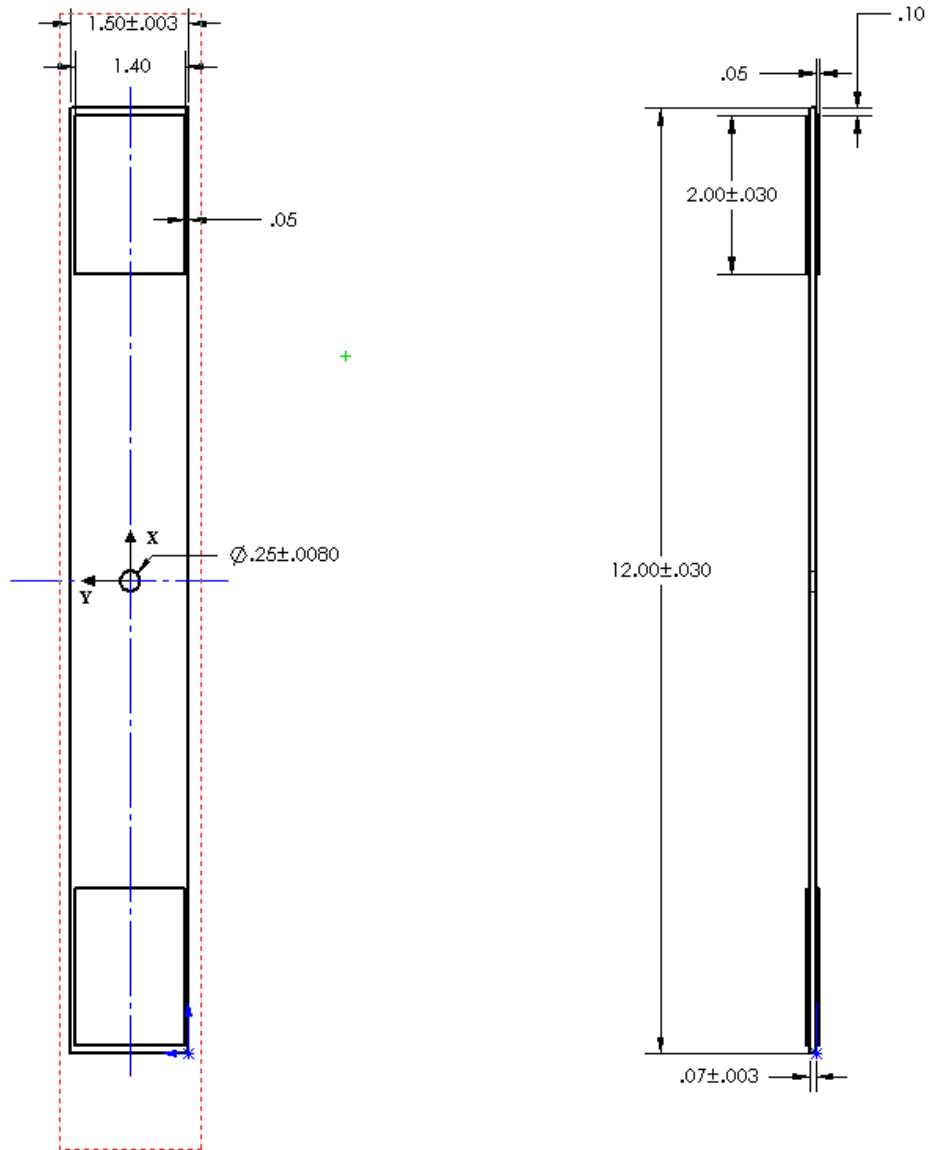
**Table A.5:** Failure indexes of non-traditional laminates at Location A

Max Strain	BL	A	B	C	D	E	F	G	H	I	J
e1/e1c	0.48	0.41	1.62	0.48	0.91	0.10	0.48	0.27	1.25	0.47	1.23
e2/e2c	5.89	6.59	3.83	5.89	10.58	3.22	5.89	4.74	3.05	1.27	0.20
e12/e12c	0.00	0.00	0.00	0.00	0.00	2.44	0.00	0.00	0.00	5.58	0.00

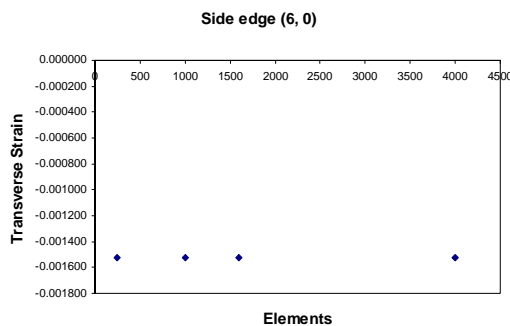
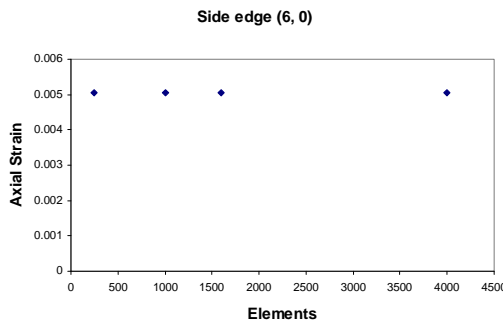
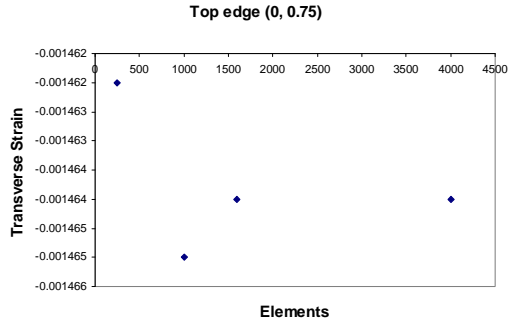
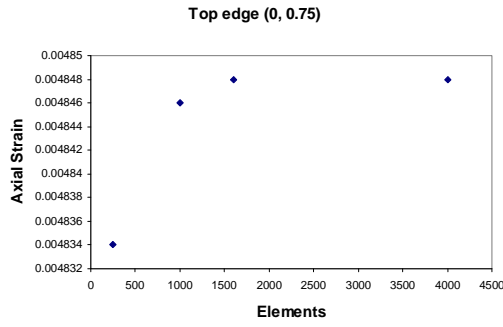
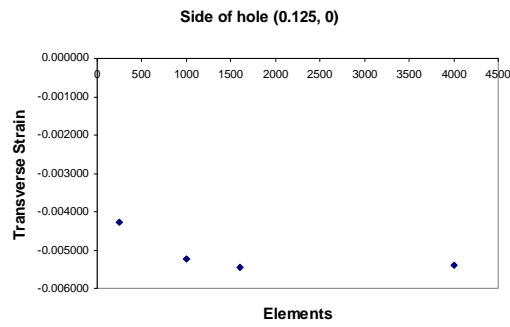
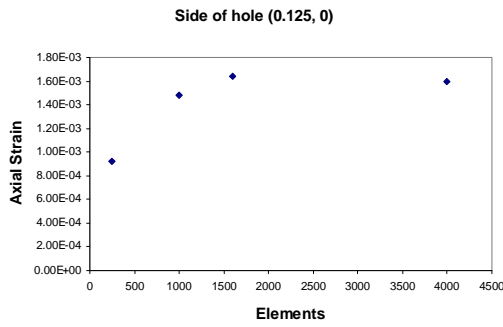
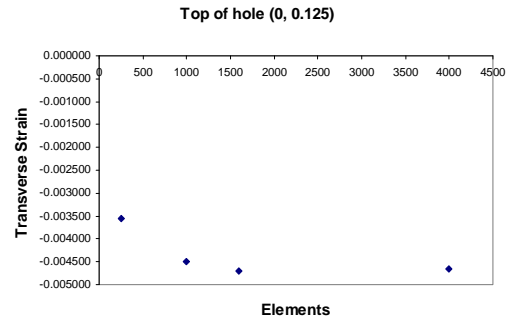
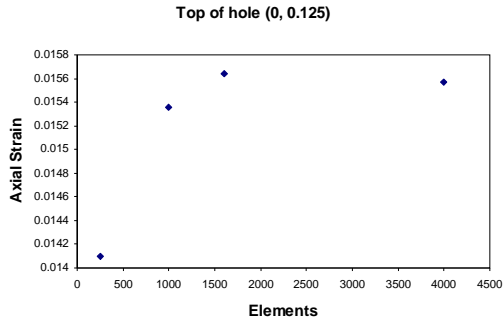
<sup>11</sup> Properties were found from in-plane tests of [0]<sub>6</sub>, [90]<sub>16</sub>, and [±45]<sub>2s</sub> that were performed at the WSU Environmental Exposure Facility by Matt Shultz. The out-of-plane material properties ( $\nu_{23}$ ) and compressive strengths were estimated from AS4/3501-6 [40] where the standard deviation was not reported.

<sup>12</sup> Material properties for Aluminum Alloy 7075 from [57].

## Appendix B: Coupon Dimensions



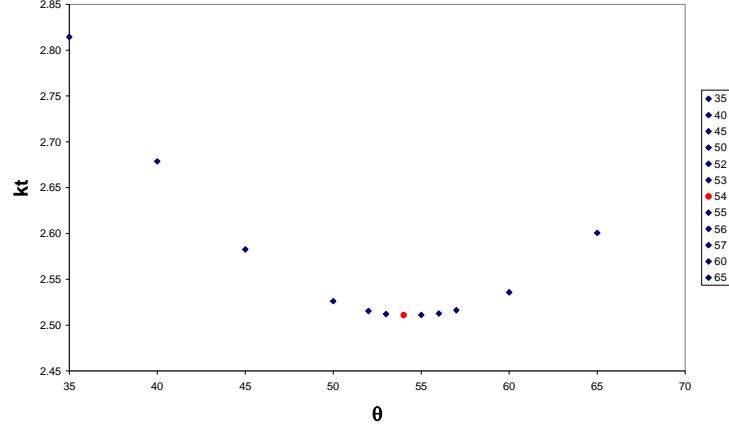
## Appendix C: Convergence Study for Locations (X,Y)



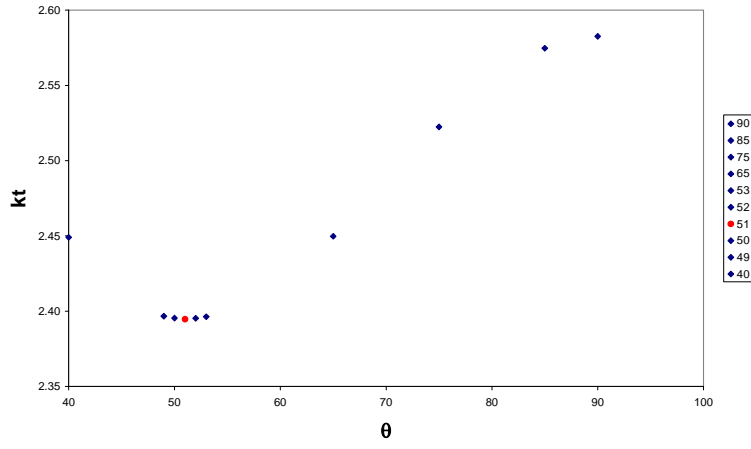
# Appendix D: Optimization using FEA

## D.1: Decrease the strain concentration, $k_t$

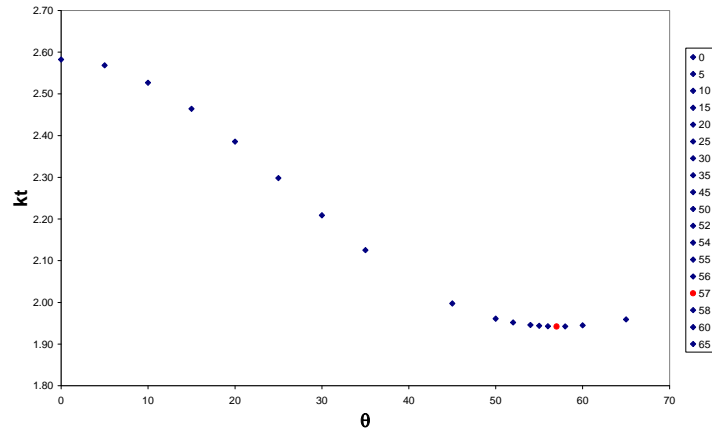
Vary  $\pm 45^\circ$



Vary  $90^\circ$



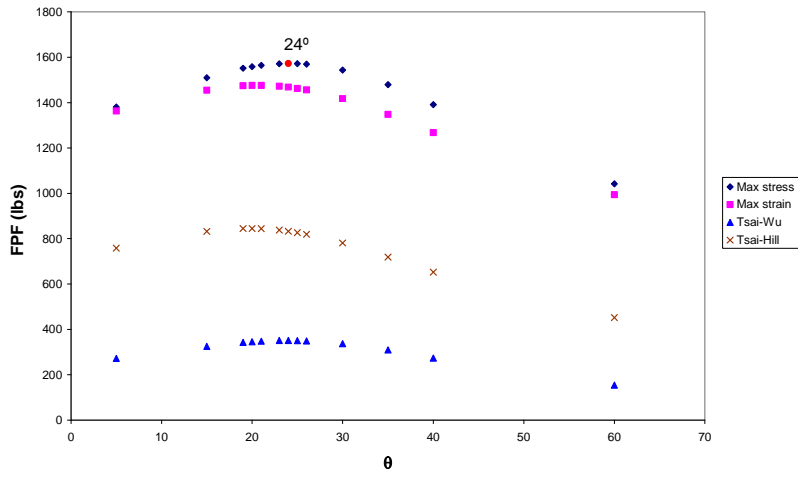
Vary  $0^\circ$



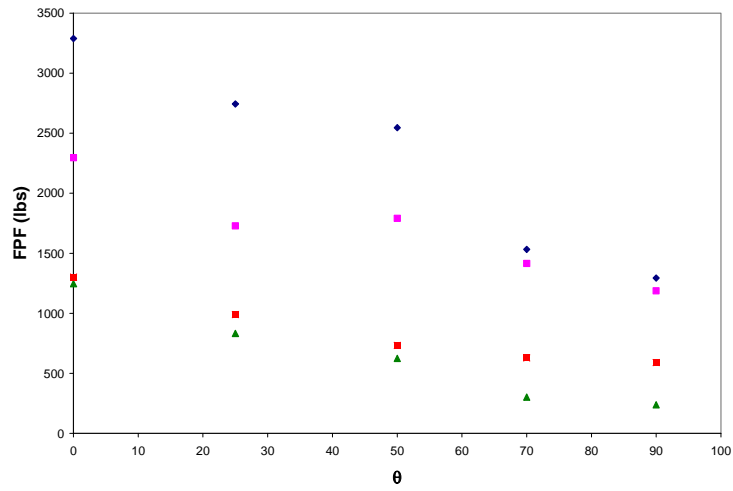


## D.2: Increase the first ply failure, FPF

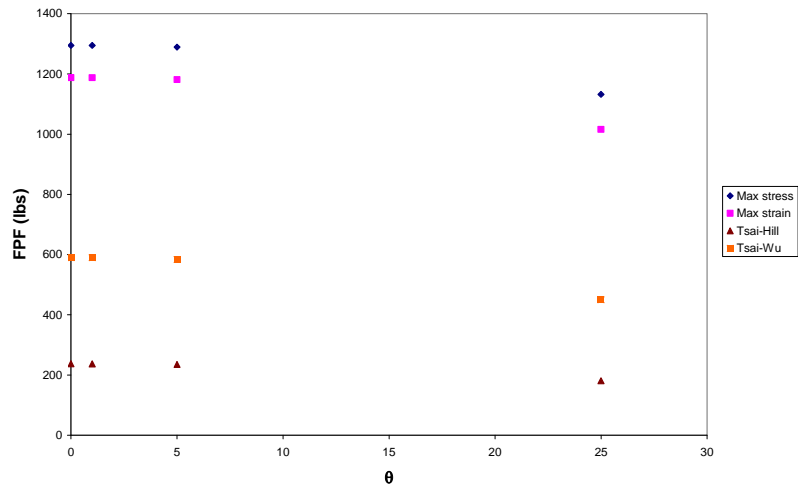
Vary  $\pm 45^\circ$



Vary  $90^\circ$



Vary  $0^\circ$



### D.3: Example of a double angle optimization

Vary 45° & 0°

Ply orientation replacing 45°

	45°	60	61	62	63	64	65
0°							
54							1.7992
55							
56							
57				1.7764			
58		1.7743					
59						1.7742	
60			1.7695	1.7693	1.7703		
61			1.7688	1.7685	1.7692		
62			1.7687	1.7683	1.7690		
63			1.7692	1.7686	1.7691		
64							
65		1.7739					
66			1.7739				
67							
68							
69							
70		1.7892					
71					1.7894		

Lowest kt, replacing 0° and 45° with 60°

## Appendix E: Matlab Program for Failure Criteria

```
%Daniel Stone
%This program uses failure theories to predict strength of a composite
%Inputs are the 2D strain state at a point (FEA), load, and ply angles
%Outputs are the strengths for each ply from max stress, max strain,
%Tsai-Hill, Tsai-Wu, Hoffman, Hashin, and Yamada-Sun

clear
clc
format long

%Material properties for T600: 125-33
E1=19.73*10^6;
E2=1.3775*10^6;
G12=0.80*10^6;
v12=0.2976;
v21=0.02575;
%Stiffness matrix
Q11=E1/(1-v12*v21);
Q12=v12*E2/(1-v12*v21);
Q22=E2/(1-v12*v21);
Q66=G12;
Q=[Q11 Q12 0,
   Q12 Q22 0,
   0 0 Q66];

%Strength
%Ultimate tensile stress
S1=332.8*10^3;
S2=6.28*10^3;
S12=9.031*10^3;
%Ultimate compressive stress
S1c=209*10^3;
S2c=33*10^3;
%Ultimate tensile strain
e1=S1/E1;
e2=S2/E2;
e12=S12/G12;
%Ultimate compressive strain
e1c=S1c/E1;
e2c=S2c/E2;

%Input ply angles
theta(1)=abs(input('1rst ply angle?'));
theta(2)=abs(input('2nd ply angle?'));
theta(3)=abs(input('3rd ply angle?'));
theta(4)=abs(input('4rth ply angle?'));

%Input load and strain state
load=input('Load (lbs)?');
epsilonx=input('Axial Strain?');
epsilony=input('Transverse Strain?');
gammaxy=input('Shear Strain?');

%Strain vector at a location on the laminate
e=[epsilonx epsilony gammaxy/2];
```

```

%Ply angle vector
theta=theta';
%Ply angle vector in radians
theta=theta*pi/180;

%Calculate strains and stresses in the fiber direction
for k=1:3
    ply=theta(k);
    c=cos(ply);
    s=sin(ply);
    %Transformation matrix
    T=[c^2 s^2 2*c*s,
        s^2 c^2 -2*c*s,
        -c*s c*s (c^2-s^2)];
    %Strain vector in the fiber direction
    strain_fiber=T*e;
    epsilon11(k)=strain_fiber(1);
    epsilon22(k)=strain_fiber(2);
    gamma12(k)=2*strain_fiber(3);
    %Stress vector in the fiber direction
    stress_fiber=Q*[epsilon11(k) epsilon22(k) gamma12(k)];
    sigma11(k)=stress_fiber(1);
    sigma22(k)=stress_fiber(2);
    tau12(k)=stress_fiber(3);
end

%Maximum stress criterion
for k=1:3
    %failure index
    if sigma11(k)>0
        maxstress(1)=abs(sigma11(k)/S1);
    else
        maxstress(1)=abs(sigma11(k)/S1c);
    end
    if sigma22(k)>0
        maxstress(2)=abs(sigma22(k)/S2);
    else
        maxstress(2)=abs(sigma22(k)/S2c);
    end
    maxstress(3)=abs(tau12(k)/S12);
    fi_maxstress = max(maxstress);
    %strength of ply
    failure_maxstress(k)=load/fi_maxstress;
end
%ply failures
failure_maxstress;

%Maximum strain criterion
for k=1:3
    %failure index
    if epsilon11(k)>0
        maxstrain(1)=abs(epsilon11(k)/e1);
    else
        maxstrain(1)=abs(epsilon11(k)/e1c);
    end
    if epsilon22(k)>0

```

```

    maxstrain(2)=abs(epsilon22(k)/e2);
else
    maxstrain(2)=abs(epsilon22(k)/e2c);
end
maxstrain(3)=abs(gamma12(k))/e12;
%To see failure indices, delete percentages in the following lines:
%k
%maxstrain
fi_maxstrain=max(maxstrain);
%strength of ply
failure_maxstrain(k)=load/fi_maxstrain;
end
%kth ply failure
failure_maxstrain;

%Tsai-Hill criterion
for k=1:3
    %failure index
    fi_tsaihill = (sigma11(k)/S1)^2 + (sigma22(k)/S2)^2 + (tau12(k)/S12)^2 - sigma11(k)*sigma22(k)/S1^2;
    %strength of ply
    failure_tsaihill(k)=load/fi_tsaihill;
end
%kth ply failure
failure_tsaihill;

%Tsai-Wu criterion
for k=1:3
    %failure index
    fi_tsaiwu = sigma11(k)^2/(S1*S1c) + sigma22(k)^2/(S2*S2c) + (tau12(k)/S12)^2 + (1/S1-1/S1c)*sigma11(k) + (1/S2-
1/S2c)*sigma22(k) - sigma11(k)*sigma22(k)/(2*S1*S1c);
    %strength of ply
    failure_tsaiwu(k)=load/fi_tsaiwu;
end
%kth ply failure
failure_tsaiwu;

%Hoffman criterion (same as Tsai-Wu except last term)
for k=1:3
    %failure index
    fi_hoffman = sigma11(k)^2/(S1*S1c) + sigma22(k)^2/(S2*S2c) + (tau12(k)/S12)^2 + (1/S1-1/S1c)*sigma11(k) + (1/S2-
1/S2c)*sigma22(k) - sigma11(k)*sigma22(k)/(2*S1*S1c);
    %strength of ply
    failure_hoffman(k)=load/fi_hoffman;
end
%kth ply failure
failure_hoffman;

%Hashin criterion
for k=1:3
    %reserve factors
    if sigma11(k) > 0
        fi_1 = ((sigma11(k)/S1)^2 + (tau12(k)/S12)^2)^.5;
    else
        fi_1 = abs(sigma11(k))/S1c;
    end
    if sigma22(k) > 0
        fi_2 = ((sigma22(k)/S2)^2 + (tau12(k)/S12)^2)^.5;

```

```

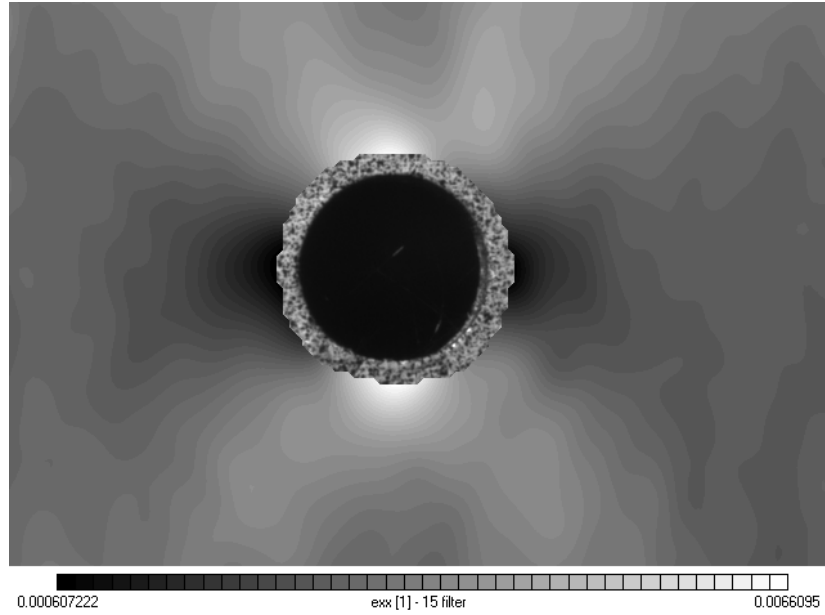
else
    a = (sigma22(k)/2*S12)^2 + (tau12(k)/S12)^2;
    b = ((S2c/2*S12)^2-1)*abs(sigma22(k)/S2c);
    c = -1;
    fi_2 = (2*a)/(-b+(b^2-4*a*c)^.5);
end
%failure index
fi_hashin = max(fi_1,fi_2);
%strength of ply
failure_hashin(k)=load/fi_hashin;
end
%kth ply failure
failure_hashin;

%Yamada-sun criterion
for k=1:3
    %failure index
    if sigma11(k)>0
        fi_yamadasun=(sigma11(k)/S1)^2 + (tau12(k)/S12)^2;
    else
        fi_yamadasun=(sigma11(k)/S1c)^2 + (tau12(k)/S12)^2;
    end
    %strength of ply
    failure_yamadasun(k)=load/fi_yamadasun;
end
%kth ply failure
failure_yamadasun;

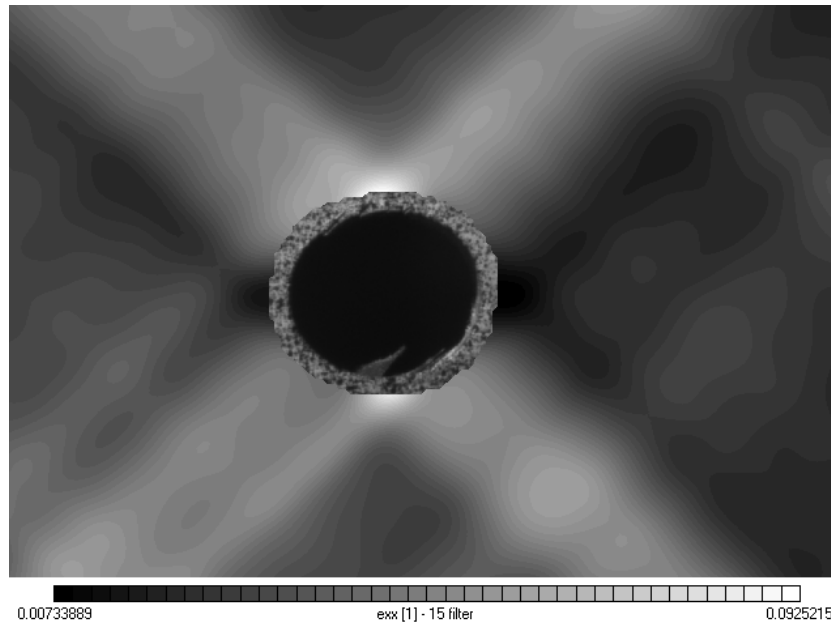
%first ply failure
FPF_maxstress=min(failure_maxstress)
FPF_maxstrain=min(failure_maxstrain)
FPF_tsaihill=min(failure_tsaihill)
FPF_tsaiwu=min(failure_tsaiwu)
FPF_hoffman=min(failure_hoffman)
FPF_hashin=min(failure_hashin)
FPF_yamadasun=min(failure_yamadasun)

```

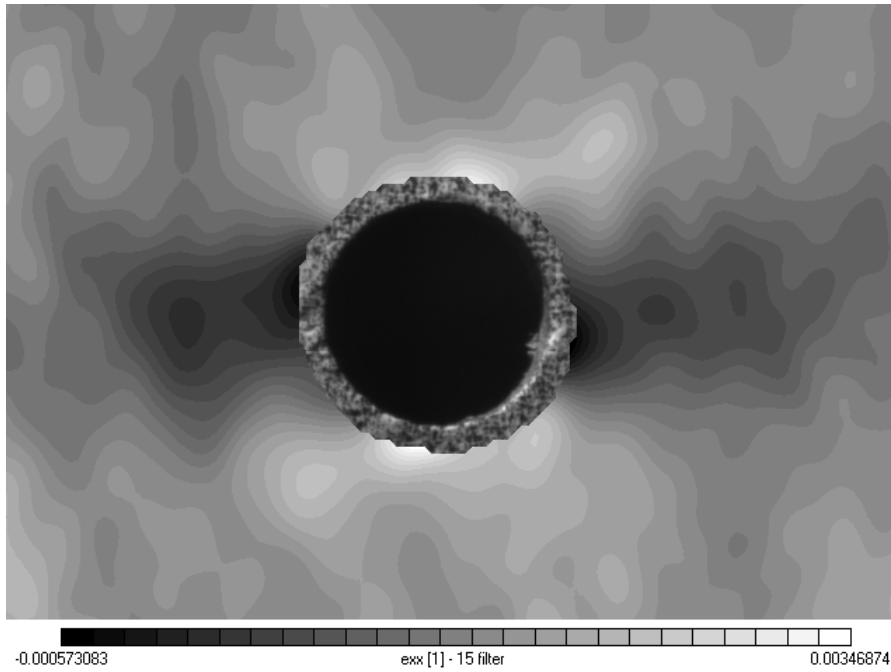
## Appendix F: Selected Strain Contours



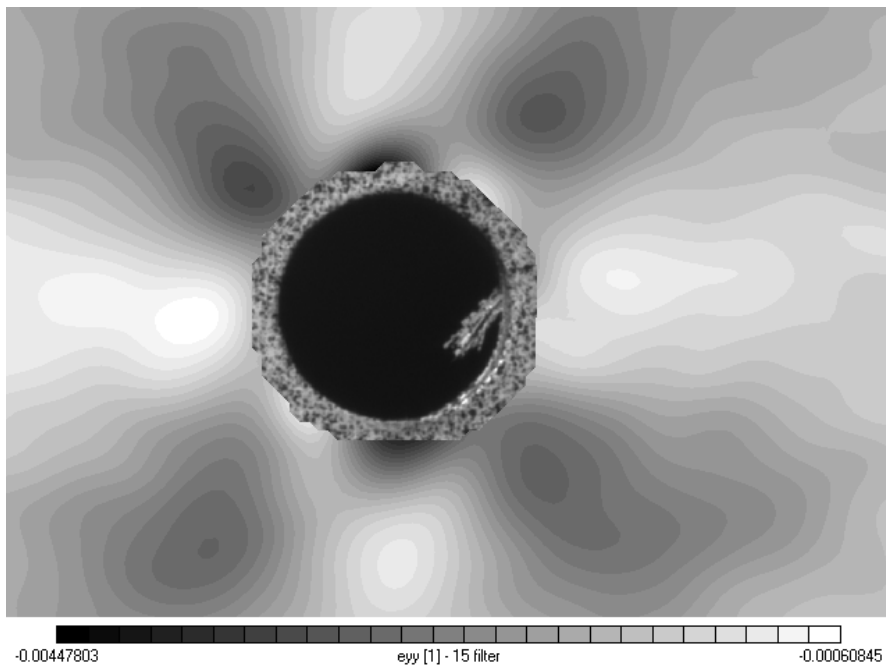
**Figure F.1:** Longitudinal strain contours for layup D at 9.43 Ksi



**Figure F.2:** Longitudinal strain contours for layup I at 14.9 Ksi

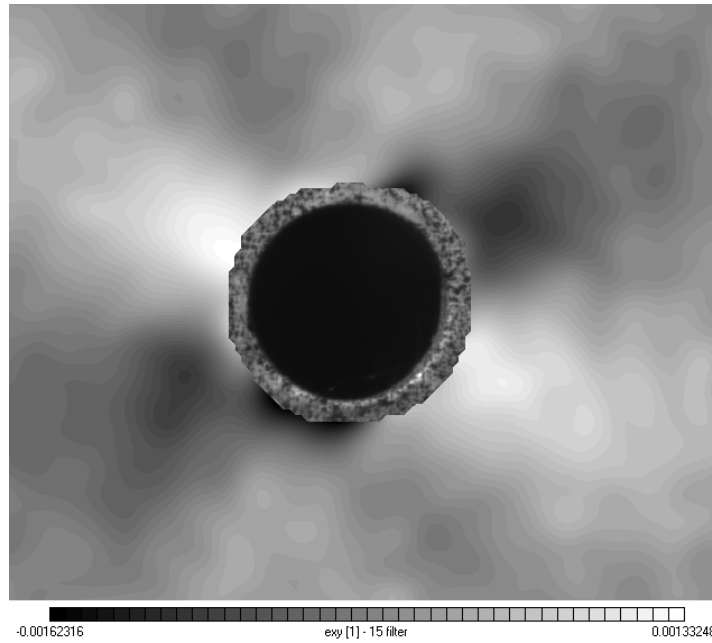


**Figure F.3:** Longitudinal strain contours for layup G at 18.3 Ksi

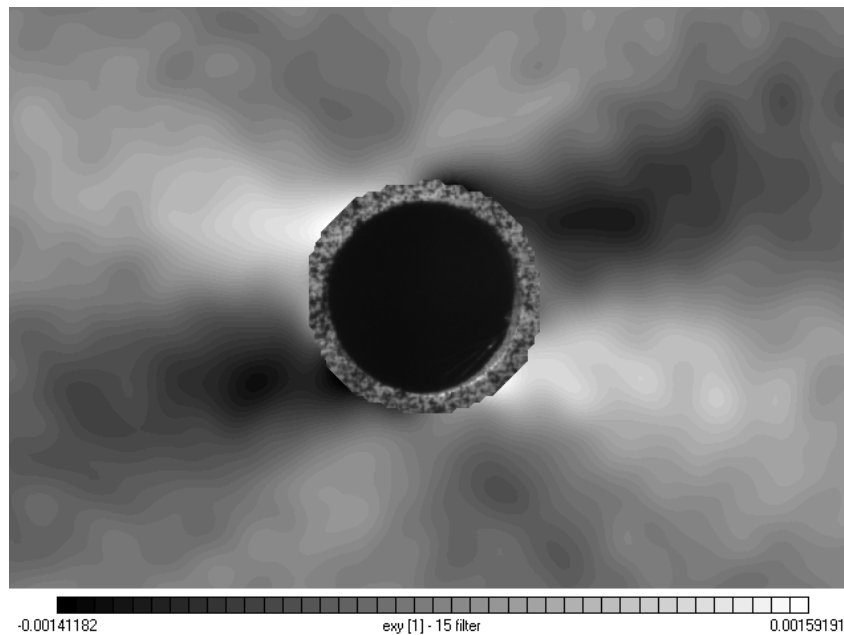


**Figure F.4:** Transverse strain contours for layup A at 39.3 Ksi



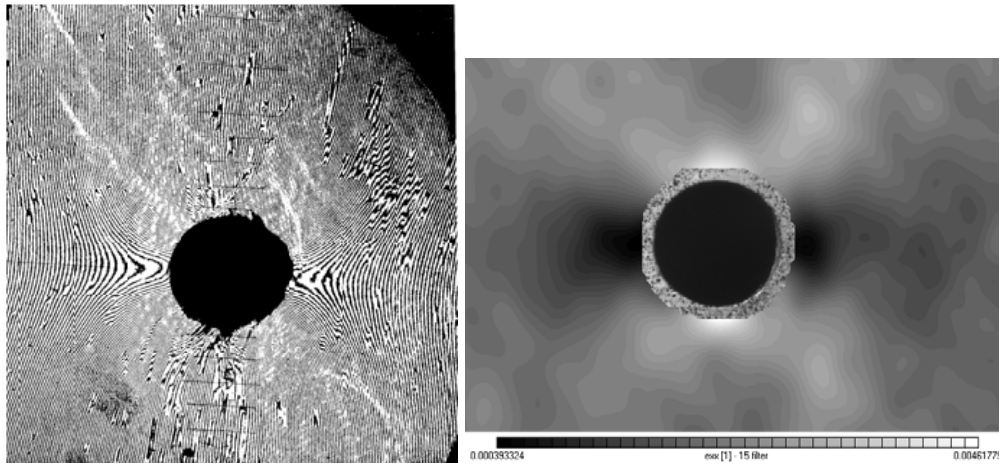


**Figure F.5:** Shear strain contours for layup E at 23.3 Ksi

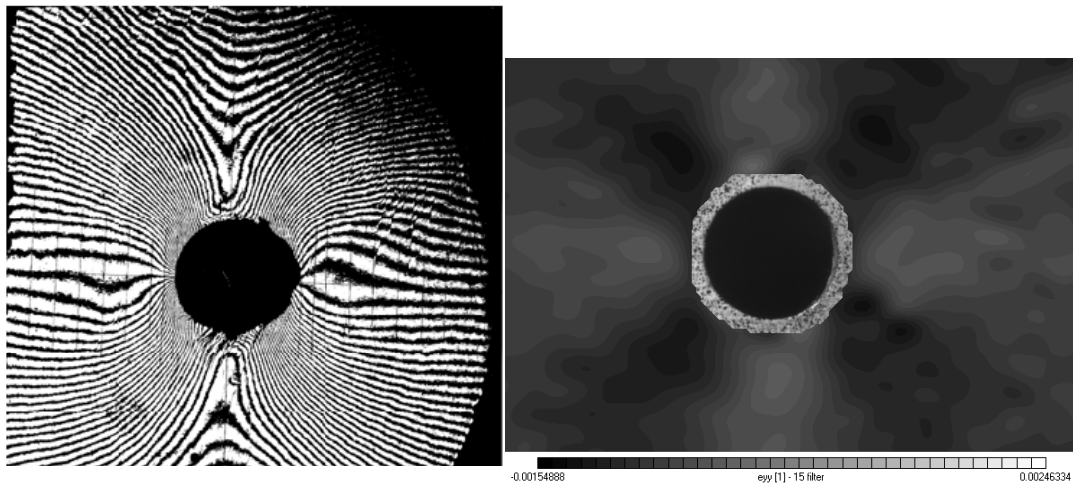


**Figure F.6:** Shear strain contours for layup H at 25.4 Ksi

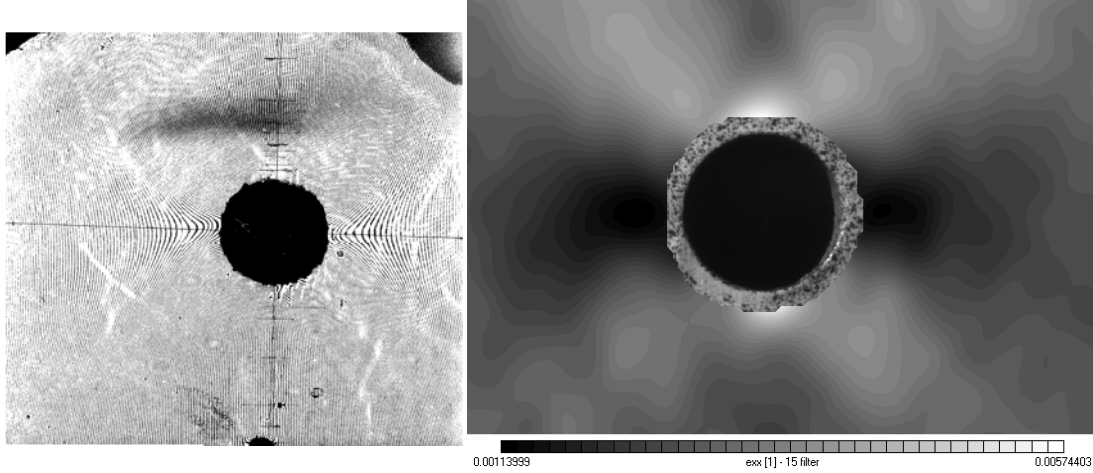
## Appendix G: DICM/Moiré Interferometry Strain Contour Comparison



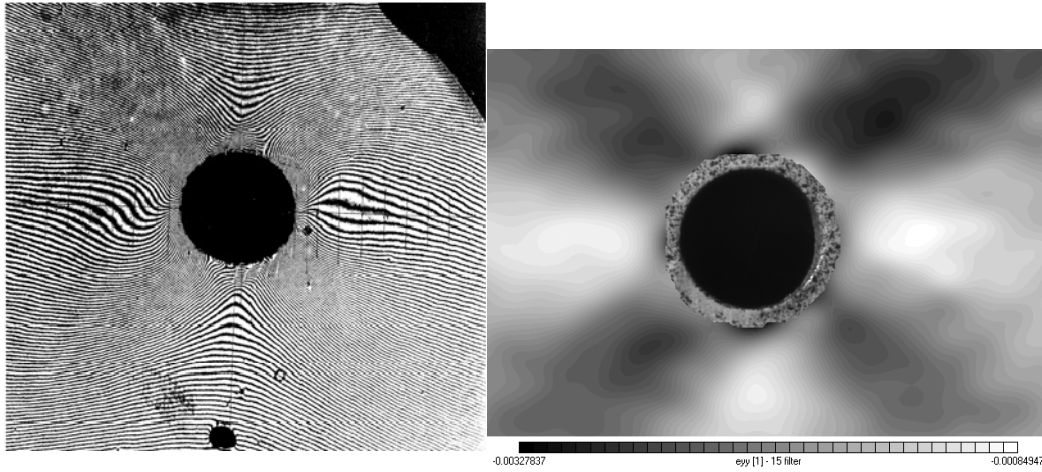
**Figure G.1:** Longitudinal Moiré deformation/DICM strain contours for the baseline layup at an applied stress of 19 Ksi



**Figure G.2:** Transverse Moiré deformation/DICM strain contours for the baseline layup at an applied stress of 19 Ksi



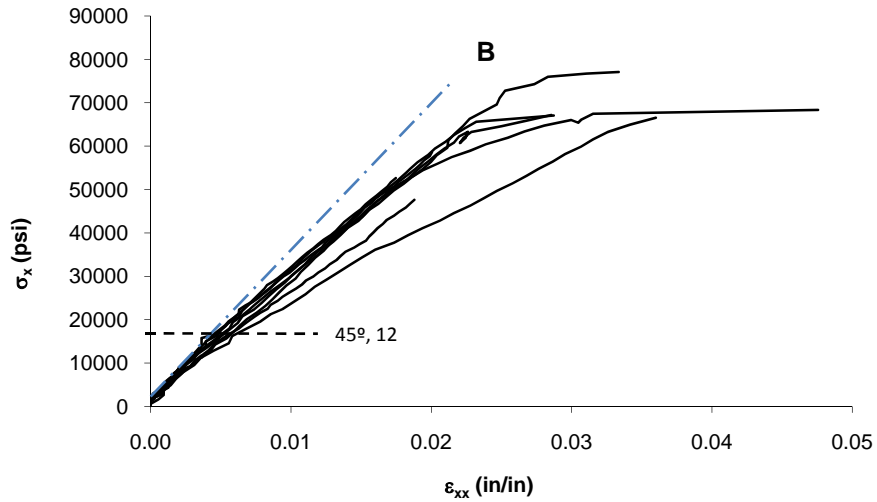
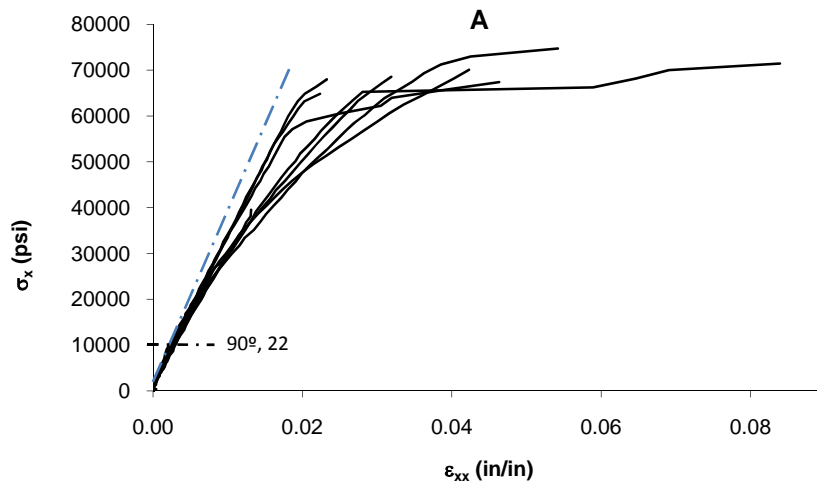
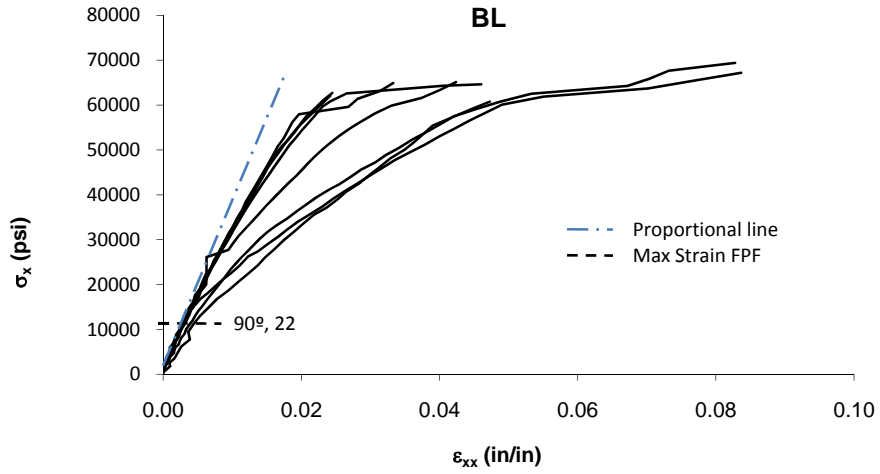
**Figure G.1:** Longitudinal Moiré deformation/DICM strain contours for the B layup at an applied stress of 19 Ksi

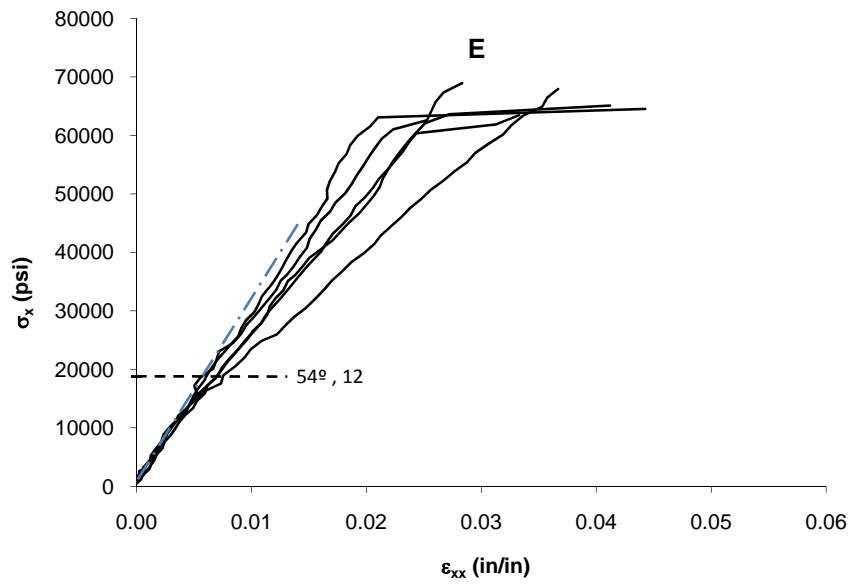
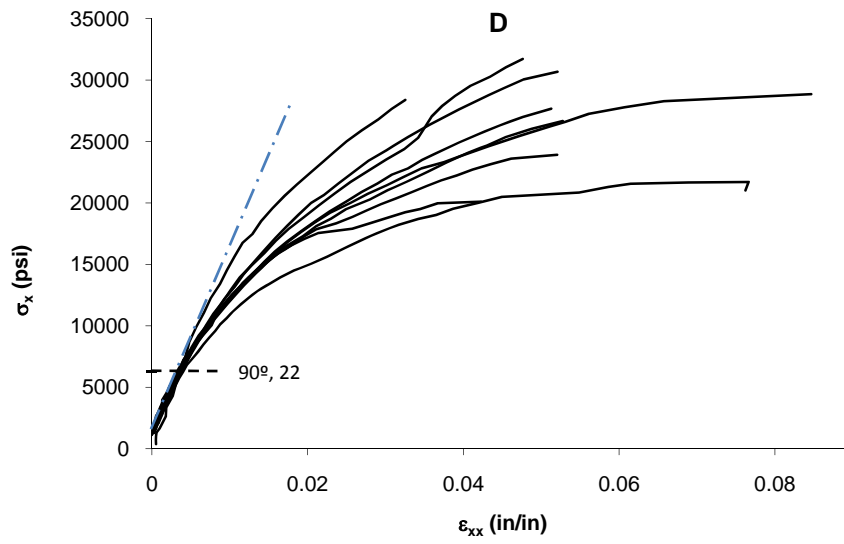
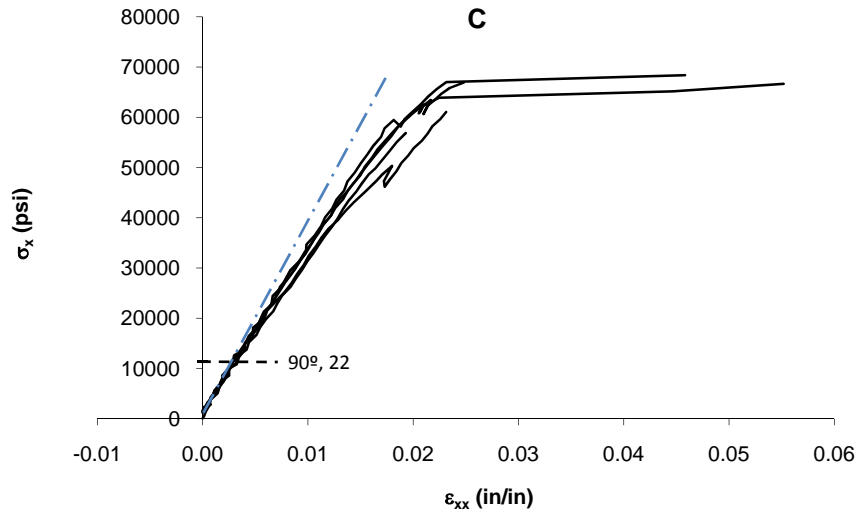


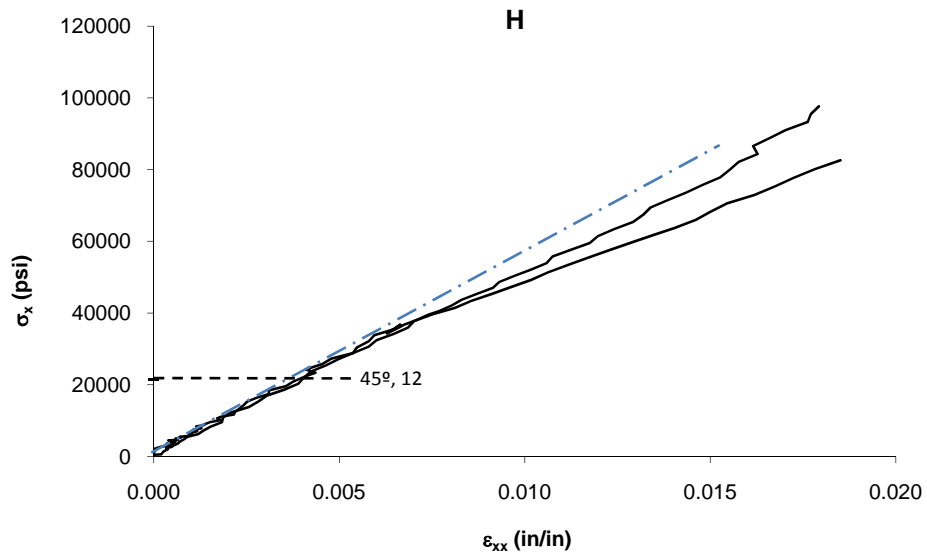
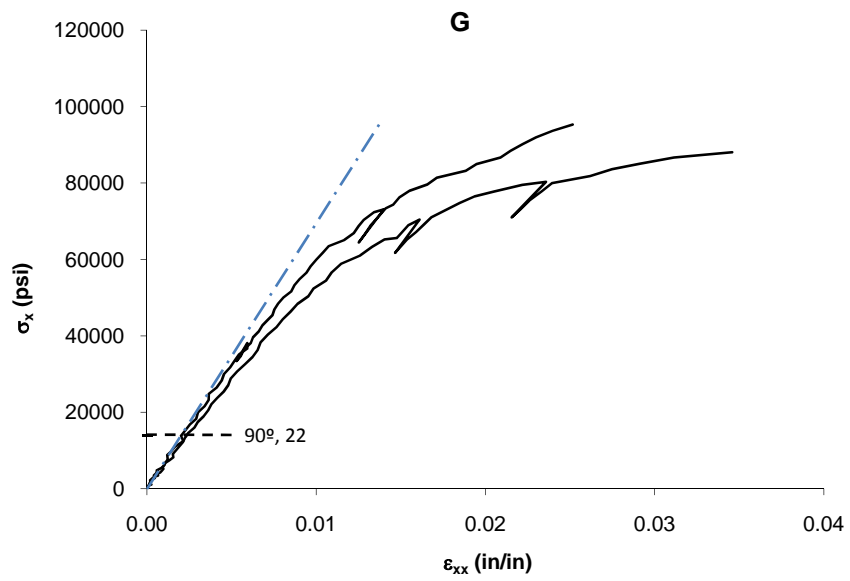
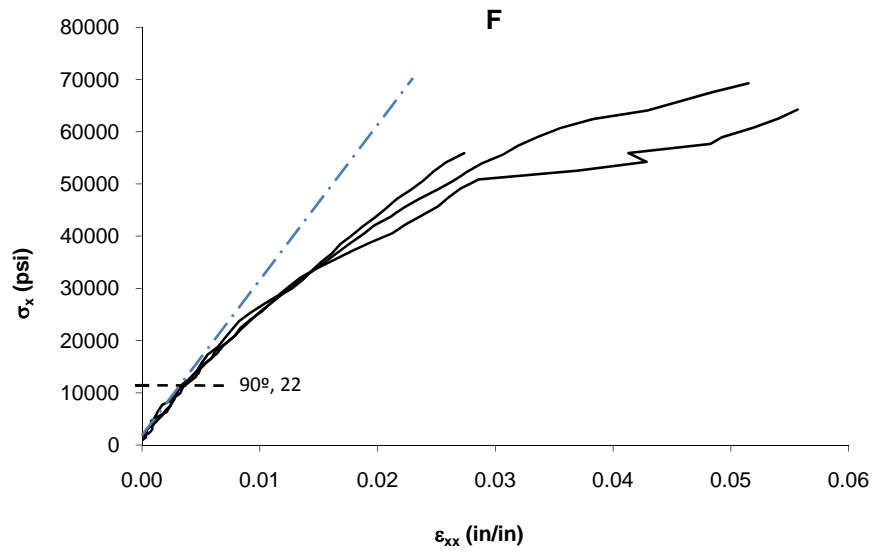
**Figure G.1:** Transverse Moiré deformation/DICM strain contours for the B layup at an applied stress of 19 Ksi

# Appendix H: Proportional Limit Compared to Maximum Strain FPF

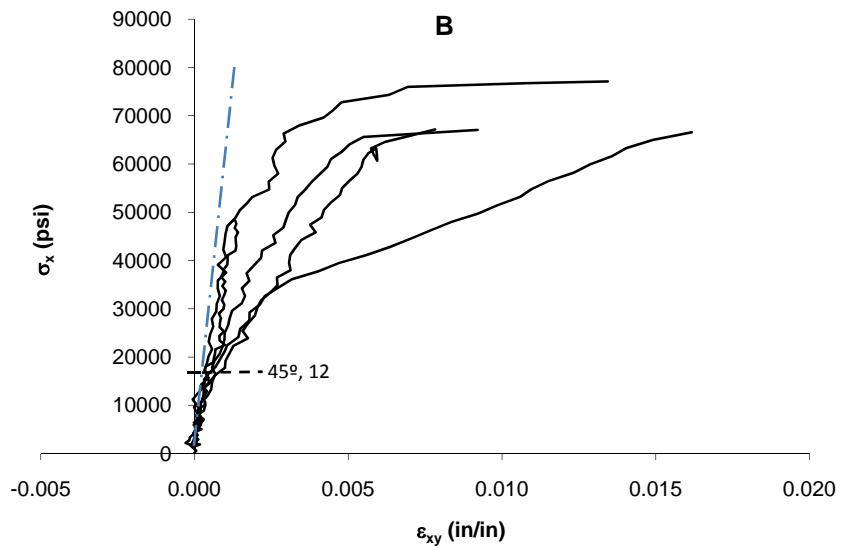
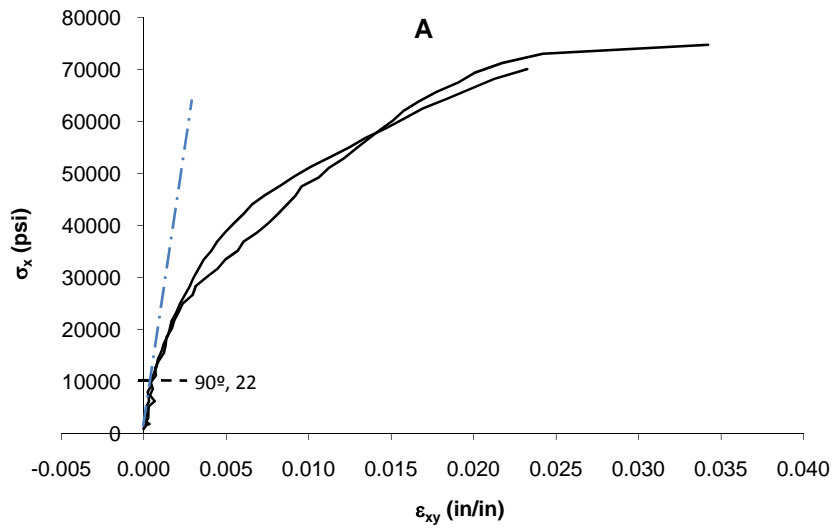
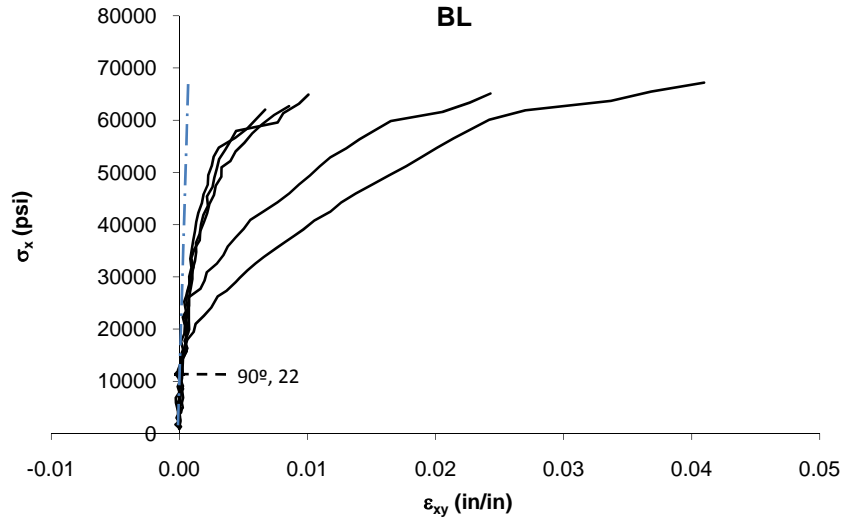
## H.1: Applied Stress vs. Longitudinal Strain

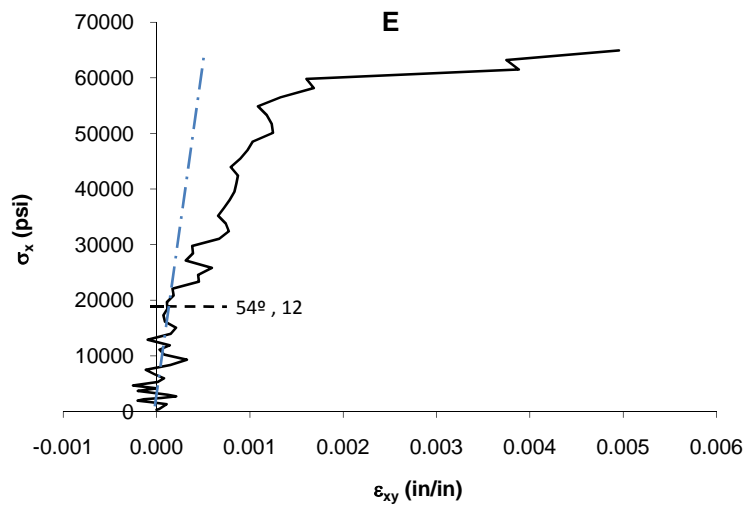
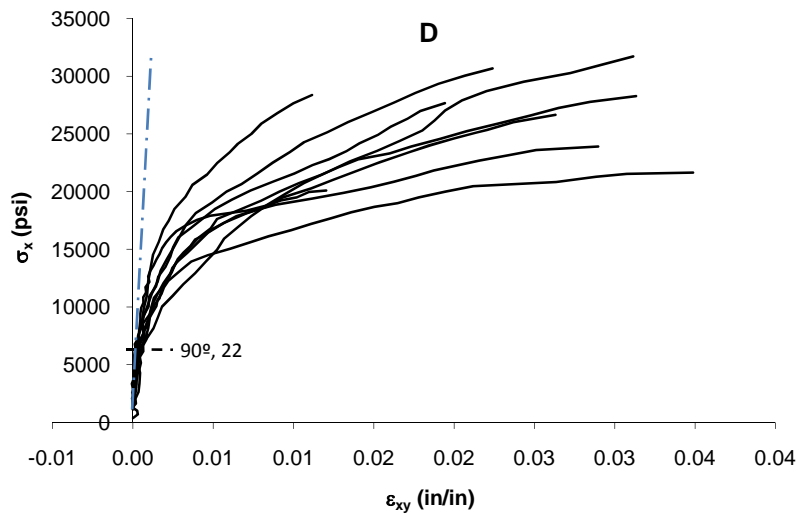
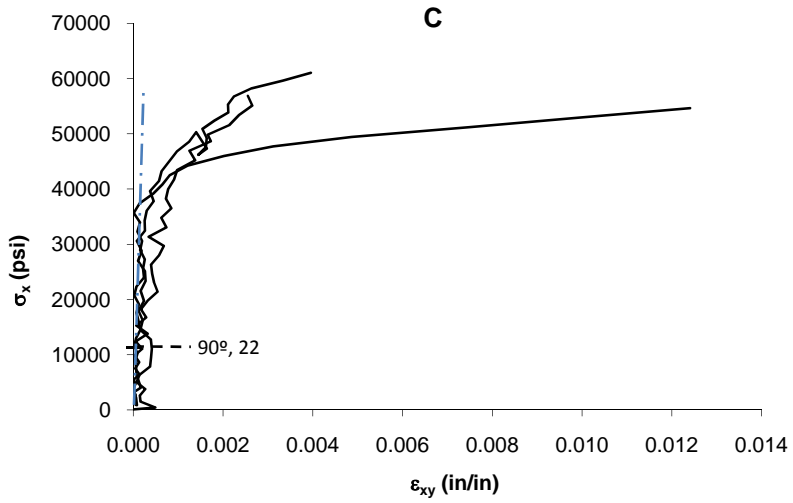




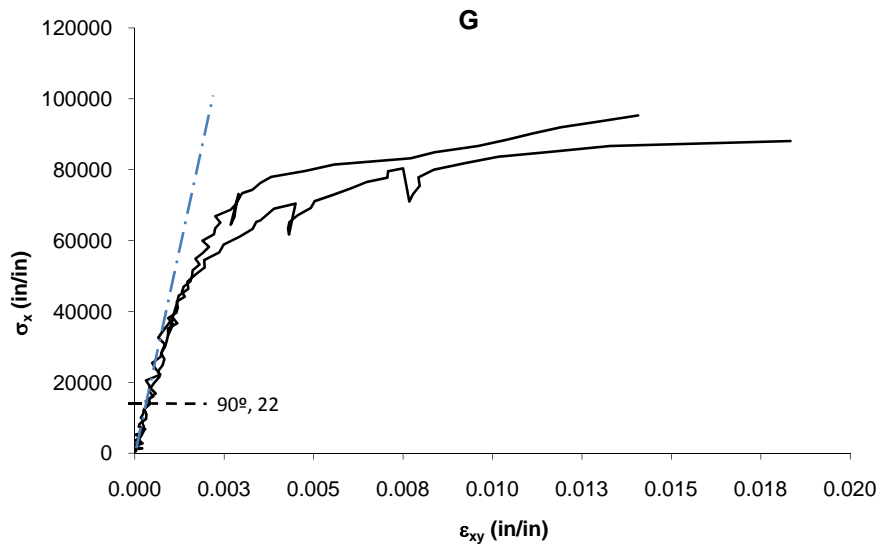
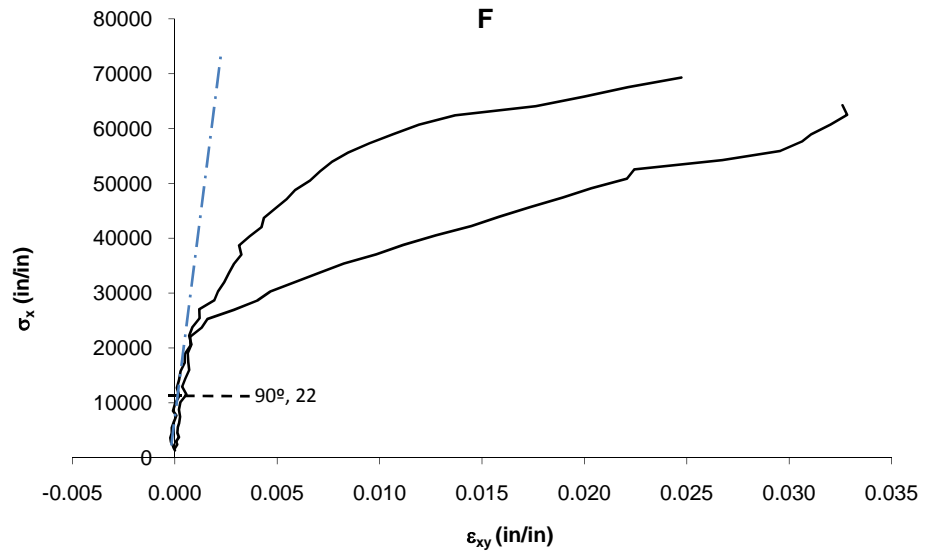


## H.2: Applied Stress vs. Shear Strain





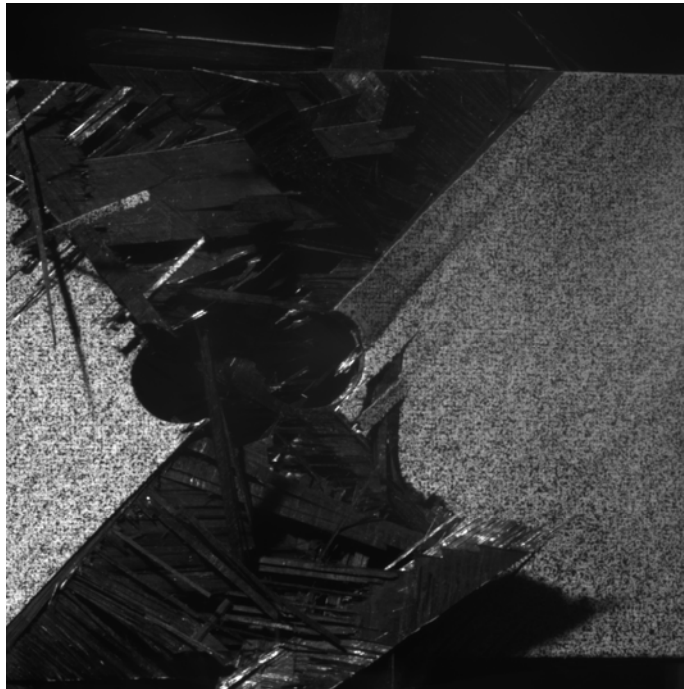




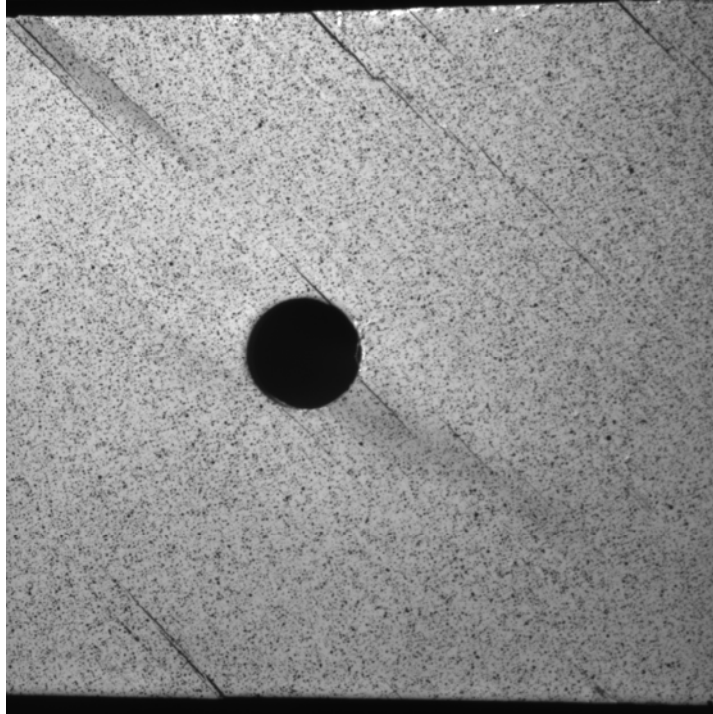
**Appendix I: Selected Laminates at or Near Failure**



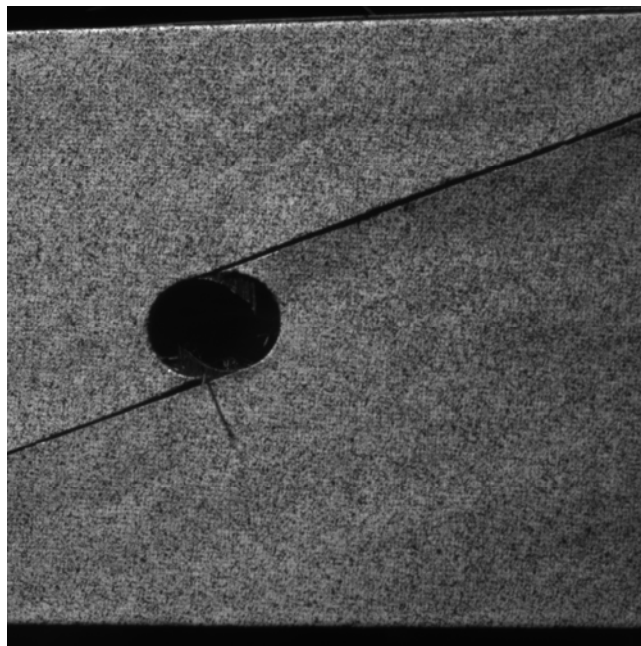
**Figure I.1:** Layup A at Failure



**Figure I.2:** Layup B at Failure



**Figure I.3:** Layup BL at 97% UTS



**Figure I.3:** Layup G at failure